

Ball Spline 示张 General Catalog



Ball Spline 行形 General Catalog

Product Descriptions

Classification of Ball Splines	A 3-4
Point of Selection	A3-6
Flowchart for Selecting a Ball Spline	A 3-6
Steps for Selecting a Ball Spline	A3-6
Selecting a Type	A 3-8
Studying the Spline Shaft Strength	A 3-12
Predicting the Service Life	A 3-21
Selecting a Preload	
Clearance in the Rotation Direction	A 3-30
Preload and Rigidity	A 3-30
Conditions and Guidelines for Selecting of a Preload	
Determining the Accuracy	A 3-34
Accuracy Grades	
Accuracy Standards	

High Torque Caged Ball Spline

Nodels SLS, SLS-L and SLF	A 3-36
Structure and Features	A 3-37
Types and Features	A 3-40
Housing Inner-diameter Tolerance	A3-41

Dimensional Drawing, Dimensional Table

Model SLS	A3-42
Model SLF	A 3-44
Spline Shaft	A3-46
Accessories	A3-48

High Torque Type Ball Spline

Structure and Features	A3-51
Applications	A3-52
Types and Features	A3-53
Housing Inner-diameter Tolerance	A3-55

Dimensional Drawing, Dimensional Table

Model LBS (Medium Load Type)	A3-56
Model LBST (Heavy Load Type)	A3-60
Model LBF (Medium Load Type)	A3-62
Model LBR	A3-64
Model LBH	A3-66
Model LBS with Recommended Shaft End Shape	A3-68
Spline Shaft	A3-69
Accessories	A3-72

Medium Torque Type Ball Spline

A 3-2

Models LT, LF, LT-X and LF-X	3-74
Structure and Features	3-75
Types and Features	3-76

· Housing Inner-diameter To	olerance A3-77
-----------------------------	----------------

Dimensional Drawing, Dimensional Table

Model LT	3-78
Model LF	
Model LT-X	3-82
Model LF-X	3-84
Model LT with Recommended Shaft End Shape 🗖	3-86
Spline Shaft	3-87
Accessories	3-91

Rotary Ball Spline

With Geared Type	Models LBG and	LBGT 🖾 3-92
------------------	----------------	-------------

Dimensional Drawing, Dimensional Table

Model LBG	A 3-96
Model LBGT	A 3-98
Spline Shaft	A3-100

Rotary Ball Spline

With Support Bearing Type Models LTR and LTR-A.. A3-104

Dimensional Drawing, Dimensional Table

Model LTR-A Compact Type	A3-108
Model LTR	
Spline Shaft	A3-112
Permissible Rotational Speed for Rotary Ball Splines	A3-114
Maximum Manufacturing Length by Accuracy	A3-115

Point of Design	A3-117
Checking List for Spline Shaft End Shape	
Housing Inner-diameter Tolerance	A3-118
Positions of the Spline-nut Keyway and Mounting Holes	A3-118

Options Lubrication Material and Surface Treatment	A3-120
Contamination Protection • Specifications of the Bellows	A3-121
Model No. Model Number Coding Precautions on Use	A3-122

B Support Book (Separate)

Features and Types	. 🖪 3-4
Features of the Ball Spline	. <mark>B</mark> 3-4
Structure and Features	. 🖪 3-4
Classification of Ball Splines	. 🖪 3-6
Point of Selection	. 🖪 3-8
Flowchart for Selecting a Ball Spline	. <mark>B</mark> 3-8
Steps for Selecting a Ball Spline	. 🖪 3-8
Selecting a Type	. 🖪 3-10
Studying the Spline Shaft Strength	. 🖪 3-14
Predicting the Service Life	. 🖪 3-19
Example of Calculating the Service Life	. B 3-24
Mounting Procedure and Maintenance.	. 🖪 3-30
Assembling the Ball Spline	. <mark>B</mark> 3-30
Mounting the Spline	. 🖪 3-30
Installing the Spline Nut	. 🖪 3-32
Installation of the Spline Shaft	. 🖪 3-32
Lubrication	. <mark>B</mark> 3-33
Options	. 🖪 3-34
	B3-34
Contamination Protection	. 🖪 3-34
Model No.	. B 3-35
Model Number Coding	
Precautions on Use	B 3-36





Square Nut Type Model LBH

A3-4 冗出比

Features and Types

Classification of Ball Splines



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A 3-6

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Flowchart for Selecting a Ball Spline

Steps for Selecting a Ball Spline

The following is a flowchart to reference when selecting a Ball Spline.



Point of Selection

Flowchart for Selecting a Ball Spline

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Selecting a Type

There are three types of the Ball Spline: high torque type, medium torque type and rotary type. You can choose a type according to the intended use. In addition, wide arrays of spline nut shapes are available for each type, enabling the user to choose a desired shape according to the mounting or service requirements.

	Classification	Туре	Shape	Shaft diameter
ged Ball type		Type SLS Type SLS-L		Nominal shaft diameter 25 to 100mm
High torque Caged Ball type		Type SLF		Nominal shaft diameter 25 to 100mm
		Type LBS Type LBST		Nominal shaft diameter 6 to 150mm
High torque type		Type LBF		Nominal shaft diameter 15 to 100mm
High tor		Type LBR		Nominal shaft diameter 15 to 100mm
		Type LBH		Nominal shaft diameter 15 to 50mm

Specification Table	Structure and features	Major application
⊠3-42	 Redesigning the shape of the conventional high torque type spline shaft to be more circular significantly improves its torsion and flexural rigidity. Models SLS/SLF adopt the caged-ball tech- nology to enable the circulating motion of evenly spaced balls to be maintained and high-speed response to be achieved, the cycle time of the machine can be improved. Models SLS/SLF adopt the caged-ball technology, they eliminate collision and mu 	
四3-44	 tual friction between balls, and realize low noise, pleasant running sound and low particle generation. Models SLS/SLF adopt the caged-ball technology to substantially increase the grease retention, thus achieving long-term maintenance-free operation. Models SLS/SLF adopt the caged-ball technology and a new circulation method, thus achieving stable and smooth motion with small rolling fluctuation. 	 Column and arm of industrial robot Automatic loader Transfer machine Automatic conveyance system Tire molding machine Spindle of spot-welding machine Guide shaft of high-speed automatic coating machine
⊠3-56	 The spline shaft has three crests equidis- tantly formed at angles of 120°. On both 	machine Riveting machine Wire winder Work head of electric discharge machine Spindle drive shaft of grinding machine Speed gears Precision indexing machine
⊠3-62	sides of each crest, two rows (six rows in total) of balls are arranged to hold the crest from both sides. The angular-contact design of the ball contact areas allows an appropri- ate preload to be evenly applied. Since the balls circulate inside the spline nut, the outer dimensions of the spline nut	
图3-64	 are compactly designed. Even under a large preload, smooth straight motion is achieved. Since the contact angle is large (45°) and the displacement is minimal, high rigidity is achieved. 	
⊠3-66	 No angular backlash occurs. Capable of transmitting a large torque. 	

511E

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	Classification	Туре	Shape	Shaft diameter
		Type LT		Nominal shaft diameter 4 to 100mm
Medium torque type		Type LF		Nominal shaft diameter 6 to 50mm
Medium to		Type LT-X		Nominal shaft diameter 4 to 30mm
		Type LF-X		Nominal shaft diameter 4 to 30mm
Rotary type	Rotation	Type LBG Type LBGT		Nominal shaft diameter 20 to 85mm
Rotary	Rotation	Type LTR-A Type LTR		Nominal shaft diameter 8 to 60mm

Selecting a Type

Specification Table	Structure and features	Major application
⊠3-78	 The spline shaft has two to three crests. On both sides of each crest, two rows (four to six rows in total) of balls are arranged to hold the crest from both sides. This design allows 	
⊠3-80	 an appropriate preload to be evenly applied. The contact angle of 20° and an appropriate preload level eliminate angular backlash, providing high-torque moment rigidity. 	 Die-set shaft and similar applications requiring straight motion under a heavy load Column and arm of industrial robot Spot-welding ma- chine Loading system and simi- lar applications requiring Book-binding ma-
⊠3-82	 The length and external diameter of the LT-X ball spline's outer cylinder are the same as those of an LM-series linear bushing, mean- ing the nut can be replaced with a linear bushing. 	rotation to a given angle at a fixed position • Automatic gas-welding machine spindle and simi- lar applications requiring a whirl-stop on one shaft
⊠3-84	• The length and external diameter of the LF-X ball spline's nut are the same as those of the Model LMF linear bushing, meaning the nut can be replaced with a linear bushing.	
⊠3-96	 A unit type that has the same contact structure as model LBS. The flange circumference on the spline nut is machined to have gear teeth, and radial and thrust needle bearings are compactly combined on the circumference of the spline nut. 	Speed gears for high torque transmission
⊠3-108	 A lightweight and compact type based on model LT, but has a spline nut circumference machined to have angular-contact type ball raceways to accommodate support bearings. 	Z axis of scalar robotWire winder

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Studying the Spline Shaft Strength

The spline shaft of the Ball Spline is a compound shaft capable of receiving a radial load and torque. When the load and torque are large, the spline shaft strength must be taken into account.

[Spline Shaft Receiving a Bending Load]

When a bending load is applied to the spline shaft of a Ball Spline, obtain the spline shaft diameter using the equation (1) below.

$$\mathbf{M} = \mathbf{\sigma} \cdot \mathbf{Z}$$
 and $\mathbf{Z} = \frac{\mathbf{M}}{\mathbf{\sigma}}$ (1)

- M : Maximum bending moment acting on the spline shaft (N-mm)
- $\sigma \qquad : \mbox{ Permissible bending stress of the} \\ spline shaft \qquad (98\mbox{N/mm}^2) \label{eq:spline}$

[Reference] Section Modulus (Solid Circle)

$$Z = \frac{\pi \cdot d^{3}}{32}$$

$$Z : Section Modulus \qquad (mm^{3})$$

$$d : Shaft outer diameter \qquad (mm)$$

[Spline Shaft Receiving a Torsion Load]

When a torsion load is applied on the spline shaft of a Ball Spline, obtain the spline shaft diameter using the equation (2) below.

$$\mathbf{T} = \boldsymbol{\tau}_{a} \cdot \mathbf{Z}_{P}$$
 and $\mathbf{Z}_{P} = \frac{\mathbf{T}}{\boldsymbol{\tau}_{a}}$ (2)

- T : Maximum torsion moment (N-mm)
- τ_a : Permissible torsion stress of the
- spline shaft $(49N/mm^2)$ Z_o : Polar modulus of section of the

(mm³)

(see Table3 on A3-17, Table4 on A3-18, Table5 on A3-19 and Table6 on A3-20)

[Reference] Section Modulus (Solid Circle)

$$Z_{P} = \frac{\pi \cdot d^{3}}{16}$$

$$Z_{P} : \text{Section modulus} \qquad (mm^{3})$$

d : Shaft outer diameter (mm)



M: Bending

moment

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spline nut

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Point of Selection

Studying the Spline Shaft Strength

[When the Spline Shaft Simultaneously Receives a Bending Load and a Torsion Load]

When the spline shaft of a Ball Spline receives a bending load and a torsion load simultaneously, calculate two separate spline shaft diameters: one for the equivalent bending moment (M_e) and the other for the equivalent torsion moment (T_e). Then, use the greater value as the spline shaft diameter.

Equivalent bending moment

$$\mathbf{M}_{\circ} = \frac{\mathbf{M} + \sqrt{\mathbf{M}^{2} + \mathbf{T}^{2}}}{2} = \frac{\mathbf{M}}{2} \left\{ \mathbf{1} + \sqrt{\mathbf{1} + \left(\frac{\mathbf{T}}{\mathbf{M}}\right)^{2}} \right\} \dots (3)$$

$$\mathbf{M}_{e} = \sigma \cdot \mathbf{Z}$$

Equivalent torsion moment

 $T_e = \tau_a \cdot Z_p$

[Rigidity of the Spline Shaft]

The rigidity of the spline shaft is expressed as a torsion angle per meter of shaft length. Its value should be limited within $1^{\circ}/4$.

$$\theta = 57.3 \times \frac{\mathbf{T} \cdot \mathbf{L}}{\mathbf{G} \cdot \mathbf{I}_{\mathbf{P}}} \quad \dots \dots \dots \dots \dots (5)$$

Rigidity of the shaft = $\frac{\text{Torsion angle}}{\text{Unit length}} = \frac{\theta \cdot \ell}{L} < \frac{1^{\circ}}{4}$

- θ : Torsion angle (°)
- L : Spline shaft length (mm)
- G : Transverse elastic modulus

- ℓ : Unit length (1000mm)
- I_{P} : Polar moment of inertia (mm⁴)

(see Table3 on A3-17, Table4 on A3-18, Table5 on A3-19 and Table6 on A3-20)



A3-13

511E

[Deflection and Deflection Angle of the Spline Shaft]

The deflection and deflection angle of the Ball Spline shaft need to be calculated using equations that meet the relevant conditions. Table1 and Table2 represent these conditions and the corresponding equations.

Table3 on $\square 3-17$, Table4 on $\square 3-18$, Table5 on $\square 3-19$ and Table6 on $\square 3-20$ show the section modulus of the spline shaft (Z) and the second moment of area (I). Using the Z and I values from the tables, the strength and displacement (deflection) of a typical ball spline within each model type can be obtained.

Support method	Condition	Deflection equation	Deflection angle equation
Both ends free		$\delta_{\max} = \frac{P\ell^3}{48EI}$	$i_1 = 0$ $i_2 = \frac{P\ell^2}{16EI}$
Both ends fas- tened		$\delta_{\max} = \frac{P\ell^3}{192EI}$	$i_1 = 0$ $i_2 = 0$
Both ends free	∑ Uniform load p ℓ ℓ	$\delta_{\max} = \frac{5p\ell^4}{384EI}$	$i_2 = \frac{p\ell^3}{24EI}$
Both ends fas- tened	Uniform load p	$\delta_{\max} = \frac{p\ell^4}{384EI}$	<i>i</i> ₂ = 0

Table1	Deflection	and	Deflection	Anale	Equations
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Ball Spline

Point of Selection

Studying the Spline Shaft Strength



Table2 Deflection and Deflection Angle Equations

 δ_{max} : Maximum deflection (mm)

- Mo: Moment (N-mm)
- ℓ: Span (mm)
- I: Geometrical moment of inertia (mm⁴)
- *i*₁: Deflection angle at loading point

- *i*₂: Deflection angle at supporting point
- P: Concentrated load (N)
- p: Uniform load (N/mm)
- E: Modulus of longitudinal elasticity 2.06×10^{5} (N/mm²)



[Dangerous Speed of the Spline Shaft]

When a Ball Spline shaft is used to transmit power while rotating, as the rotational speed of the shaft increases, the rotation cycle nears the natural frequency of the spline shaft. It may cause resonance and eventually result in inability to move. Therefore, the maximum rotational speed of the shaft must be limited to below the dangerous speed that does not cause resonance.

The dangerous speed of the spline shaft is obtained using the equation (6).

(0.8 is multiplied as a safety factor)

If the shaft's rotation cycle exceeds or nears the resonance point during operation, it is necessary to reconsider the spline shaft diameter.

• Dangerous Speed

$$\mathbf{N}_{e} = \frac{\mathbf{60\lambda}^{2}}{\mathbf{2\pi} \cdot \ell_{b}^{2}} \cdot \sqrt{\frac{\mathbf{E} \times \mathbf{10}^{3} \cdot \mathbf{I}}{\gamma \cdot \mathbf{A}}} \times \mathbf{0.8} \quad \cdots (\mathbf{6})$$

 $\ell_{\text{b}} \hspace{0.5cm} : \text{Distance between two mounting surfaces} \hspace{0.5cm} (\text{mm})$

E : Young's modulus (2.06×10⁵ N/mm²) I : Minimum geometrical moment of

inertia of the shaft (mm⁴)

$$I = \frac{\pi}{64} d^4$$
 d:Minor diameter (mm)

(see Table10, Table11, Table12 and Table13 on 3-24)

 γ : Density (specific gravity) (7.85×10⁻⁶kg/mm³)

$$A = \frac{\pi}{4} d^2 \qquad d: Minor diameter \quad (mm)$$

(see Table10, Table11, Table12 and Table13 on 3-24)

- A : Spline shaft cross-sectional area (mm²)
- λ : Factor according to the mounting method
 - (1) Fixed free $\lambda = 1.875$
 - (2) Supported supported λ =3.142
 - (3) Fixed supported λ =3.927
 - (4) Fixed fixed λ =4.73

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A3-16











Fixed - supported



Fixed - fixed

Ball Spline

Point of Selection

Studying the Spline Shaft Strength

[Cross-sectional Characteristics of the Spline Shaft] Cross-sectional Characteristics of the Spline Shaft for Ball Spline Models SLS, SLS-L and SLF

Nominal shaft diameter		I: Geometrical moment of inertia mm⁴	Z: Modulus section mm ³	l _P : Polar moment of inertia mm⁴	Z _P : Section modulus mm ³
25	Solid shaft	1.61×10⁴	1.29×10 ³	3.22×10⁴	2.57×10^{3}
20	Hollow shaft	1.51×10⁴	1.20×10 ³	3.01×10⁴	2.41×10 ³
30	Solid shaft	3.33×10⁴	2.22×10 ³	6.65×10⁴	4.43×10 ³
50	Hollow shaft	3.00×10⁴	2.00×10 ³	6.01×10⁴	4.00×10 ³
40	Solid shaft	1.09×10⁵	5.47×10 ³	2.19×10⁵	1.09×10⁴
40	Hollow shaft	9.79×10⁴	4.90×10 ³	1.96×10⁵	9.79×10 ³
50	Solid shaft	2.71×10⁵	1.08×10⁴	5.41×10⁵	2.17×10⁴
50	Hollow shaft	2.51×10⁵	1.01×10⁴	5.03×10⁵	2.01×10⁴
60	Solid shaft	5.83×10⁵	1.94×10⁴	1.17×10 ⁶	3.89×104
00	Hollow shaft	5.32×10⁵	1.77×10⁴	1.06×10 ⁶	3.54×10⁴
70	Solid shaft	1.06×10 ⁶	3.02×10⁴	2.11×10 ⁶	6.04×10 ⁴
80	Solid shaft	1.82×10 ⁶	4.55×10⁴	3.64×10 ⁶	9.10×10⁴
00	Hollow shaft	1.45×10 ⁶	3.62×10⁴	2.90×10 ⁶	7.24×10⁴
100	Solid shaft	4.50×10 ⁶	9.00×10 ⁴	9.00×10 ⁶	1.80×10⁵
100	Hollow shaft	3.48×10 ⁶	6.96×10 ⁴	6.96×10 ⁶	1.36×10⁵

Table3 Cross-sectional Characteristics of the Spline Shaft for Models SLS, SLS-L and SLF

Note) For the hole-shape of the hollow spline shaft, see 3-46.



• Cross-sectional Characteristics of the Spline Shaft for Ball Spline Models LBS, LBST, LBF, LBR, LBH, LBG and LBGT

Nominal shaft diameter		I: Geometrical moment of inertia mm⁴	Z: Modulus section mm ³	I _P : Polar momentof inertia mm⁴	Z _P : Section modulus mm ³
6	Solid shaft	50.6	17.8	1.03×10 ²	36.2
8	Solid shaft	1.64×10^{2}	42.9	3.35×10 ²	87.8
10	Solid shaft	3.32×10 ²	73.0	6.80×10 ²	1.50×10 ²
15	Solid shaft	1.27×10 ³	2.00×10 ²	2.55×10 ³	4.03×10 ²
20	Solid shaft	3.82×10 ³	4.58×10 ²	7.72×10 ³	9.26×10 ²
20	Hollow shaft	3.79×10 ³	4.56×10 ²	7.59×10 ³	9.11×10 ²
05	Solid shaft	9.62×10 ³	9.14×10 ²	1.94×10 ⁴	1.85×10 ³
25	Hollow shaft	9.50×10 ³	9.05×10 ²	1.90×10⁴	1.81×10 ³
20	Solid shaft	1.87×10⁴	1.50×10 ³	3.77×10⁴	3.04×10 ³
30	Hollow shaft	1.78×10⁴	1.44×10 ³	3.57×10⁴	2.88×10 ³
40	Solid shaft	6.17×10⁴	3.69×10 ³	1.25×10⁵	7.46×10 ³
40	Hollow shaft	5.71×10⁴	3.42×10 ³	1.14×10⁵	6.84×10 ³
50	Solid shaft	1.49×10⁵	7.15×10 ³	3.01×10⁵	1.45×104
50	Hollow shaft	1.34 × 10⁵	6.46×10 ³	2.69×10⁵	1.29×104
60	Solid shaft	3.17×10⁵	1.26×104	6.33×10⁵	2.53×104
00	Hollow shaft	2.77×10⁵	1.11×10 ⁴	5.54×10⁵	2.21×10⁴
70	Solid shaft	5.77×10⁵	1.97×104	1.16×10 ⁶	3.99×104
70	Hollow shaft	5.07×10⁵	1.74×10 ⁴	1.01×10 ⁶	3.49×10⁴
85	Solid shaft	1.33×10 ⁶	3.69×104	2.62×10 ⁶	7.32×104
co	Hollow shaft	1.11×10 ⁶	3.10×10⁴	2.22×10 ⁶	6.20×104
100	Solid shaft	2.69×10 ⁶	6.25×104	5.33×10 ⁶	1.25×10⁵
100	Hollow shaft	2.18×10 ⁶	5.10×10 ⁴	4.37×10 ⁶	1.02×10 ⁵
120	Solid shaft	5.95×10°	1.13×10⁵	1.18×10 ⁷	2.26×10⁵
120	Hollow shaft	5.28×10 ⁶	1.01×10⁵	1.06×10 ⁷	2.02×10 ⁵
150	Solid shaft	1.61×10 ⁷	2.40×10⁵	3.20×10 ⁷	4.76×10⁵
150	Hollow shaft	1.40×107	2.08×10⁵	2.79×10 ⁷	4.16×10⁵

Table4 Cross-sectional Characteristics of the Spline Shaft for Models LBS, LBST, LBF, LBR, LBH, LBG and LBGT

Note) For the hole-shape of the hollow spline shaft, see **3-69** and **3-100**.

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Studying the Spline Shaft Strength

• Cross-sectional Characteristics of the Spline Shaft for Ball Spline Models LT, LF, LTR and LTR-A

Table5 Cross-sectional Characteristics of the Spline Shaft for Models LT, LF, LTR and LTR-A

Nominal shaft diameter		I: Geometrical moment of inertia mm⁴	Z: Modulus section mm ³	l⊧: Polar moment of inertia mm⁴	Z _P : Section modulus mm ³	
4	Solid shaft		11.39	5.84	22.78	11.68
5	Solid shaft			11.43	55.76	22.85
-	Solid shaft		57.80	19.7	1.19×10 ²	40.50
6	6 Hollow shaft Type K		55.87	18.9	1.16×10 ²	39.20
	Solid shaft		1.86×10 ²	47.4	3.81×10 ²	96.60
8	8 Hollow shaft		1.81×10 ²	46.0	3.74×10 ²	94.60
	Solid shaft		4.54×10 ²	92.6	9.32×10 ²	1.89×10 ²
10	Hollow shaft	Туре К	4.41×10 ²	89.5	9.09×10 ²	1.84×10 ²
	Solid shaft		1.32×10 ³	2.09×10 ²	2.70×10 ³	4.19×10 ²
13	Hollow shaft	Туре К	1.29×10 ³	2.00×10 ²	2.63×10 ³	4.09×10 ²
	Solid shaft		3.09×103	3.90×10 ²	6.18×10 ³	7.80×10 ²
16	Lallau at - 0	Туре К	2.97×103	3.75×10 ²	5.95×10 ³	7.51×10 ²
	Hollow shaft	Type N	2.37×10 ³	2.99×10 ²	4.74×10 ³	5.99×10 ²
	Solid shaft		7.61×10 ³	7.67×10 ²	1.52×104	1.53×10 ³
20	1.1.1.1	Туре К	7.12×10 ³	7.18×10 ²	1.42×104	1.43×10 ³
	Hollow shaft	Type N	5.72×10 ³	5.77×10 ²	1.14×104	1.15×10 ³
	Solid shaft			1.50×10 ³	3.71×104	2.99×10 ³
25	Ty	Туре К	1.75×10⁴	1.41×10 ³	3.51×10⁴	2.83×10 ³
	Hollow shaft	Туре N	1.34×10⁴	1.08×10 ³	2.68×104	2.16×10 ³
	Solid shaft	Solid shaft		2.59×10 ³	7.71×104	5.18×10 ³
30	Hollow shaft	Туре К	3.53×10⁴	2.37×10 ³	7.07×104	4.74×10 ³
		Туре N	2.90×10⁴	1.95×10 ³	5.80×104	3.89×10 ³
Solid shaft			5.01×10⁴	3.15×10 ³	9.90×104	6.27×10 ³
32	Hollow shaft	Туре К	4.50×10⁴	2.83×10 ³	8.87×104	5.61×10^{3}
	HONOW SHAIL	Type N	3.64×10⁴	2.29×10 ³	7.15×10⁴	4.53×10 ³
	Solid shaft		1.22×10⁵	6.14×10 ³	2.40×10 ⁵	1.21×10⁴
40	Hollow shaft	Туре К	1.10×10⁵	5.55×10 ³	2.17×10 ⁵	1.10×10⁴
	HONOW SHAIL	Type N	8.70×10⁴	4.39×10 ³	1.71×10⁵	8.64×10 ³
	Solid shaft		2.97×10 ⁵	1.20×104	5.94×10⁵	2.40×10 ⁴
50	Hollow shaft	Туре К	2.78×10⁵	1.12×10 ⁴	5.56×10⁵	2.24×10⁴
		Type N	2.14×10 ⁵	8.63×10 ³	4.29×10 ⁵	1.73×10⁴
60	Solid shaft		6.16×10⁵	2.07×104	1.23×10 ⁶	4.14×10 ⁴
00	Hollow shaft	Туре К	5.56×10⁵	1.90×10⁴	1.13×10 ⁶	3.79×10⁴
80	Solid shaft		1.95×10 ⁶	4.91×10⁴	3.90×10 ⁶	9.82×10⁴
50	Hollow shaft	Туре К	1.58×10 ⁶	3.97×10⁴	3.15×10 ⁶	7.95×10⁴
100	Solid shaft		4.78×10 ⁶	9.62×10⁴	9.56×10 ⁶	1.92×10⁵
100	Hollow shaft	Туре К	3.76×10 ⁶	7.57×10⁴	7.52×10 ⁶	1.51×10⁵

Note) For the hole-shape of the hollow spline shaft. For type K: see ▲3-88 and ▲3-112. For type N: see ▲3-88 and ▲3-112.

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• Cross-sectional Characteristics of the Spline Shaft for Ball Spline Models LT-X and LF-X

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Nominal shaft diameter		I:Geometrical moment of inertia mm⁴	Z: Modulus section mm ³	I _P :Polar moment of inertia mm⁴	Z _P :Section modulus mm ³			
4	Solid shaft	11.2	5.7	23.2	11.8			
5	Solid shaft	27.7	11.3	57.2	23.3			
6	Solid shaft	57.7	19.6	119.1	40.4			
8	Solid shaft	175.6	45	366.2	93.9			
10	Solid shaft	422.3	86.5	896.9	183.8			
10	Туре К	409.7	84	871.7	178.6			
40	Solid shaft	1215.3	191.3	2574.6	405.3			
13	Туре К	1184.6	186.5	2513.2	395.6			
	Solid shaft	2734.3	350.8	5844.5	749.7			
16	Туре К	2616.4	335.6	5608.8	719.5			
	Type N	2015.6	258.6	4407.2	565.4			
	Solid shaft	7043.9	716.5	14731.7	1498.5			
20	Туре К	6553	666.6	13749.9	1398.7			
	Type N	5158.1	524.7	10960.2	1114.9			
	Solid shaft	17268.2	1404.2	36067.4	2932.9			
25	Туре К	16250.3	1321.4	34031.6	2767.4			
	Type N	12115.2	985.2	25761.4	2094.8			
	Solid shaft	36115.8	2444.1	75160	5086.3			
30	Туре К	32898.8	2226.4	68726.1	4650.9			
	Type N	26569.7	1798	56067.4	3794.2			

Table6 Cross-sectional Characteristics of the Spline Shaft for Models LT-X and LF-X

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Predicting the Service Life

Predicting the Service Life

[Static Safety Factor]

To calculate a load applied to the ball spline, you must first know the average load used to calculate the service life and the maximum load used to calculate the static safety factor.

In particular, if the system starts and stops frequently, or if impact loads are applied, a large moment load or torque caused by overhung loads may be applied to the system. When selecting a model number, make sure that the desired model is capable of handling the required maximum load (whether stationary or in motion). The reference values for the static safety factor are shown in the table below.

e	f _T · f _c · C _o
IS -	Pmax

s	: Statio	c safety	factor
---	----------	----------	--------

C_{0}	: Basic static load rating*	(N)
Pma	: Maximum applied load	(N)

- f⊤ : Temperature factor (see Fig.1 on A3-23)
- fc : Contact factor (see Table8 on A3-23)

*The basic static load rating is a static load of a defined direction and size where the sum of the permanent deformation of the ball and that of the rolling groove at the contact area under maximum stress is 0.0001 times the ball diameter.

Table7 Reference Values of Static Safety Factor (fs)

Machine using the Ball Spline	Load conditions	Minimum reference values
	Without vibration or impact	3.0 to 6.0
General industrial machinery	Without vibration or impact	4.0 to 7.0
	With vibration or impact under combined loads	5.0 to 8.0

*The reference values for the static safety factor may vary depending on the load conditions as well as the environment. lubrication status, precision of the mounted surface, and/or rigidity.

[Nominal Life]

The service life of a Ball Spline varies from unit to unit even if they are manufactured through the same process and used in the same operating conditions. Therefore, the nominal life defined below is normally used as a guidepost for obtaining the service life of a Ball Spline.

Nominal life is the total travel distance that 90% of a group of identical ball splines independently operating under the same conditions can achieve without showing flaking (scale-like pieces on a metal surface).



[Calculating the Nominal Life]

The nominal life of a Ball Spline varies with types of loads applied during operation: torque load, radial load and moment load. The corresponding nominal life values are obtained using the equations (7) to (10) below. (The basic load ratings in these loading directions are indicated in the specification table for the corresponding model number.)

When a Torque Load is Applied

$$\mathbf{L} = \left(\frac{\mathbf{f}_{\tau} \cdot \mathbf{f}_{c}}{\mathbf{f}_{w}} \cdot \frac{\mathbf{C}_{\tau}}{\mathbf{T}_{c}}\right)^{3} \times 50 \quad \dots \dots (7)$$

When a Radial Load is Applied

$$\mathbf{L} = \left(\frac{\mathbf{f}_{\tau} \cdot \mathbf{f}_{c}}{\mathbf{f}_{w}} \cdot \frac{\mathbf{C}}{\mathbf{P}_{c}}\right)^{3} \times 50 \quad \dots \dots (8)$$

L	: Nominal life	(km)
C_{T}	: Basic dynamic torque rating	(N-m)
T_{C}	: Calculated torque applied	(N-m)
С	: Basic dynamic load rating	(N)
P_{C}	: Calculated radial load	(N)
f⊤	: Temperature factor	
	(see Fig.1 d	on A3-23)
fc	: Contact factor	
	(see Table8 o	on A3-23)

(see Table9 on A3-23) fw : Load factor



• When a Torque Load and a Radial Load are Simultaneously Applied

When a torque load and a radial load are simultaneously applied, calculate the nominal life by obtaining the equivalent radial load using the equation (9) below.

$\mathbf{P}_{\mathbf{E}} = \mathbf{P}_{\mathbf{c}} + \frac{4 \cdot \mathbf{T}_{\mathbf{c}}}{i \cdot \mathbf{d} \mathbf{p}} \cdot$	×10 ³ (9)		
P _E : Equivalent r	adial load	(N)	
$\cos \alpha$: Contact ang	le <i>i</i> =Number of rows of	balls under a load	
/ Type LBS α =45°	<i>i</i> =2 (LBS10 or smaller)	Type SLS α =40°	<i>i</i> =3 \
	<i>i</i> =3 (LBS15 or greater)		
Type LTα=70°	<i>i</i> =2 (LT13 or smaller)	Type LT-X α =65°	<i>i</i> =2
(<i>i</i> =3 (LT16 or greater)		/
dn · Ball center-to	-center diameter (mm)	

- dp : Ball center-to-center diameter (mm)(see Table10, Table11, Table12 and Table13 on 3-24)
- When a Moment Load is Applied to a Single Nut or Two Nuts in Close Contact with Each Other

Obtain the equivalent radial load using the equation (10) below.

$P_u = K \cdot M$(10)

- Pu : Equivalent radial load (with a moment applied)
- Κ : Equivalent Factors

Μ

(see Table14 on **3-27**, Table15 on **3-28**, Table16 and Table17 on **3-29**)

(N)

(N-mm) : Applied moment

However, M should be within the range of the static permissible moment.

When a Moment Load and a Radial Load are Simultaneously Applied

Calculated the nominal life from the sum of the radial load and the equivalent radial load.

Calculating the Service Life Time

When the nominal life (L) has been obtained in the equation above, if the stroke length and the number of reciprocations per minute are constant, the service life time is obtained using the equation (11) below.

$$\mathbf{L}_{h} = \frac{\mathbf{L} \times \mathbf{10}^{3}}{\mathbf{2} \times \boldsymbol{\ell}_{s} \times \mathbf{n}_{1} \times \mathbf{60}} \qquad \cdots \cdots$$

·(11)

- L_b : Service life time (h) (m)
- ls : Stroke length
- : Number of reciprocations per minute n₁

(min⁻¹)

Ball Spline

Point of Selection

Predicting the Service Life

■f_T: Temperature Factor

If the temperature of the environment surrounding the operating Ball Spline exceeds 100°C, take into account the adverse effect of the high temperature and multiply the basic load ratings by the temperature factor indicated in Fig.1.

In addition, the Ball Spline must be of a high temperature type.

Note) If the environment temperature exceeds 80°C, hightem-perature types of seal and retainer are required. ContactTHK for details.

■fc: Contact Factor

When multiple spline nuts are used in close contact with each other, their linear motion is affected by moments and mounting accuracy, making it difficult to achieve uniform load distribution. In such applications, multiply the basic load rating (C) and (C₀) by the corresponding contact factor in Table8. Note) if uneven load distribution is expected in a large ma-

chine, take into account the respective contact factor indicated in Table8.

■f_w: Load Factor

In general, reciprocating machines tend to involve vibrations or impact during operation. It is extremely difficult to accurately determine vibrations generated during high-speed operation and impact during frequent start and stop. When loads applied on a Ball Spline cannot be measured, or when speed and impact have a significant influence, divide the basic load rating (C or C_0), by the corresponding load factor in the table of empirically obtained data on Table9.

Table8 Contact Factor (f_c)

Number of spline nuts in close contact with each other	Contact factor f.
2	0.81
3	0.72
4	0.66
5	0.61
Normal use	1

Table9 Load Factor (f_w)

Vibrations/ impact	Speed (V)	f _w
Faint	Very low V≦0.25m/s	1 to 1.2
Weak	Slow 0.25 <v≦1m s<="" td=""><td>1.2 to 1.5</td></v≦1m>	1.2 to 1.5
Medium	Medium 1 <v≦2m s<="" td=""><td>1.5 to 2</td></v≦2m>	1.5 to 2
Strong	High V>2m/s	2 to 3.5

[Sectional Shape of the Spline Shaft]

Spline Shaft for Models SLS, SLS-L and SLF

Table10 Sectional Shape										
ødp ød ød	Nominal shaft diameter	25	30	40	50	60	70	80	100	
	Minor diameter ϕ d	21.6	25.8	35.2	44.4	54.0	62.8	71.3	90.0	
	Major diameter $\phi D_0 h7$	25	30	40	50	60	70	80	100	
	Ball center-to-center diameter ϕ dp	25.2	30.2	40.6	50.6	61.0	71.0	80.8	101.2	
4										

*The minor diameter ϕ d must be a value at which no groove is left after machining.

• Spline Shaft for Models LBS, LBST, LBF, LBR, LBH, LBG and LBGT



Table11 Sectional Shape											Un	it: mm
Nominal shaft diameter	15	20	25	30	40	50	60	70	85	100	120	150
Minor diameter ϕ d	11.7	15.3	19.5	22.5	31	39	46.5	54.5	67	81	101	130
Outer diameter ϕ D ₀	14.5	19.7	24.5	29.6	39.8	49.5	60	70	84	99	117	147
Ball center-to-center diameter \u00f6 dp	15	20	25	30	40	50	60	70	85	100	120	150

*The minor diameter d must be a value at which no groove is left after machining.

Spline Shaft for Models LT, LF, LTR and LTR-A



Nominal shaft diameter: 13 mm or less

Nominal shaft diameter: 16 mm or more

Table12 Sectional Shape Unit: mm Nominal shaft 4 5 6 8 10 13 16 20 25 30 32 40 50 60 80 100 diameter Minor diameter 3.5 4.5 5 7 8.5 11.5 14.5 18.5 23 28 30 37.5 46.5 56.5 75.5 95 φ d Outer diameter 5 100 4 6 8 10 13 16 20 25 30 32 40 50 60 80 *φ* D₀ h7 Outer diameter 0 0 0 0 0 0 0 -0.012 -0.015 -0.018 -0.021 -0.025 -0.03 -0.035 tolerance Ball center-to-center 4.6 5.7 7 9.3 11.5 14.8 17.8 22.1 27.6 33.2 35.2 44.2 55.2 66.3 87.9 109.5 diameter ø dp

*The minor diameter d must be a value at which no groove is left after machining.

• Spline Shaft for Models LT-X and LF-X

Table13 Sectional Shape										nit: mm
Nominal shaft diameter	4X	5X	6X	8X	10X	13X	16X	20X	25X	30X
Minor diameter ϕ d	3.6	4.5	5.4	7	8.6	11.3	13.9	17.9	22.4	27
Major diameter ϕD_0	4	5	6	8	10	13	16	20	25	30
Ball center-to-center diameter ϕ dp	4.4	5.5	6.6	8.6	10.7	13.8	17.1	21.1	26.4	31.6



A3-24



Predicting the Service Life

[Calculating the Average Load]

When the load applied on the spline shaft fluctuates according to varying conditions, such as an industrial robot arm traveling forward while holding a workpiece and traveling backward with empty weight, and a machine tool handling various workpieces, this varying load condition must be taken into account in service life calculation.

The average load (P_m) is a constant load under which the service life of an operating Ball Spline with its spline nut receiving a fluctuation load in varying conditions is equivalent to the service life under this varying load condition.

The following is the basic equation.

$$\mathbf{P}_{m} = \sqrt[3]{\frac{1}{L} \cdot \sum_{n=1}^{n} (\mathbf{P}_{n}^{3} \cdot \mathbf{L}_{n})}$$

Pm	: Average Load	(N)
Pn	: Varying load	(N)
L	: Total travel distance	(mm)
Ln	: Distance traveled under Pn	(mm)

• When the Load Fluctuates Stepwise

- P_m : Average Load
- Pn : Varying load
- L : Total travel distance (m)
- L_n : Distance traveled under load P_n (m)



(N)

(N)

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• When the Load Fluctuates Monotonically







(a) $P_m \doteq 0.65P_{max} \cdots (14)$ (b) $P_m \doteq 0.75P_{max} \cdots (15)$









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Predicting the Service Life

[Equivalent Factor] Table14 on **A3-27**, Table15 on **A3-28**, Table16 and Table17 on **A3-29** show equivalent radial load factors calculated under a moment load.

• Table of Equivalent Factors for Ball Spline Models SLS/SLF





Table14			
	Equivalent factor: K		
Model No.	Single spline nut	Two spline nuts in close contact with each other	
SLS/SLF 25	0.187	0.030	
SLS 25L	0.148	0.027	
SLS/SLF 30	0.153	0.027	
SLS 30L	0.129	0.024	
SLS/SLF 40	0.114	0.021	
SLS 40L	0.102	0.019	
SLS/SLF 50	0.109	0.018	
SLS 50L	0.091	0.017	
SLS/SLF 60	0.080	0.015	
SLS 60L	0.072	0.014	
SLS/SLF 70	0.101	0.016	
SLS 70L	0.076	0.014	
SLS/SLF 80	0.083	0.013	
SLS 80L	0.072	0.012	
SLS/SLF 100	0.068	0.011	
SLS 100L	0.056	0.010	

Ball Spline

• Table of Equivalent Factors for Ball Spline Model LBS





Model LBS	Table15			
	Equivalent factor: K			
Model No.	Single spline nut	Two spline nuts in close contact with each other		
LBS 6	0.61	0.074		
LBS 8	0.46	0.060		
LBS 10	0.54	0.049		
LBS 15	0.22	0.039		
LBS 20	0.24	0.03		
LBST 20	0.17	0.027		
LBS 25	0.19	0.026		
LBST 25	0.14	0.023		
LBS 30	0.16	0.022		
LBST 30	0.12	0.02		
LBS 40	0.12	0.017		
LBST 40	0.1	0.016		
LBS 50	0.11	0.015		
LBST 50	0.09	0.014		
LBST 60	0.08	0.013		
LBS 70	0.1	0.013		
LBST 70	0.08	0.012		
LBS 85	0.08	0.011		
LBST 85	0.07	0.01		
LBS 100	0.08	0.009		
LBST 100	0.06	0.009		
LBST 120	0.05	0.008		
LBST 150	0.045	0.006		

Note1) Values of equivalent factor K for model LBF are the same as that for model LBS.

Note2) Values of equivalent factor K for models LBR, LBG, LBGT and LBH are the same as that for model LBST.

However the values of model LBF60 are the same

as that for model LBST60. The values of model LBH15 are the same as that for model LBS15.

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Point of Selection

Predicting the Service Life

• Table of Equivalent Factors for Ball Spline Model LT





Table16				
	Equivalent factor: K			
Model No.	Single spline nut	Two spline nuts in close contact with each other		
LT 4	0.65	0.096		
LT 5	0.55	0.076		
LT 6	0.47	0.06		
LT 8	0.47	0.058		
LT 10	0.31	0.045		
LT 13	0.3	0.042		
LT 16	0.19	0.032		
LT 20	0.16	0.026		
LT 25	0.13	0.023		
LT 30	0.12	0.02		
LT 40	0.088	0.016		
LT 50	0.071	0.013		
LT 60	0.07	0.011		
LT 80	0.062	0.009		
LT100	0.057	0.008		

Note) Values of equivalent factor K for models LF, LTR and LTR-A are the same as that for model LT. However, the equivalent factor for model LTR32 is the same as that for model LT30.





Table17			
	Equivalent factor: K		
Model No.	Single spline nut	Two spline nuts in close contact with each other	
LT 4X	0.995	0.135	
LT 5X	0.980	0.125	
LT 5XL	0.430	0.0740	
LT 6X	0.660	0.0993	
LT 6XL	0.360	0.0633	
LT 8X	0.420	0.0626	
LT 8XL	0.210	0.0409	
LT 10X	0.251	0.0470	
LT 13X	0.241	0.0420	
LT 16X	0.173	0.0320	
LT 20X	0.129	0.0250	
LT 25X	0.114	0.0220	
LT 30X	0.101	0.0200	

Note) The values shown are those for models equipped with

Values of equivalent factor K for model LF-X are the same as that for model LT-X.





Selecting a Preload

A preload on the Ball Spline significantly affects its accuracy, load resistance and rigidity. Therefore, it is necessary to select the most appropriate clearance according to the intended use.

Specific clearance values are standardized for each model, allowing you to select a clearance that meets the conditions.

Clearance in the Rotation Direction

With the Ball Spline, the sum of clearances in the circumferential direction is standardized as the clearance in the rotational direction. For models LBS and LT, which are especially suitable for transmission of rotational torque, clearances in the rotational directions are defined.

Clearance in the rotational direction (BCD)



Fig.5 Measurement of Clearance in the Rotational Direction

Preload and Rigidity

A3-30

Preload is defined as the load preliminarily applied to the ball in order to eliminate angular backlash (clearance in the rotational direction) and increase rigidity. When given a preload, the Ball Spline is capable of increasing its rigidity by eliminating the angular backlash according to the magnitude of the preload. Fig.6 shows the displacement in the rotational direction when a rotational torque is applied.

Thus, the effect of a preload can be obtained up to 2.8 times that of the applied preload. When given the same rotational torque, the displacement when a preload is applied is 0.5 or less of that without a preload. The rigidity with a preload is at least twice greater than that without a preload.

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Selecting a Preload

Conditions and Guidelines for Selecting of a Preload

Table18 provides guidelines for selecting a clearance in the rotational direction with given conditions of the Ball Spline.

The rotational clearance of the Ball Spline significantly affects the accuracy and rigidity of the spline nut. Therefore, it is essential to select a correct clearance according to the intended use. Generally, the Ball Spline is provided with a preload. When it is used in repeated circular motion or reciprocating straight motion, the Ball Spline is subject to a large vibration impact, and therefore, its service life and accuracy are significantly increased with a preload.

Clearance in the rotation direction	Condition	Examples of applications
Normal grade (No symbol)	 Smooth motion with a small force is desired. A torque is always applied in the same direction. 	 Measuring instruments Automatic drafting machine Geometrical measuring equipment Dynamometer Wire winder Automatic welding machine Main shaft of horning machine Automatic packing machine
Light preload (CL)	 An overhang load or moment load is present. High positioning repeatability is required. Alternating load is applied. 	 Industrial robot arm Automatic loaders Guide shaft of automatic coating machine Main shaft of electric discharge machine Guide shaft for press die setting Main shaft of drilling machine
Medium preload (CM)	 High rigidity is required and vibrations and impact are applied. Receives a moment load with a single spline nut. 	 Steering shaft of construction vehicle Shaft of spot-welding machine Indexing shaft of automatic lathe tool rest

Table18 Guidelines for Selecting a Clearance in the Rotational Direction for the Ball Spline



Fig.7 Comparison between LBS and LT for Zero Clearance



Fig.8 Comparison between LBS and LT for Clearance CL

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Unit:µm

			1
Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	СМ
25 30 40	+1 to -2	-2 to -6	-6 to -10
50 60	+2 to -4	-4 to -8	-8 to -12
70 80 100	+4 to -8	–8 to –12	–12 to –20

Table20 Clearance in the Rotational Direction for Models LBS, LBF, LBST, LBR and LBH $$\mbox{Unit: }\mu\mbox{m}$$

Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	СМ
6 8	-2 to +1	−6 to −2	—
10 15	–3 to +2	–9 to –3	–15 to –9
20 25 30	-4 to +2	-12 to -4	-20 to -12
40 50 60	–6 to +3	–18 to –6	-30 to -18
70 85	8 to +4	-24 to -8	-40 to -24
100 120	-10 to +5	-30 to -10	–50 to –30
150	-15 to +7	-40 to -15	-70 to -40

Table21 Clearance in the Rotational Direction for Models LT and LF

Unit: µm

Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	СМ
4 5 6 8 10 13	–2 to +1	-6 to -2	—
16 20	–2 to +1	-6 to -2	–9 to –5
25 30	-3 to +2	-10 to -4	-14 to -8
40 50	-4 to +2	–16 to –8	-22 to -14
60 80	–5 to +2	-22 to -12	-30 to -20
100	6 to +3	-26 to -14	–36 to –24

Table22 Clearance in the Rotational Direction for Models LT-X and LF-X

Unit: µm

Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	СМ
4568	–2 to +1	-6 to -2	—
10 13	–2 to +1	_4 to −2	_
16 20	–2 to +1	−5 to −2	–8 to –5
25 35	-3 to +1	−7 to −3	–11 to –7

Point of Selection

Selecting a Preload

Unit: µm

Table23 Clearance in the Rotational Direction for Models LBG and LBGT

Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	СМ
20 25 30	-4 to +2	-12 to -4	-20 to -12
40 50 60	-6 to +3	–18 to –6	-30 to -18
70 85	8 to +4	24 to8	-40 to -24

Unit: µm

Ball Spline

Table24 Clearance in the Rotational Direction for Model LTR

Symbol	Normal	Light preload	Medium preload
Nominal shaft diameter	No Symbol	CL	СМ
8 10	-2 to +1	6 to2	—
16 20	-2 to +1	6 to2	–9 to –5
25 32	-3 to +2	-10 to -4	-14 to -8
40 50	-4 to +2	-16 to -8	-22 to -14
60	–5 to +2	-22 to -12	–30 to –20

Determining the Accuracy

Accuracy Grades

The accuracy of the Ball Spline is classified into three grades: normal grade (no symbol), high accuracy grade (H) and precision grade (P), according to the runout of spline nut circumference in relation to the support of the spline shaft. Fig.9 shows measurement items.



Fig.9 Accuracy Measurement Items of the Ball Spline

Accuracy Standards

Table25 to Table29 show measurement items of the Ball Spline.

																								-	
Accu	iracy		Runout(max)																						
	al shaft neter	41	to 8	Note		10		13	3 to 2	20	25	i to 3	32	4	0, 5	0	60) to 8	30	85	to 1	20		150	
Overal shaft len		nal	gh	sion	nal	gh	sion	nal	h	sion	nal	gh	sion	nal	gh	sion	nal	gh	sion	nal	h	sion	nal	gh	sion
Above	Or less	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision
—	200	72	46	26	59	36	20	56	34	18	53	32	18	53	32	16	51	30	16	51	30	16	_	-	_
200	315	133	(89)	_	83	54	32	71	45	25	58	39	21	58	36	19	55	34	17	53	32	17	_	-	—
315	400	_	_	_	103	68	_	83	53	31	70	44	25	63	39	21	58	36	19	55	34	17	_	_	—
400	500	_	_	_	123	_	_	95	62	38	78	50	29	68	43	24	61	38	21	57	35	19	46	36	19
500	630	-	_	_	_	—	—	112	-	-	88	57	34	74	47	27	65	41	23	60	37	20	49	39	21
630	800	—	_	—	_	-	—	—	—	-	103	68	42	84	54	32	71	45	26	64	40	22	53	43	24
800	1000	-	_	_	_	—	—	-	—	-	124	83	—	97	63	38	79	51	30	69	43	24	58	48	27
1000	1250	—	_	—	_	_	—	—	—	—	—	—	-	114	76	47	90	59	35	76	48	28	63	55	32
1250	1600	—	_	_	—	—	—	-	—	-	—	—	—	139	93	—	106	70	43	86	55	33	80	65	40
1600	2000	_	_	_	_	_	_	_	—	_	_	_	_	_	_	—	128	86	54	99	65	40	100	80	50
2000	2500	_	—	—	-	_	—	—	_	_	_	—	—	—	—	—	156	—	—	117	78	49	125	100	68
2500	3000	-	_	—	-	_	_	_	-	-	-	_	—	—	—	—	_	—	_	143	96	61	150	129	84

Table25 Runout of the Spline Nut Circumference in Relation to the Support of the Spline Shaft

Unit: µm

Note) Dimensions in parentheses do not apply to nominal shaft diameter of 4. Note) Applicable to models SLS, SLF, LBS, LBST, LBF, LBR, LT and LF.

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Determining the Accuracy

Table26 Runout of the Spline Nut Outer Diameter in Relation to the Support Sections for Model LT-X and LF-X Spline Shafts Unit:µm

																•
Accu	iracy	Runout(max)														
Nominal shaft diameter		4, 5			6, 8			10			13, 16, 20			25, 30		
	l spline gth (mm)	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision	Normal	High	Precision
Above	Or less	No	I	Prec	°N N	エ	Prec	°2	T	Prec	٩	I	Prec	°2		Prec
—	200	72	46	26	72	46	26	59	36	20	56	34	18	53	32	18
200	315	133*1	—	-	133	89*2	57*3	83	54	32	71	45	25	58	39	21
315	400	—	—	—	171	114	—	103	68	41	83	53	31	70	44	25
400	500	—	—	—	214	—	—	123	82	51	95	62	38	78	50	29
500	630	—	—	—	—	—	—	151	102	—	112	75	46	88	57	34
630	800	—	—	-	—	—	-	190	—	—	137	92	58	103	68	42
800	1000	_	_	_	_	_	_	_	_		170	115	75	124	83	52
1000	1250	—	—	—	—	—	—	—	—	—	—	—	—	151	102	65
1250	1600	_	_	_	_	_	_	_	_	_	_	_	_	190	130	85

*1 Except #4. Value is applicable to #5 up to 250 mm. *2 Value is applicable to #6 up to 250 mm. *3 Except #6.

Table27 Perpendicularity of the Spline Shaft End Face in Relation to the Support of the Spline Shaft Unit: µm

Accuracy	Perpendicularity (max)						
Nominal shaft diameter	Normal grade (No symbol)	High accuracy grade (H)	Precision Grade (P)				
4 5 6 8 10	22	9	6				
13 15 16 20	27	11	8				
25 30 32	33	13	9				
40 50	39	16	11				
60 70 80	46	19	13				
85 100 120	54	22	15				
150	63	25	18				

Table28 Concentricity of the Part-mounting in Relation to the Support of the Spline Shaft $Unit: \mu m$

Accuracy	Concentricity (max)						
Nominal shaft diameter	Normal grade (No symbol)	High accuracy grade (H)	Precision Grade (P)				
4 5 6 8	33	14	8				
10	41	17	10				
13 15 16 20	46	19	12				
25 30 32	53	22	13				
40 50	62	25	15				
60 70 80	73	29	17				
85 100 120	86	34	20				
150	100	40	23				

Table29 Straightness of the Flange-mounting Surface of the Spline Nut in Relation to the Support of the Spline Shaft Unit: μm

Accuracy	Perpendicularity (max)							
Nominal shaft diameter	Normal grade (No symbol)	High accuracy grade (H)	Precision Grade (P)					
4 5 6 8	27	11	8					
10 13	33	13	9					
15 16 20 25 30	39	16	11					
40 50	46	19	13					
60 70 80 85	54	22	15					
100	63	25	18					

Note) This table does not apply to models LBG, LBGT, LTR and LTR-A.

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High Torque Caged Ball Spline

Models SLS, SLS-L and SLF

A3-36



Fig.1 Structure of High Torque Caged Ball Spline

Point of Selection	⊠3-6
Point of Design	⊠3-117
Options	⊠3-120
Model No.	⊠3-122
Precautions on Use	⊠3-123
Accessories for Lubrication	⊠24-1
Mounting Procedure and Maintenance	B 3-30
Cross-sectional Characteristics of the Spline Shaft	⊠3-17
Equivalent factor	⊠3-27
Clearance in the Rotation Direction	⊠3-30
Accuracy Standards	⊠3-34
Maximum Manufacturing Length by Accuracy	⊠3-115
Structure and Features

The caged-ball technology, developed by bringing together THK's technologies and know-how, is now integrated in the new Ball Spline.

The integration of the ball cage enables the circulating motion of evenly spaced balls and highspeed response to be achieved.

It eliminates collision and mutual friction between balls, and realizes low noise, pleasant running sound and low particle generation. As the grease retention is increased, long-term maintenance-free operation is also achieved.

The high-torque design provides the nut with excellent torsional rigidity. The spline shaft also has enhanced rigidity, thanks to its rounded design.



[High-speed Response]

Models SLS/SLF adopt the caged-ball technology to enable the circulating motion of evenly spaced balls to be maintained and high-speed response to be achieved, the cycle time of the machine can be improved.

[Conditions]

Model tested	SLS50			
Testing environment	22 to 27.5℃			
Stroke	1000mm			
Maximum speed	200m/min			
Acceleration/deceleration	5G(49m/s²)			
Applied load	Light preload(CL)			
Lubricant	THK AFB-LF Grease			

Appearance of the test machine (high-speed durability test)



Appearance of the specimen



[Test results]

No anomaly after travelling 10,000 km

[Improvement on the spline shaft's rigidity]

Redesigning the shape of the conventional high torque type spline shaft to be more circular significantly improves its torsion and flexural rigidity.

		Unit: mm
Nominal shaft diameter 25	LBS	SLS
Minor diameter ϕ d	19.5	21.6
Major diameter ϕ D ₀	24.5	25.0
Ball center-to-center diameter ϕ dp	25	25.2



501HX

A 3-37

Ball Spline

[Low Noise, Pleasant Running Sound and Low Particle Generation]

Models SLS/SLF adopt the caged-ball technology, they eliminate collision and mutual friction between balls, and realize low noise, pleasant running sound and low particle generation.

[Conditions]

Model tested	SLF50/LBF50				
Stroke	600mm				
Speeds	30,50,100,150m/min				
Measuring instrument	Noise level meter				

Overview of the test machine





Noise level comparison

[Conditions]

Model tested	SLF50CL+350LP/ LBS50CL+350LP				
Maximum speed	30m/min				
Acceleration	2.84m/s ²				
Stroke	200mm				
Amount of air supplied	1ℓ/200sec				
Lubricant	THK AFE-CA Grease				
Equipment using the product	Particle counter				



Appearance of the test machine







Ball Spline

[Long-term Maintenance-free Operation]

Models SLS/SLF adopt the caged-ball technology to substantially increase the grease retention, thus achieving long-term maintenance-free operation.

[Smooth Motion (Small Rolling Fluctuation)]

Models SLS/SLF adopt the caged-ball technology and a new circulation method, thus achieving stable and smooth motion with small rolling fluctuation.

[Conditions]

Model tested	SLF50
Speed	10mm/sec
Applied load	Medium preload(CM)
Lubricant	THK AFB-LF Grease



Rolling resistance test



Types and Features

[Types of Spline Nuts]

Cylindrical Type Ball Spline Model SLS (Medium Load Type)

The circumference of the spline nut is shaped in a straight cylinder.

Using a key, this model can be secured to the housing, or transmit a torque.



Cylindrical Type Ball Spline Model SLS-L (Heavy Load Type)

A heavy-load type with the same outer diameter as model SLS and a longer spline nut. It is optimal in cases where a large torque is applied in a small space, and in cases where an overhang load or moment is applied.

Specification Table⇒▲3-42



Flanged Type Ball Spline Model SLF

The housing can be secured with bolts on models equipped with a flange.

This model is easily assembled and can accommodate a shorter housing compared to models with housing secured by a key.

Specification Table⇒A3-44



▲3-40 1元出版

[Types of Spline Shafts]

Precision Solid Spline Shaft (Standard Type)

The spline shaft is cold-drawn and its raceway is precision ground. It is used in combination with a spline nut.



Special Spline Shaft

THK manufactures a spline shaft with thicker ends or thicker middle area through special processing at your request.

Hollow Spline Shaft (Type K)

A drawn, hollow spline shaft is available for requirements such as piping, wiring, air-vent and weight reduction.

Housing Inner-diameter Tolerance

When fi tting the spline nut to the housing, transition fi t is normally recommended. If the accuracy of the Ball Spline does not need to be very high, clearance fi tting is also acceptable.

Table1 Housing Inner-diameter Tolerance

Housing Inner-diameter	General conditions	H7
Tolerance	When clearance needs to be small	J6







Model SLS



		Spline nut dimensions										
Model No.	Outer	diameter	Le	ength	Key	way dimens	ions	Greasing hole				
	D	Tolerance	L	Tolerance	b H8	t +0.1 0	lo	d₀				
SLS25 SLS25L	37	0	60 70		5	3	33	2				
SLS30 SLS30L	45	-0.016	70 80		7	4	41	3				
SLS40 SLS40L	60	0	90 100	0	10	4.5	55	3				
SLS50 SLS50L	75	-0.019	100 112	-0.3	15	5	60	4				
SLS60 SLS60L	90		127 140		18	6	68	4				
SLS70 SLS70L	100	0 -0.022	110 135		18	6	68	4				
SLS80 SLS80L	120		140 155	0	20	7	80	5				
SLS100 SLS100L	140	0 -0.025	160 185	-0.4	28	9	93	5				

Model number coding



in the rotational direction

(*2)

Symbol for standard hollow spline shaft (*4)

Accuracy symbol (*3)

Contamination protection Overall spline shaft length (*5) accessory symbol (*1)

(in mm)

Number of spline nuts on one shaft (no symbol for one nut)

(*1)See A3-120. (*2) See A3-30. (*3) See A3-34. (*4) See A3-46. (*5) See A3-115.

▲3-42 1元出长

High Torque Caged Ball Spline



Unit: mm

Ball Spline

Basic torque rating		Basic load rating			rmissible nent	Mass		
C⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _{A1} N-m	M _{A2} N-m	Spline Nut kg	Spline shaft kg/m	
219.9 261.9	306.8 394.5	18.2 21.7	22.5 29.0	136 220	851 1203	0.15 0.18	3.51	
366.5 416.4	513.3 616.0	25.4 28.9	31.5 37.8	233 330	1341 1803	0.30 0.34	5.05	
818.9 890.0	1135.4 1277.3	42.8 46.5	52.5 59.1	520 652	2801 3529	0.69 0.79	9.18	
1373.4 1571.2	1783.1 2165.2	57.6 65.9	66.2 80.4	687 996	4156 5349	1.30 1.47	14.45	
2506.7 2723.2	3321.0 3736.2	87.8 95.3	103.0 115.8	1452 1820	7733 9570	2.25 2.50	21.23	
2986.3 3708.4	3474.7 4738.2	89.7 111.4	92.5 126.1	1038 1867	6392 10135	2.13 2.71	28.57	
4664.6 5195.3	5477.4 6390.4	122.8 136.8	127.7 148.9	1739 2327	11482 14491	4.22 4.77	37.49	
8922.3 10424.4	10211.6 12764.6	188.2 219.8	190.7 238.4	3155 4816	19118 26463	5.20 6.22	58.97	





		Spline nut dimensions										
Model No.	Outer	r diameter	L	ength	Flange	e diameter			Greasing hole			
	D	Tolerance	L	Tolerance	D1	Tolerance	н	F	d₀	PCD		
SLF25	37	0	60		60		9	21	2	47		
SLF30	45	-0.016	70		70	0 _0.2	10	25	3	54		
SLF40	60		90	0	90		14	31	3	72		
SLF50	75	0 -0.019	100	-0.3	113		16	34	4	91		
SLF60	90		127		129	0	18	45.5	4	107		
SLF70	100	0	110		142	0.3	20	35	4	117		
SLF80	120	-0.022	140	0	168		22	48	5	138		
SLF100	140	0 -0.025	160	-0.4	195	0 -0.4	25	55	5	162		

Model number coding

A3-44 1元出K



(*1) See 🖾 3-120. (*2) See 🖾 3-30. (*3) See 🖾 3-34. (*4) See 🖾 3-46. (*5) See 🖾 3-115.

Download data by searching for the corresponding model number on the Technical Support site.



Unit: mm

		Basic torque rating		Basic load rating			rmissible nent	Mass	
	Mounting hole								
	$d_1 \times d_2 \times h$	C⊤ N-m	С₀т N-m	C kN	C₀ kN	M _{A1} N-m	M _{A2} N-m	Spline Nut kg	Spline shaft kg/m
	5.5×9.5×5.4	219.9	306.8	18.2	22.5	136	851	0.26	3.51
	6.6×11×6.5	366.5	513.3	25.4	31.5	233	1341	0.45	5.05
	9×14×8.6	818.9	1135.4	42.8	52.5	520	2801	1.06	9.18
	11×17.5×11	1373.4	1783.1	57.6	66.2	687	4156	1.90	14.45
	11×17.5×11	2506.7	3321.0	87.8	103.0	1452	7733	3.08	21.23
	14×20×13	2986.3	3474.7	89.7	92.5	1038	6392	3.25	28.57
	16×23×15.2	4664.6	5477.4	122.8	127.7	1739	11482	5.82	37.49
	18×26×17.5	8922.3	10211.6	188.2	190.7	3155	19118	7.66	58.97

Spline Shaft

Spline shafts are divided in shape into precision solid spline shaft, special spline shaft and hollow spline shaft (type K), as described on **A3-41**.

Since production of a spline shaft with a specific shape is performed at your request, provide a drawing of the desired shaft shape when asking an estimate or placing an order.

[Sectional Shape of the Spline Shaft]

Table2 shows the sectional shape of a splineshaft. If the spline shaft ends need to be cylindrical, the minor diameter (ϕ d) value should not be exceeded if possible.



Nominal shaft diameter	25	30	40	50	60	70	80	100
Minor diameter ϕ d	21.6	25.8	35.2	44.4	54.0	62.8	71.3	90.0
Major diameter <i>φ</i> D₀ h7	25.0	30.0	40.0	50.0	60.0	60.0	80.0	100.0
Ball center-to-center diameter ϕ dp	25.2	30.2	40.6	50.6	61.0	71.0	80.8	101.2
Mass(kg/m)	3.51	5.05	9.18	14.45	21.23	28.57	37.49	58.97

Table2 Sectional Shape of the Spline Shaft

*The minor diameter ϕ d must be a value at which no groove is left after machining.

[Hole Shape of the Standard Hollow Type Spline Shaft]

Table3 shows the hole shape of the standard hollow type spline shaft. Use this table when a requirement such as piping, wiring, air-vent or weight reduction needs to be met.



Table3 Sectional Shape of the Standard Hollow Type Spline Shaft

Unit: mm

Unit⁻ mm

Nominal shaft diameter	25	30	40	50	60	70	80	100
Minor diameter ϕ d	21.6	25.8	35.2	44.4	54.0	62.8	71.3	90.0
Major diameter ϕ D ₀ h7	25.0	30.0	40.0	50.0	60.0	60.0	80.0	100.0
Ball center-to-center diameter ϕ dp	25.2	30.2	40.6	50.6	61.0	71.0	80.8	101.2
Hole diameter(ϕ d ₄)	12	16	22	25	32	_	52.5	67.5
Mass(kg/m)	2.62	3.47	6.19	10.59	14.90	_	20.48	30.85

*The minor diameter ϕ d must be a value at which no groove is left after machining.



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[Chamfering of the Spline Shaft Ends]

To facilitate the insertion of the spline shaft into a spline nut, the shaft ends are normally chamfered with the dimensions indicated below unless otherwise specified.

The ends are chamfered whether they are used, such as with stepped, tapped, or drilled ends, or not used, such as with cantilevered supports.



SLS25 to 100

Table 4 Chamfer Dimensions of Spline Shaft Ends

Nominal shaft diameter	25	30	40	50	60	70	80	100
Chamfer C	0.5	0.5	1.0	1.0	2.0	2.0	2.0	2.0

[Length of the Incomplete Area of a Special Spline Shaft]

If any part of the spline shaft is thicker than the minor diameter (ϕ d), an area with incomplete splines is required to secure a recess for grinding. The relationship between the flange diameter (ϕ df) and the length of incomplete splines (S) is shown in Table 5.



Table 5 Length of Incomplete Spline Area: S

Unit: mm

				-	-	-					•••••
Flange diameter ¢df	25	30	35	40	50	60	80	100	120	140	160
Nominal shaft diameter											
25	29	54	63	72	—	—	—	—	—	—	—
30	—	34	56	65	80	—	—	—	—	—	—
40	—	—	—	36	66	81	104	—	_	—	—
50	—	—	—	—	35	59	83	100	—	—	—
60	—	—	—	_	—	37	73	92	108	—	—
70	—	—	—	—	—	—	62	84	101	115	—
80	_	_	_	_	_	_	45	76	95	109	_
100	—	—	—	—	—	—	_	48	77	96	110



Ball Spline

Unit: mm

Accessories

Ball Spline models SLS and SLS-L are provided with a standard key as indicated in Table6.



Table6 Standard Keys for Models SLS and SLS-L

Nominal shaft		Width b		Height h		Length <i>l</i>	R	с
diameter		Tolerance(p7)		Tolerance(h9)		Tolerance(h12)	ĸ	C
SLS 25 SLS 25L	5	+0.024 +0.012	5	0 -0.030	33	0	2.5	0.5
SLS 30 SLS 30L	7	+0.030	7		41	-0.250	3.5	0.5
SLS 40 SLS 40L	10	+0.015	8	0 -0.036	55		5	0.8
SLS 50 SLS 50L	15		10		60	0	7.5	0.0
SLS 60 SLS 60L	18	+0.036 +0.018	12		68	-0.300	9	
SLS 70 SLS 70L	10		12	0	00		9	1.2
SLS 80 SLS 80L	20	+0.043	13	-0.043	80	0 -0.350	14	1.2
SLS 100 SLS 100L	28	+0.022	18		93	0 -0.400	14	

Unit: mm

A3-48 17HK

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High Torque Caged Ball Spline

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High Torque Type Ball Spline

Models LBS, LBST, LBF, LBR and LBH



Fig.1 Structure of High Torque Type Ball Spline Model LBS

Point of Selection	A3-6
Point of Design	⊠3-117
Options	⊠3-120
Model No.	⊠3-122
Precautions on Use	⊠3-123
Accessories for Lubrication	⊠24-1
Mounting Procedure and Maintenance	₿3-30
Cross-sectional Characteristics of the Spline Shaft	⊠3-17
Equivalent factor	⊠3-27
Clearance in the Rotation Direction	⊠3-30
Accuracy Standards	⊠3-34
Maximum Manufacturing Length by Accuracy	⊠3-115

Structure and Features

With the high torque type Ball Spline, the spline shaft has three crests positioned equidistantly at 120°, and along both sides of each crest, two rows of balls (six rows in total) are arranged so as to hold the crest, as shown in Fig.1.

The raceways are precision ground into R-shaped grooves whose diameters are approximate to the ball diameter. When a torque is generated from the spline shaft or the spline nut, the three rows of balls on the load-bearing side evenly receive the torque, and the center of rotation is automatically determined. When the rotation reverses, the remaining three rows of balls on the unloaded side receive the torque.

The rows of balls are held in a retainer incorporated in the spline nut so that they smoothly roll and circulate. With this design, balls will not fall even if the spline shaft is removed from the nut.

[No Angular Backlash]

With the high torque type Ball Spline, a single spline nut provides a preload to eliminate angular backlash and increase the rigidity.

Unlike conventional ball splines with circular-arc groove or Gothic-arch groove, the high torque type Ball Spline eliminates the need for twisting two spline nuts to provide a preload, thus allowing compact design to be achieved easily.

[High Rigidity and Accurate Positioning]

Since this model has a large contact angle and provides a preload from a single spline nut, the initial displacement is minimal and high rigidity and high positioning accuracy are achieved.

[High-speed Motion, High-speed Rotation]

Adoption of a structure with high grease retention and a rigid retainer enables the ball spline to operate over a long period with grease lubrication even in high-speed straight motion. Since the distance in the radius direction is almost uniform between the loaded balls and the unloaded balls, the balls are little affected by the centrifugal force and smooth straight motion is achieved even during high-speed rotation.

[Compact Design]

Unlike conventional ball splines, unloaded balls do not circulate on the outer surface of the spline nut with this model. As a result, the outer diameter of the spline nut is reduced and a space-saving and compact design is achieved.

[Ball Retaining Type]

Use of a retainer prevents the balls from falling even if the spline shaft is pulled out of the spline nut.

[Can be Used as a Linear Bushing for Heavy Loads]

Since the raceways are machined into R grooves whose diameter is almost equal to the ball diameter, the contact area of the ball is large and the load capacity is large also in the radial direction.

[Double, Parallel Shafts can be Replaced with a Single Shaft]

Since a single shaft is capable of receiving a load in the torque direction and the radial direction, double shafts in parallel configuration can be replaced with a single-shaft configuration. This allows easy installation and achieves space-saving design.



Applications

The high torque type Ball Spline is a reliable straight motion system used in a wide array of applications such as the columns and arms of industrial robot, automatic loader, transfer machine, automatic conveyance system, tire forming machine, spindle of spot welding machine, guide shaft of highspeed automatic coating machine, riveting machine, wire winder, work head of electric discharge machine, spindle drive shaft of grinding machine, speed gears and precision indexing shaft.

A3-52 冗出比

Types and Features

[Types of Spline Nuts]

Cylindrical Type Ball Spline Model LBS (Medium Load Type)

The most compact type with a straight cylindrical spline nut. When transmitting a torque, a key is driven into the body. The outer surface of the spline nut is provided with anti-carbonation treatment.



Ball Spline

Cylindrical Type Ball Spline Model LBST (Heavy Load Type)

A heavy load type that has the same spline nut diameter as model LBS, but has a longer spline nut length. It is optimal for locations where the space is small, a large torque is applied, and an overhang load or moment load is applied.

Specification Table⇒▲3-60

Specification Table⇒▲3-62



Flanged Type Ball Spline Model LBF

The spline nut can be attached to the housing via the flange, making assembly simple. It is optimal for locations where the housing may be deformed if a keyway is machined on its surface, and where the housing width is small.

69



Flanged Type Ball Spline Model LBR

Based on the heavy load type model LBST, this model has a flange in the central area, making itself optimal for locations under a moment load such as arms of industrial robots.



Rectangular Type Ball Spline Model LBH

Its rigid rectangular spline nut does not require a housing and can be directly mounted on the machine body. Thus, a compact, highly rigid linear guide system is achieved.



Specification Table⇒▲3-66

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[Types of Spline Shafts]

Precision Solid Spline Shaft (Standard Type)

The spline shaft is cold-drawn and its raceway is precision ground. It is used in combination with a spline nut.



Special Spline Shaft

THK manufactures a spline shaft with thicker ends or thicker middle area through special processing at your request.

Hollow Spline Shaft (Type K)

A drawn, hollow spline shaft is available for requirements such as piping, wiring, air-vent and weight reduction.

Housing Inner-diameter Tolerance

When fitting the spline nut to the housing, transition fit is normally recommended. If the accuracy of the Ball Spline does not need to be very high, clearance fitting is also acceptable.

Table1 Housing Inner-diameter Tolerance

Housing Inner-diameter	General conditions	H7
Tolerance	When clearance needs to be small	J6







Model LBS (Medium Load Type)





				Spl	line nut	dimen	sions					
Model No.	Oute	er diameter	Length					Keyway dimensions				
	D	Tolerance	L	Tolerance	L2	L3	D ₂	b H8	t +0.1 0	lo	r	С
LBS 6	12	0	20	1	20.8	11	11.5	2	0.8	10	—	0.3
LBS 8	16	-0.011	25	0 -0.2	26.4	14.5	15.5	2.5	1.2	12.5	—	0.3
LBS 10	19	0 -0.013	30	0	-		-	3	1.5	17		0.3

Note) Models LBS6 and 8 are end cap types. Please refrain from subjecting them to impacts, etc.

Model number coding	3S6 UU CL +2(лој н
т —	del No. Symbol for clearance in the rotational direction (*2)	Accuracy symbol (*3)
Number of s on one shaf (no symbol) Joerall spline shaft length (*4) in mm)
	(*1) See Δ3-120 . (*2)	See 🖾 3-30. (*3) See 🖾 3-34. (*4) See 🖾 3-115.

A3-56

High Torque Type Ball Spline



Model LBS10





Unit: mm

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		e shaft iameter	Basic torque rating		Basic load rating (radial)		Static pe mon	rmissible nent	Mass	
Greasing hole										
d₀	D₀	d₅	C⊤ N-m	C₀⊤ N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline nut kg	Spline shaft kg/m
1.2	6	5.3	1.53	2.41	0.637	0.785	2.2	19.4	0.0066	0.22
1.2	8	7.3	4.07	6.16	1.18	1.42	5.1	39.6	0.0154	0.42
1.5	10	8.3	7.02	10.4	1.62	1.96	8.1	67.6	0.0367	0.55

Note) **M_{A1} indicates the permissible moment in the axial direction when a single spline nut is used. **M_{A2} indicates the allowable moment load in the axial direction when using two spline nuts in contact with each other. (The accuracy of a configuration consisting of a single LBS unit will not be stable. We recommend using a single LBST unit or two LBS units in close contact with each other.) For details on the maximum lengths of ball spline shafts by accuracy, please see **⊠3-115**.

Model LBS (Medium Load Type)





Models LBS15 to 100

				Spl	line nut dimensions								
Model No.	Oute	er diameter		Length				Keywa	Keyway dimensions				
	D	Tolerance	L	Tolerance	L2	L3	D2	b H8	t +0.1 0	lo	r	С	
LBS 15	23	0 -0.013	40	0	—	_	_	3.5	2	20	0.5	—	
○● LBS 20	30		50	-0.2	_	_	_	4	2.5	26	0.5	_	
○● LBS 25	37	0 _0.016	60		—	—	—	5	3	33	0.5	—	
○● LBS 30	45		70		—	—	—	7	4	41	1	—	
○● LBS 40	60	0	90	0 -0.3	—	—	_	10	4.5	55	1	—	
○● LBS 50	75	-0.019	100		—	—	—	15	5	60	1.5	—	
○● LBS 70	100	0	110		—	—	—	18	6	68	2	—	
○● LBS 85	120	-0.022	140	0	—	—	—	20	7	80	2.5	—	
○● LBS 100	140	0 -0.025	160	-0.4	_	_	_	28	9	93	3	_	

Note) 〇: Model numbers able to handle high temperatures (metal retainers: operating temperature up to 100°C) Compatible model numbers: LBS20 to 100

(Example) LBS20 A CL+500L H

High temperature symbol

EMODE NUMBERS COMPATIBLE WITH Felt seals. Compatible model numbers: LBS20 to 100 Felt seals cannot be attached to ball spline models using metal retainers. When equipping felt seals, the length dimensions of the nuts will change.





Unit: mm

		e shaft iameter	Basic tore	que rating		ad rating Jial)		rmissible nent	Ma	ass
Greasing hole										
d₀	D₀	d₅	C⊤ N-m	C₀⊤ N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut kg	Spline shaft kg/m
2	14.5	—	30.4	74.5	4.4	8.4	25.4	185	0.06	1
2	19.7	_	74.5	160	7.8	14.9	60.2	408	0.14	1.8
2	24.5	_	154	307	13	23.5	118	760	0.25	2.7
3	29.6	—	273	538	19.3	33.8	203	1270	0.44	3.8
3	39.8	—	599	1140	31.9	53.4	387	2640	1	6.8
4	49.5	—	1100	1940	46.6	73	594	4050	1.7	10.6
4	70	—	2190	3800	66.4	102	895	6530	3.1	21.3
5	84	_	3620	6360	90.5	141	2000	12600	5.5	32
5	99	_	5190	12600	126	237	3460	20600	9.5	45

Note) **M_{A1} indicates the permissible moment value in the axial direction when a single spline nut is used. **M_{A2} indicates the allowable moment load value in the axial direction when using two spline nuts in contact with each

**M_{A2} indicates the allowable moment load value in the axial direction when using two spline nuts in contact with each other.

(Single LBS-unit configuration is not stable in accuracy. We recommend using a single LBST unit or two LBS units in For details on the maximum lengths of ball spline shafts by accuracy, please see **A3-115**.

Model LBST (Heavy Load Type)



				Spline nut dim	ensions						
Model No.	Out	er diameter		Length	Keyw	ay dimer	isions		Greasing hole		
Model No.	D	Tolerance	L	Tolerance	b H8	t +0.1 0	lo	r	do		
O● LBST 20	30	0	60	0 0.2	4	2.5	26	0.5	2		
O LBST 25	37	-0.016	70		5	3	33	0.5	2		
O LBST 30	45		80		7	4	41	1	3		
O LBST 40	60	0	100	0	10	4.5	55	1	3		
O LBST 50	75	-0.019	112	-0.3	15	5	60	1.5	4		
O LBST 60	90		127		18	6	68	1.5	4		
O LBST 70	100	0	135		18	6	68	2	4		
○● LBST 85	120	0.011	155	0	20	7	80	2.5	5		
○● LBST 100	140	0	175	-0.4	28	9	93	3	5		
O LBST 120	160	-0.025	200	0	28	9	123	3.5	6		
O LBST 150	205	0 -0.029	250	-0.5	32	10	157	3.5	6		

Note) : indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C).

(Example) LBST25 A CM+400L H

— High temperature symbol

•: indicates model numbers for which felt seal types are available (see **3-120**).

A felt seal cannot be attached to Ball Spline models using metal retainer.

Model number coding

A3-60

2 LBST50 Model No.	Symbol for clearance in the rotational direction	Accuracy symbol
Number of spline nuts on one shaft (no symbol for one nut)	accessory symbol (i	iverall spline shaft length (*5) n mm)
(*1) See I	A3-120. (*2) See A3-3	30. (*3) See 🖾 3-34. (*4) See 🖾 3-69. (*5) See 🖾 3-115

Download data by searching for the corresponding model number on the Technical Support site.



Unit: mm

Ball Spline

Basic tore	que rating	Basic load ra	ating (radial)	Static permis	sible moment	Ma	ass
C⊤ N-m	C₀⊤ N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut kg	Spline shaft kg/m
90.2	213	9.4	20.1	103	632	0.17	1.8
176	381	14.9	28.7	171	1060	0.29	2.7
312	657	22.5	41.4	295	1740	0.5	3.8
696	1420	37.1	66.9	586	3540	1.1	6.8
1290	2500	55.1	94.1	941	5610	1.9	10.6
1870	3830	66.2	121	1300	8280	3.3	15.6
3000	6090	90.8	164	2080	11800	3.8	21.3
4740	9550	119	213	3180	17300	6.1	32
6460	14400	137	271	4410	25400	10.4	45
8380	19400	148	306	5490	32400	12.9	69.5
13900	32200	196	405	8060	55400	28	116.6

Note) **M_{A1} indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above. **M_{A2} indicates the permissible moment value in the axial direction when two spline nuts in close contact with each other are used, as shown in the figure above. For details on the maximum lengths of ball spline shafts by accuracy, please see **⊠3-115**.

Model LBF (Medium Load Type)





				Splir	mensions					
Model No.	Outer	diameter	Length		Flange	Flange diameter			Greasing hole	
	D	Tolerance	L	Tolerance	D1	Tolerance	н	F	d₀	PCD
LBF 15	23	0 0.013	40	0	43		7	13	2	32
○● LBF 20	30		50	-0.2	49		7	18	2	38
○● LBF 25	37	0 0.016	60		60	0 0.2	9	21	2	47
○● LBF 30	45		70		70		10	25	3	54
○● LBF 40	57		90	0	90		14	31	3	70
○● LBF 50	70	0 0.019	100	-0.3	108		16	34	4	86
O LBF 60	85		127		124	0	18	45.5	4	102
○● LBF 70	95	0	110		142	-0.3	20	35	4	117
○● LBF 85	115	-0.022	140	0	168		22	48	5	138
○● LBF 100	135	0 0.025	160	-0.4	195	0 0.4	25	55	5	162

Note) : indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C). (Example) LBF20 A CL+500L H

- High temperature symbol

•: indicates model numbers for which felt seal types are available (see **3-120**). A felt seal cannot be attached to Ball Spline models using metal retainer.





Unit: mm

	Basic toro	que rating		ad rating lial)	Static pe mon	rmissible nent	Ma	ass
Mounting hole								
$d_1 \times d_2 \times h$	C⊤ N-m	C₀⊤ N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut kg	Spline shaft kg/m
4.5×8×4.4	30.4	74.5	4.4	8.4	25.4	185	0.11	1
4.5×8×4.4	74.5	160	7.8	14.9	60.2	408	0.2	1.8
5.5×9.5×5.4	154	307	13	23.5	118	760	0.36	2.7
6.6×11×6.5	273	538	19.3	33.8	203	1270	0.6	3.8
9×14×8.6	599	1140	31.9	53.4	387	2640	1.2	6.8
11×17.5×11	1100	1940	46.6	73	594	4050	1.9	10.6
11×17.5×11	1870	3830	66.2	121	1300	8280	3.5	15.6
14×20×13	2190	3800	66.4	102	895	6530	3.6	21.3
16×23×15.2	3620	6360	90.5	141	2000	12600	6.2	32
18×26×17.5	5910	12600	126	237	3460	20600	11	45

Note) **M_{A1} indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above.

figure above. ** M_{A_2} indicates the permissible moment value in the axial direction when two spline nuts in close contact with each other are used, as shown in the figure above.

(Single spline nut configuration is not stable in accuracy. We recommend using two spline nuts in close contact with each other.)

For details on the maximum lengths of ball spline shafts by accuracy, please see **3-115**.

Model LBR



	Spline nut dimensions										
Model No.	Oute	er diameter	Outerdiameter	Le	ength	Flangediameter					
	D	Tolerance	D₃	L	Tolerance	D1	н	E	PCD		
LBR 15	25	0 0.013	25.35	40	0	45.4	9	15.5	34		
○● LBR 20	30		30.35	60	-0.2	56.4	12	24	44		
○● LBR 25	40	0 0.016	40.35	70		70.4	14	28	54		
○● LBR 30	45	0.016	45.4	80]	75.4	16	32	61		
○● LBR 40	60	0	60.4	100	0	96.4	18	41	78		
○● LBR 50	75	-0.019	75.4	112	-0.3	112.4	20	46	94		
O LBR 60	90		90.5	127		134.5	22	52.5	112		
○● LBR 70	95	0 -0.022	95.6	135		140.6	24	55.5	117		
○● LBR 85	120		120.6	155	0	170.6	26	64.5	146		
⊖● LBR 100	140	0 0.025	140.6	175	-0.4	198.6	34	70.5	170		

Note)
O: indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C). (Example) LBR40 A CM+600L H

_____ High temperature symbol

•: indicates model numbers for which felt seal types are available (see **A3-120**). A felt seal cannot be attached to Ball Spline models using metal retainer.

Model number coding UU СМ 2 LBR30 +700L н Κ Model No. Symbol for clearance Accuracy symbol in the rotational direction (*3) Symbol for spline shaft (*4) (*2) Number of spline nuts Contamination protection Overall spline shaft length (*5) on one shaft accessory symbol (in mm) (no symbol for one nut) (*1) (*1) See A3-120. (*2) See A3-30. (*3) See A3-34. (*4) See A3-69. (*5) See A3-115. ▲3-64 17日米 Download data by searching for the corresponding model number on the Technical Support site. https://tech.thk.com



Unit: mm

Mounting hole Greasing hole			Basic tore	que rating	Basic load rating (radial)			rmissible nent	Mass	
Mounting hole		Greasing hole								
d1	F	d₀	C⊤ N-m	C₀⊤ N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut kg	Spline shaft kg/m
4.5	7.5	2	30.4	74.5	4.4	8.4	25.4	185	0.14	1
5.5	12	2	90.2	213	9.4	20.1	103	632	0.33	1.8
5.5	14	2	176	381	14.9	28.7	171	1060	0.54	2.7
6.6	16	3	312	657	22.5	41.4	295	1740	0.9	3.8
9	20.5	3	696	1420	37.1	66.9	586	3540	1.7	6.8
11	23	4	1290	2500	55.1	94.1	941	5610	2.7	10.6
11	26	4	1870	3830	66.2	121	1300	8280	3.7	15.6
14	27	4	3000	6090	90.8	164	2080	11800	6	21.3
16	32	5	4740	9550	119	213	3180	17300	8.3	32
18	35	5	6460	14400	137	271	4410	25400	14.2	45

Note) **M_{A1} indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above. **M_{A2} indicates the permissible moment value in the axial direction when two spline nuts in close contact with each

other are used, as shown in the figure above. For details on the maximum lengths of ball spline shafts by accuracy, please see **II3-115**.

Model LBH





		Spline nut dimensions												
Model No.	Height	eight Width Length J W1												
	м	W	L	В	С	S×ℓ	±0.15	±0.15	т	к				
O LBH 15	29	34	43	26	26	M4×10	15	17	6	20				
○● LBH 20	38	48	62	35	35	M6×12	20	24	7	26				
○● LBH 25	47	60	73	40	40	M8×16	25	30	8	33				
○● LBH 30	57	70	83	50	50	M8×16	30	35	10	39				
○● LBH 40	70	86	102	60	60	M10×20	38	43	15	50				
○● LBH 50	88	100	115	75	75	M12×25	48	50	18	63				

Note) ○: indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100℃).

(Example) LBH30 A CM+600L H

▲3-66 101米

High temperature symbol

•: indicates model numbers for which felt seal types are available (see **M3-120**). A felt seal cannot be attached to Ball Spline models using metal retainer.

Model number coding UU CL +700L LBH40 2 Ρ Κ Model No. Symbol for clearance Accuracy symbol in the rotational direction (*3) Symbol for spline shaft (*4) (*2) Number of spline nuts Contamination protection Overall spline shaft length (*5) on one shaft accessory symbol (in mm) (no symbol for one nut) (*1) (*1) See A3-120. (*2) See A3-30. (*3) See A3-34. (*4) See A3-69. (*5) See A3-115.

Ball Spline



			Basic tore	que rating	Basic loa (rad	ad rating dial)	Static permissible moment		
R	N	Grease nipple	C⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _A ** N-m	Spline Nut	Spline shaft kg/m
14	5		30.4	74.5	4.4	8.4	25.4	0.23	1
18	7	A-M6F	90.2	213	9.4	20.1	103	0.58	1.8
22	6	A-M6F	176	381	14.9	28.7	171	1.1	2.7
26	8	A-M6F	312	657	22.5	41.4	295	1.73	3.8
32	10	A-M6F	696	1420	37.1	66.9	586	3.18	6.8
40	13.5	A-PT1/8	1290	2500	55.1	94.1	941	5.1	10.6

Note) **MA indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the For details on the maximum lengths of ball spline shafts by accuracy, please see **II3-115**.

Model LBS with Recommended Shaft End Shape



Unit: mm

Model No.	d₃	Tolerance	l2	Sм	l4	S×ℓ₅
LBS 15	10	0 0.015	23	M10×1.25	14	M6×10
LBS 20	14	0	30	M14×1.5	18	M8×15
LBS 25	18	-0.018	42	M18×1.5	25	M10×18
LBS 30	20	0	46	M20×1.5	27	M12×20
LBS 40	30	-0.021	70	M30×2	40	M18×30
LBS 50	36	0 -0.025	80	M36×3	46	M20×35

Note) For details of chamfer A, see 3-70.

511E

Spline Shaft

Spline shafts are divided in shape into precision solid spline shaft, special spline shaft and hollow spline shaft (type K), as described on **▲3-55**.

Since production of a spline shaft with a specific shape is performed at your request, provide a drawing of the desired shaft shape when asking an estimate or placing an order.

[Sectional Shape of the Spline Shaft]

Table2 shows the sectional shape of a spline shaft. If the spline shaft ends need to be cylindrical, the minor diameter (ϕ d) value should not be exceeded if possible.



Unit: mm

Nominal shaft diameter	15	20	25	30	40	50	60	70	85	100	120	150
Minor diameter ød	11.7	15.3	19.5	22.5	31	39	46.5	54.5	67	81	101	130
Major diameter ϕD_0	14.5	19.7	24.5	29.6	39.8	49.5	60	70	84	99	117	147
Ball center-to-center diameter ødp	15	20	25	30	40	50	60	70	85	100	120	150
Mass (kg/m)	1	1.8	2.7	3.8	6.8	10.6	15.6	21.3	32	45	69.5	116.6

Table2 Sectional Shape of the Spline Shaft

*The minor diameter ϕ d must be a value at which no groove is left after machining.

[Hole Shape of the Standard Hollow Type Spline Shaft]

Table3 shows the hole shape of the standard hollow type spline shaft. Use this table when a requirement such as piping, wiring, air-vent or weight reduction needs to be met.



Table3 Sectional Shape of the Standard Hollow Type Spline Shaft Unit: m												
Nominal shaft diameter	20	25	30	40	50	60	70	85	100	120	150	
Minor diameter ød	15.3	19.5	22.5	31	39	46.5	54.5	67	81	101	130	
Major diameter ϕD_0	19.7	24.5	29.6	39.8	49.5	60	70	84	99	117	147	
Ball center-to-center diameter ødp	20	25	30	40	50	60	70	85	100	120	150	
Hole diameter (ϕ d ₄)	6	8	12	18	24	30	35	45	56	60	80	
Mass (kg/m)	1.6	2.3	2.9	4.9	7	10	13.7	19.5	25.7	47.3	77.1	

*The minor diameter \u00f6d must be a value at which no groove is left after machining.

[Chamfering of the Spline Shaft Ends]

To facilitate the insertion of the spline shaft into a spline nut, the shaft ends are normally chamfered with the dimensions indicated below unless otherwise specified.

Chamfer A

If the spline shaft ends are stepped, tapped, or drilled as in Fig. 2, they are machined with the Chamfer A dimensions indicated in Table 4.



Chamfer B

A3-70

TOHK

If either end of the spline shaft is not used, such as for cantilever support, it is machined with the chamfer B dimensions indicated in Table 4.



Fig. 3 Chamfer B

Table 4 Chamfer Dimensions of Spline Shaft Ends Unit												Unit: mm
Nominal shaft diameter	15	20	25	30	40	50	60	70	85	100	120	150
Chamfer A	1	1	1.5	2.5	3	3.5	5	6.5	7	7	7.5	8
Chamfer B	3.5	4.5	5.5	7	8.5	10	13	15	16	17	17	18

Note) Spline shafts with nominal diameters 6, 8, and 10 are chamfered to C0.5.

Unit: mm

[Length of Imperfect Area of a Special Spline Shaft]

If the middle area or the end of a spline shaft is to be thicker than the minor diameter (ϕ d), an imperfect spline area is required to secure a recess for grinding. Table5 shows the relationship between the length of the incomplete section (S) and the flange diameter (ϕ df).

(This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.)

Table5	Length	of Im	perfect	Spline	Area:	S

														· ···.	
Flange diameter ødf Nominal shaft	15	20	25	30	35	40	50	60	80	100	120	140	160	180	200
diameter															
15	32	42	49	55	60	—	—	—	—	—	—	—	—	—	—
20	—	35	43	51	57	62	—	—	—	—	—	—	—	—	—
25	—	—	51	64	74	82	97	—	—	—	—	—	—	—	—
30	_	—	—	54	67	76	92	105	—	_	—	_	—	—	—
40	—	—	—	—	—	59	80	95	119	—	—	—	—	—	—
50	—	—	—	—	—	—	63	83	110	131	—	—	—	—	—
60	_	—	—	—	—	_	—	66	100	123	140	_	—	—	—
70	_	—	—	—	—	_	—	—	89	115	134	150	—	—	—
85	_	—	—	_	—	_	_	—	61	98	122	140	_	—	—
100	—	—	—	—	—	—	—	—	—	78	108	130	147	—	—
120	_	_	_	_	_	_	_	_	_	_	81	111	133	150	—
150	—	—	_	—	—	—	_	—	_	—	—	64	101	125	144

*This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.





Accessories

Ball Spline models LBS and LBST are provided with a standard key as indicated in Table6.



Table6 Standard Keys for Models LBS and LBST

Width b Height h Length ℓ Nominal shaft С R diameter Tolerance(p7) Tolerance(h9) Tolerance(h12) 0 LBS 6 2 10 1 1.3 -0.150 +0.0160 0.3 LBS 8 2.5 +0.006 2 -0.025 12.5 1.25 0 LBS 10 3 2.5 17 -0.180 1.5 LBS 15 3.5 3.5 20 1.75 0 LBS 20 -0.210 4 +0.024 4 0 26 2 LBST 20 +0.012 -0.030 LBS 25 0.5 5 5 33 25 LBST 25 0 -0.250 LBS 30 7 7 41 3.5 LBST 30 +0.030 +0.015 LBS 40 0 10 8 55 5 LBST 40 -0.036 0.8 LBS 50 15 10 60 0 7.5 LBST 50 -0.300 +0.036 LBST 60 +0.018 LBS 70 18 12 68 9 LBST 70 LBS 85 0 20 13 80 14 1.2 -0.043 LBST 85 -0.350 +0.043 LBS 100 28 +0.022 18 93 14 LBST 100 0 LBST 120 28 18 123 14 -0.400 +0.051 0 LBST 150 32 16 20 157 2 +0.026 -0.052

四3-72 元光代

Unit: mm
High Torque Type Ball Spline

17日代 🛛 🖾 3-73

Medium Torque Type Ball Spline

Models LT, LF, LT-X and LF-X



Fig.1 Structure of Medium Torque Type Ball Spline Model LT

Point of Selection	A3-6
Point of Design	⊠3-117
Options	⊠3-120
Model No.	⊠3-122
Precautions on Use	⊠3-123
Accessories for Lubrication	⊠24-1
Mounting Procedure and Maintenance	B 3-30
Cross-sectional Characteristics of the Spline Shaft	⊠3-17
Equivalent factor	⊠3-27
Clearance in the Rotation Direction	⊠3-30
Accuracy Standards	⊠3-34
Maximum Manufacturing Length by Accuracy	⊠3-115

Structure and Features

With the medium torque type Ball Spline, the spline shaft has two to three crests on the circumference, and along both sides of each crest, two rows of balls (four or six rows in total) are arranged to hold the crest so that a reasonable preload is applied.

The rows of balls are held in a special resin retainer incorporated in the spline nut so that they smoothly roll and circulate. With this design, balls will not fall even if the nut is removed from the spline shaft.

[Large Load Capacity]

The raceways are formed into circular-arc grooves approximate to the ball curvature and ensure angular contact. Thus, this model has a large load capacity in the radial and torque directions.

[No Angular Backlash]

Two rows of balls facing one another hold a crest, formed on the circumference of the spline nut, at a contact angle of 20° to provide a preload in an angular-contact structure. This eliminates an angular backlash in the rotational direction and increases the rigidity.

[High Rigidity]

Since the contact angle is large and an appropriate preload is given, high rigidity against torque and moment is achieved.

[Ball Retaining Type]

Use of a retainer prevents the balls from falling even if the spline shaft is pulled out of the spline nut. (except for models LT4 and 5)

Ball Spline

Types and Features

[Types of Spline Nuts]

Cylindrical Type Ball Spline Model LT

The most compact type with a straight cylindrical spline nut. When transmitting a torque, a key is driven into the body.



Flanged Type Ball Spline Model LF

The spline nut can be attached to the housing via the flange, making assembly simple. It is optimal for locations where the housing may be deformed if a keyway is machined on its surface, and where the housing width is small.

Specification Table⇒▲3-80



Model LT-X Miniature Ball Spline

The nut is more compact than that of the current Model LT thanks to the new circulating pathways.

The outer diameter of the nut is the same as that of the linear bushing.

The Model LT-XL is suitable for moment loads, torque, and overhung loads that exceed those tolerated by the Model LT-X.

Model LF-X Miniature Ball Spline

The nut is more compact than that of the current Model LF thanks to the new circulating pathways.

The outer diameter of the nut is the same as that of the linear bushing.

The Model LF-XL is suitable for moment loads, torque, and overhung loads that exceed those tolerated by the Model LF-X.

Specification Table⇒A3-82

Specification Table⇒▲3-84







Medium Torque Type Ball Spline

[Types of Spline Shafts]

Precision Solid Spline Shaft (Standard Type)

The raceway of the spline shaft is precision ground. It is used in combination with a spline nut.



Special Spline Shaft

THK manufactures a spline shaft with thicker ends or thicker middle area through special processing at your request.

Hollow Spline Shaft (Type K)

A drawn, hollow spline shaft is available for requirements such as piping, wiring, air-vent and weight reduction.



Je



Hollow Spline Shaft (Type N)

A drawn, hollow spline shaft is available for requirements such as piping, wiring, air-vent and weight reduction.



Thin

Housing Inner-diameter Tolerance

When fitting the spline nut to the housing, transition fit is normally recommended. If the accuracy of the Ball Spline does not need to be very high, clearance fitting is also acceptable.

Table1 Housing Inner-diameter Tolerance

Housing Inner-diameter	General conditions	H7
Tolerance	When clearance needs to be small	J6



Model LT





Model LT13 or smaller

Model LT16 or greater

				Sp	line nut dim	ensions			
Model No.	Oute	er diameter	l	_ength		Keyway d	imensions		Greasing hole
	D	Tolerance	L	Tolerance	b H8	t +0.1 0	lo	r	do
Note) LT 4	10	0 0.009	16		2	1.2	6	0.5	-
Note) LT 5	12	0	20		2.5	1.2	8	0.5	—
LT 6	14	-0.011	25	0	2.5	1.2	10.5	0.5	1
LT 8	16	-0.011	25	-0.2	2.5	1.2	10.5	0.5	1.5
LT 10	21	0	33	-0.2	3	1.5	13	0.5	1.5
LT 13	24	-0.013	36		3	1.5	15	0.5	1.5
O LT 16	31	0.010	50		3.5	2	17.5	0.5	2
O LT 20	35	0	63		4	2.5	29	0.5	2
O LT 25	42	-0.016	71		4	2.5	36	0.5	3
O LT 30	47	0.010	80	0	4	2.5	42	0.5	3
O LT 40	64	0	100	-0.3	6	3.5	52	0.5	4
O LT 50	80	-0.019	125		8	4	58	1	4
O LT 60	90	0	140		12	5	67	1	5
O LT 80	120	-0.022	160	0	16	6	76	2	5
O LT 100	150	0 0.025	185	-0.4	20	7	110	2.5	5

Note) Models LT4 and 5 do not have a retainer. Do not remove the shaft from the spline nut. (It will cause balls to fall off.) : indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C).

(Example) LT20 A CL+500L H

— High temperature symbol



A3-78







Unit: mm

 Spline shaft diameter	Rows of balls	Basic toro	que rating	Basic Loa	ad Rating	Static permissible moment		Ma	Mass	
D₀ h7		C⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut g	Spline shaft kg/m	
4	4	0.59	0.78	0.44	0.61	0.88	6.4	5.2	0.1	
5	4	0.88	1.37	0.66	0.88	1.5	11.6	9.1	0.15	
6	4	0.98	1.96	1.18	2.16	4.9	36.3	17	0.23	
8	4	1.96	2.94	1.47	2.55	5.9	44.1	18	0.4	
10	4	3.92	7.84	2.84	4.9	15.7	98	50	0.62	
13	4	5.88	10.8	3.53	5.78	19.6	138	55	1.1	
16	6	31.4	34.3	7.06	12.6	67.6	393	165	1.6	
20	6	56.9	55.9	10.2	17.8	118	700	225	2.5	
25	6	105	103	15.2	25.8	210	1140	335	3.9	
30	6	171	148	20.5	34	290	1710	375	5.6	
40	6	419	377	37.8	60.5	687	3760	1000	9.9	
50	6	842	769	60.9	94.5	1340	7350	1950	15.5	
60	6	1220	1040	73.5	111.7	1600	9990	2500	22.3	
80	6	2310	1920	104.9	154.8	2510	16000	4680	39.6	
100	6	3730	3010	136.2	195	3400	24000	9550	61.8	

Note) **MA1 indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the

figure above. **M_{A2} indicates the permissible moment value in the axial direction when two spline nuts in close contact with each other are used, as shown in the figure above.

(Single LT-unit configuration is not stable in accuracy. We recommend using two units in close contact with each other.) For details on the maximum lengths of ball spline shafts by accuracy, please see **M3-115**.

_

Model LF

4-ø d1 through hole, ø d2 counter bore depth h



Model LF13 or smaller

4-ø d1 through hole, ø d2 counter bore depth h



Model LF16 or greater

						Splin	ie nut	dime	nsions	6			
Model No.	-	Outer ameter	Length		Flange diameter						Greasing hole		Mounting hole
	D	Tolerance	L	Tolerance	D1	Tolerance	Н	F	С	r	d₀	PCD	d₁×d₂×h
LF 6	14	0	25		30		5	7.5	0.5	0.5	1.5	22	3.4×6.5×3.3
LF 8	16	-0.011	25		32		5	7.5	0.5	0.5	1.5	24	3.4×6.5×3.3
LF 10	21		33] 0 [42] [6	10.5	0.5	0.5	1.5	32	4.5×8×4.4
LF 13	24	0 -0.013	36	-0.2	44		7	11	0.5	0.5	1.5	33	4.5×8×4.4
O LF 16	31		50		51	0	7	18	0.5	0.5	2	40	4.5×8×4.4
O LF 20	35		63		58	-0.2	9	22.5	0.5	0.5	2	45	5.5×9.5×5.4
O LF 25	42	0 -0.016	71		65		9	26.5	0.5	0.5	3	52	5.5×9.5×5.4
O LF 30	47		80	0	75		10	30	0.5	0.5	3	60	6.6×11×6.5
O LF 40	64	0	100	-0.3	100		14	36	1	0.5	4	82	9×14×8.6
O LF 50	80	0.019	125		124		16	46.5	1	1	4	102	11×17.5×11

Note) : indicates model numbers for which high temperature types are available (with metal retainer; service temperature: up to 100°C).

Ρ Ν

Accuracy symbol

(in mm)

(Example) LF30 A CL+700L H

——— High temperature symbol



Number of spline nuts Contamination protection Overall spline shaft length (*5) on one shaft accessory symbol (no symbol for one nut) (*1)

A3-80

in the rotational direction (*3)

Symbol for spline shaft (*4)

(*1) See A3-120. (*2) See A3-30. (*3) See A3-34. (*4) See A3-87. (*5) See A3-115.



Ball Spline

Medium Torque Type Ball Spline







Unit: mm

 Spline shaft diameter	Rows of balls	Basic toro	que rating	Basic loa	ad rating		Static permissible moment		Mass	
D₀ h7		C⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _{A.1} ** N-m	M _{A.2} ** N-m	Spline Nut g	Spline shaft kg/m	
6	4	0.98	1.96	1.18	2.16	4.9	36.3	35	0.23	
8	4	1.96	2.94	1.47	2.55	5.9	44.1	37	0.4	
10	4	3.92	7.84	2.84	4.9	15.7	98	90	0.62	
13	4	5.88	10.8	3.53	5.78	19.6	138	110	1.1	
16	6	31.4	34.3	7.06	12.6	67.6	393	230	1.6	
20	6	56.9	55.9	10.2	17.8	118	700	330	2.5	
25	6	105	103	15.2	25.8	210	1140	455	3.9	
30	6	171	148	20.5	34	290	1710	565	5.6	
40	6	419	377	37.8	60.5	687	3760	1460	9.9	
50	6	842	769	60.9	94.5	1340	7350	2760	15.5	

Note) **M_{A1} indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the

**Main indicates the permissible moment value in the axial direction when two spline nuts in close contact with each other are used, as shown in the figure above.
(Single LF-unit configuration is not stable in accuracy. We recommend using two units in close contact with each other.) For details on the maximum lengths of ball spline shafts by accuracy, please see **33-115**.

A3-81

Model LT-X

Model number coding

▲3-82 冗出比



LT4X





L (Full length with seals)

LT5X, LT5XL, LT6X LT6XL, LT8X, LT8XL

	Spline shaft diameter		Spline nut dimensions									
		Outer	diameter		Length				nsions	Greasing hole		
Model No.	D₀ h7	D	Tolerance	L (With seals)	L ₁ (Without seals)	L2	b H8	t	lo	d₀		
LT 4X	4	8	0 -0.009	14.4	12	7.5	_	1	_	1		
LT 5X LT 5XL	5	10	0 -0.009	15 26	13.6 24.6	7.3 18.3	2	1.2	4.7	1		
LT 6X LT 6XL	6	12	0 -0.009	19 30	17.6 28.6	10.2 21.2	2	1.2	6	1		
LT 8X LT 8XL	8	15	0 -0.011	25 40	23.8 38.8	14.6 29.6	2.5	1.2	8	1		
LT 10X	10	19	0 -0.013	33	30.8	23.9	3	1.5	13	1.5		
LT 13X	13	23	0 -0.013	36	32.4	24	3	1.5	15	1.5		
LT 16X	16	28	0 -0.013	50	46.4	35.4	3.5	2	17.5	2		
LT 20X	20	32	0 -0.016	63	59	47.4	4	2.5	29	2		
LT 25X	25	40	0 -0.016	71	67	52.6	4	2.5	36	3		
LT 30X	30	45	0 -0.016	80	75.6	59.6	4	2.5	42	3		

2 LT20X UU CL +700L P K

Model No.	Symbol for c in the rotation
Number of spline nuts on one shaft (no symbol for one nut)	(*2) Contamination accessory syn

Symbol for clearance n the rotational direction ^{*2)} ntamination protection cessory symbol (*1)

Download data by searching for the corresponding model number on the Technical Support site.

Accuracy Symbol for spline shaft (*4) symbol (*3)

https://tech.thk.com

Overall spline shaft length (*5) (in mm)

(*1) See 🖾 3-120. (*2) See 🖾 3-30. (*3) See 🖾 3-34. (*4) See 🖾 3-87. (*5) See 🖾 3-115.

Ball Spline



LT10X to 30X

Unit: mm

Basic tore	que rating	Basic loa	ad rating	Static	permissible m	noment	Ma	Mass	
C⊤ N-m	C₀⊤ N-m	C kN	C₀ kN	M _{A1}	M _{A2} (With seal)	M _{A2} (Without seal)	Spline Nut g	Spline shaft kg/m	
0.49	0.82	0.42	0.7	0.84	6.2	5.0	2.2	0.1	
0.82 1.59	1.25 3.20	0.56 1.09	0.85 2.19	1.04 6.11	8.2 35.5	6.6 28.4	3.3 8	0.15	
1.73 2.81	2.77 5.54	0.98 1.60	1.58 3.15	2.85 10.6	19 59.8	15.2 47.8	6.6 13.3	0.21	
6.00 10.10	9.23 19.4	1.39 2.35	2.15 4.53	5.13 21.1	34.3 110.9	27.4 88.7	14.3 24.3	0.38	
9.41	17.3	2.94	5.40	21.5	114	104	30	0.59	
17.1	28.7	4.16	6.96	28.9	164	149	40	1.01	
42.9	68.6	8.40	13.4	77.4	419	381	81	1.52	
66.4	117	10.5	18.6	144	735	669	130	2.41	
125	207	15.9	26.2	230	1183	1077	235	3.71	
196	319	20.8	34.0	335	1714	1560	295	5.37	

Note) The mass of the spline nut does not include the seal. Please check the spline shaft strength tests (**M3-12**) before use.

Model LF-X

LF4X

Model number coding

A3-84



LF5X, LF5XL, LF6X, LF6XL, LF8X, LF8XL

	Spline shaft diameter							Splin	ie nut	dime	nsior	IS	
		Outer	r diameter		Length		Flange Outer Diameter					Greasing hole	
Model No.	D₀ h7	D	Tolerance	L (With seal)	L ₁ (Without seal)	L2	D1	н	F	E	E1	d₀	PCD
LF 4X	4	8	0 -0.009	14.4	12	7.5	20	2.5	—	5.95	4.74		15
LF 5X LF 5XL	5	10	0 -0.009	15 26	13.6 24.6	7.3 18.3	23	2.7	— 6.5	6.55	5.35	1	17
LF 6X LF 6XL	6	12	0 -0.009	19 30	17.6 28.6	10.2 21.2	25	2.7	2.4 7.9	7.1	6.4	1	19
LF 8X LF 8XL	8	15	0 -0.011	25 40	23.8 38.8	14.6 29.6	28	3.8	3.5 11	9	8.4	1.5	22
LF 10X	10	19	0 -0.013	33	30.8	23.9	39	6	5.95	10.55	9.45	1.5	29
LF 13X	13	23	0 -0.013	36	32.4	24	43	6	6	12	10.2	1.5	33
LF 16X	16	28	0 -0.013	50	46.4	35.4	48	6	11.7	13.3	11.5	2	38
LF 20X	20	32	0 -0.016	63	59	47.4	54	8	15.7	15.8	13.8	2	43
LF 25X	25	40	0 -0.016	71	67	52.6	62	8	18.3	17.2	15.2	3	51
LF 30X	30	45	0 -0.016	80	75.6	59.6	74	10	19.8	20.2	18	3	60

LF20X UU CL +700L P K 2

Model No.

Symbol for clearance

Accuracy Symbol for spline shaft (*4) in the rotational direction symbol (*3)

https://tech.thk.com

Number of spline nuts

(*2)

Download data by searching for the corresponding model number on the Technical Support site.

on one shaft Contamination protection (no symbol for one nut) accessory symbol (*1) (in mm)

(*1) See A3-120. (*2) See A3-30. (*3) See A3-34. (*4) See A3-87. (*5) See A3-115.





øD

Unit: mm

LF10X to 30X

Basic torque rating Basic load rating Static permissible moment Mass Mounting hole M_{A1} M_{A2} M_{A2} (With seal) (Without seal) Spline Nut Spline shaft Ст Сот C $d_1 \times d_2 \times h$ kŇ N-m N-m kN kg/m g ŧ⊞⊧ ŧ<u>it nci</u>i⊧ ≡{[±_H_]]≡ N-m N-m N-m 3.4 through 0.49 0.82 0.42 0.7 0.84 6.2 4.9 4.7 0.1 1.25 1.04 0.82 0.56 0.85 8.2 6.5 9.9 3.4 through 0.15 1.59 3.20 1.09 2.19 6.11 35.5 28.4 14.6 1.73 2.77 0.98 1.58 2.85 19 15.2 13.8 3.4 through 0.21 2.81 5.54 1.60 3.15 10.6 59.8 47.8 20.5 6.00 9.23 1.39 2.15 5.13 34.3 27.4 26.5 3.4 through 0.38 2.35 36.5 10.10 19.4 4.53 21.1 110.9 88.7 4.5×8×4.4 17.3 2.94 114 66 0.59 9.41 5.40 21.5 104 4.5×8×4.4 17.1 28.7 4.16 6.96 28.9 164 149 82 1.01 4.5×8×4.4 42.9 68.6 8.40 13.4 77.4 419 381 131 1.52 5.5×9.5×5.4 66.4 117 10.5 18.6 144 735 669 212 2.41 5.5×9.5×5.4 125 207 15.9 26.2 230 1183 1077 335 3.71 $6.6 \times 11 \times 6.5$ 196 319 20.8 34.0 335 1714 1560 489 5.37

Note) The mass of the spline nut does not include the seal.

Please check the spline shaft strength tests (3-12) before use.

Options⇒A3-120

PCE

A3-85

Model LT with Recommended Shaft End Shape



Unit: mm

Model No.	d₃	Tolerance	l2	Sм	l4	S×ℓ₅
LT 6	5	0	12	M5×0.8	7	M2.5×4
LT 8	6	-0.012	14	M6×1	8	M3×5
LT 10	8	0	18	M8×1	11	M4×6
LT 13	10	-0.015	23	M10×1.25	14	M5×8
LT 16	14	0	30	M14×1.5	18	M6×10
LT 20	16	-0.018	38	M16×1.5	22	M8×15
LT 25	22	0	50	M22×1.5	28	M10×18
LT 30	27	-0.021	60	M27×2	34	M14×25
LT 40	36	0	80	M36×3	45	M18×30
LT 50	45	-0.025	100	M45×4.5	58	M22×40

Ball Spline

Spline Shaft

Spline shafts are divided in shape into precision solid spline shaft, special spline shaft and hollow spline shaft (types K and N), as described on **M3-77**.

Since production of a spline shaft with a specific shape is performed at your request, provide a drawing of the desired shaft shape when asking an estimate or placing an order.

[Sectional Shape of the Spline Shaft]

Table2 shows the sectional shape of a spline shaft. If the spline shaft ends need to be cylindrical, the minor diameter (ϕ d) value should not be exceeded if possible.



Model LT13 or smaller Model LT16 or greater

Table2 Cross-Sectional Shape of the Spline S	Shaft for Models LT and LF
--	----------------------------

Nominal shaft diameter	4	5	6	8	10	13	16	20	25	30	40	50	60	80	100
Minor diameter ød	3.5	4.5	5	7	8.5	11.5	14.5	18.5	23	28	37.5	46.5	56.5	75.5	95
Major diameter ϕD_0 h7	4	5	6	8	10	13	16	20	25	30	40	50	60	80	100
Ball center-to-center diameter ødp	4.6	5.7	7	9.3	11.5	14.8	17.8	22.1	27.6	33.2	44.2	55.2	66.3	87.9	109.5
Mass(kg/m)	0.1	0.15	0.23	0.4	0.62	1.1	1.6	2.5	3.9	5.6	9.9	15.5	22.3	39.6	61.8

*The minor diameter ød must be a value at which no groove is left after machining.



Unit: mm

Unit: mm

Nominal shaft diameter	4	5	6	8	10	13	16	20	25	30
Minor diameter ø d	3.6	4.5	5.4	7	8.6	11.3	13.9	17.9	22.4	27
Major diameter ϕ D ₀	4	5	6	8	10	13	16	20	25	30
Ball center-to-center diameter ϕ dp	4.4	5.5	6.6	8.6	10.7	13.8	17.1	21.1	26.4	31.6
Mass (g/m)	100	150	210	380	590	1010	1520	2410	3710	5370

Table3 Cross-Sectional Shape of the Spline Shaft for Models LT-X and LF-X

[Hole Shape of the Standard Hollow Type Spline Shaft]

Table4 shows the hole shape of the standard hollow type spline shaft (types K and N). Use this table when a requirement such as piping, wiring, air-vent or weight reduction needs to be met.



Table4 Cross-Sectional Shape of the Standard Hollow Spline Shaft for Models LT and LF

Nominal shaft diameter		6	8	10	13	16	20	25	30	40	50	60	80	100
Major diameter <i>φ</i> D₀ h7		6	8	10	13	16	20	25	30	40	50	60	80	100
Ball center-to-center diameter ϕ dp		7	9.3	11.5	14.8	17.8	22.1	27.6	33.2	44.2	55.2	66.3	87.9	109.5
Tuno K	Hole diameter (\u03c6 d_4)	2.5	3	4	5	7	10	12	16	22	25	32	52.5	67.5
Турек	Type K Mass(kg/m)		0.35	0.52	0.95	1.3	1.8	3	4	6.9	11.6	16	22.6	33.7
Tupo N	Hole diameter (\u03c6 d4)	_	—	_	_	11	14	18	21	29	36	_	_	_
Type N	Mass(kg/m)	-	—	_	—	0.8	1.3	1.9	2.8	4.7	7.4	_	_	

Note) The standard hollow type Spline Shaft is divided into types K and N. Indicate "K" or "N" at the end of the model number to distinguish between them when placing an order.



Type K (Thick)

Table5 Cross-Sectional Shape of the Hollow Spline Shaft for Models LT-X and LF-X (K Type) Unit: mm

Nominal shaft diameter	4	5	6	8	10	13	16	20	25	30
Minor diameter ϕ d	—	—	—	—	8.6	11.3	13.9	17.9	22.4	27
Major diameter ϕ D ₀	_	—	—	—	10	13	16	20	25	30
Ball center-to-center diameter ϕ dp	—	—	—	—	10.7	13.8	17.1	21.1	26.4	31.6
Hole diameter ϕ d ₄	_	—	—	—	4	5	7	10	12	16
Mass (g/m)	—	—	—	—	490	850	1220	1790	2820	3780



Type N (Thin)

Table6 Cross-Sectional Shape of the Hollow Spline Shaft for Models LT-X and LF-X (N Type)

Unit: mm

Nominal shaft diameter	4	5	6	8	10	13	16	20	25	30
Minor diameter ϕ d	—	—	—	—	—	—	13.9	17.9	22.4	27
Major diameter ϕ D ₀	—	—	—	—	—	—	16	20	25	30
Ball center-to-center diameter ϕ dp	—	—	—	—	—	—	17.1	21.1	26.4	31.6
Hole diameter ϕ d ₄	_	—	—	—	—	—	11	14	18	21
Mass (g/m)	—	—	—	—	—	—	770	1190	1700	2630



Unit: mm



[Chamfering of the Spline Shaft Ends]

To facilitate the insertion of the spline shaft into a spline nut, the shaft ends are normally chamfered with the dimensions indicated below unless otherwise specified.

The ends are chamfered whether they are used, such as with stepped, tapped, or drilled ends, or not used, such as with cantilevered supports.



Nominal shaft diameter	4	5	6	8	10	13	16	20	25	30	32	40	50	60	80	100
Chamfer C	0.3	0.3	0.5	0.5	0.5	0.5	0.5	0.5	0.5	0.5	0.5	1.0	1.0	2.0	2.0	2.0

Nominal shaft diameter	4	5	6	8	10	13	16	20	25	30
Chamfer C	0.3	0.3	0.5	0.5	1.5	1.5	1.5	1.5	2.0	2.0

[Length of the Incomplete Area of a Special Spline Shaft]

If the middle area or the end of a spline shaft is to be thicker than the minor diameter (ϕ d), an imperfect spline area is required to secure a recess for grinding. Table9 shows the relationship between the length of the incomplete section (S) and the flange diameter (ϕ df).

(This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.)



Unit: mm

Unit[.] mm



Table9 Length of Imperfect Spline Area: S Miniature type Unit: mm

Flange diameter ødf	4	5	6	8	10
Nominal shaft diameter	4	5	0	0	10
4	23	25	27	31	—
5	—	24	26	29	33

Flange diameter ϕ df 16 20 25 30 40 50 60 80 100 120 140 160 6 8 10 13 Nominal shaft diameter 6 24 28 31 39 _ _ _ 8 25 29 35 41 --10 26 31 38 45 _ _ _ _ 13 33 39 46 56 ____ ____ _ _ _ ____ ____ _ _ _ _ _ 16 36 47 58 67 _ _ _ _ _ _ _ _ _ _ _ 20 37 50 60 76 _ _ _ 25 38 51 72 88 _ _ _ _ _ _ _ _ _ _ _ _ 30 40 62 80 95 40 42 63 107 81 _ _ _ _ _ _ _ _ _ _ _ _ 50 45 65 96 118 _ 60 50 87 114 134 _ _ _ _ _ _ _ 80 53 89 135 _ _ _ _ _ _ _ _ _ 115 _ _ 100 57 90 116 136 _ ____ _ _ _ _

*This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.

Compact Type Uni												nit: mm			
	Flange diameter ϕ df	4	5	6	8	10	13	16	20	25	30	35	40	50	60
	Nominal shaft diameter	4	5	0	0	10	13	10	20	25	30	35	40	50	00
	4X	23	25	27	31	-	_	_	_	_	_	_	_	_	—
	5X	—	24	26	29	33	—	_	—	—	_	—	_	—	—
	6X	—	_	24	28	31	39	_	—	—	_	—	_	—	_
	8X	—	-	-	25	29	35	41	—	—	_	—	—	—	—
	10X	—	_	-	_	26	40	48	56	_	_	—	_	—	—
	13X	—	—	—	—	—	33	41	51	61	_	—	—	—	—
	16X	—	-	-	-	-	—	36	47	58	67	—	_	—	-
	20X	—	_	—	—	-	—	—	37	50	60	67	76	—	—
	25X	—	_	—	_	—	—	_	—	38	51	59	72	88	—
	30X	_	_	_	_	_	_	_	_	_	40	50	62	80	95

Standard Type

Unit: mm

▲3-90 冗出比

Compact Type

I Init

Accessories

Ball Spline model LT is provided with a standard key as indicated in Table10.





Table10 Standard Key for Model LT

		Та	ble10 Sta	ndard Key for Mode	el LT			Unit: mm
Nominal shaft		Width b		Height h		Length <i>l</i>	R	С
diameter		Tolerance(p7)		Tolerance(h9)		Tolerance(h12)		C
LT 4	2		2		6	0 -0.120	1	0.3
LT 5	2.5	+0.016	2.5	0	8	0 0.150	1.25	0.5
LT 6 LT 8	2.5	+0.006	2.5	-0.025	10.5		1.25	
LT 10	3]	3	1 1	13	0 -0.180	1.5	
LT 13	3		3		15	0.180	1.5	
LT 16	3.5		3.5		17.5		1.75	
LT 20	4	+0.024	4	0	29	0 -0.210	2	0.5
LT 25	4	+0.012	4	-0.030	36	0	2	
LT 30	4		4		42	-0.250	2	
LT 40	6		6		52		3	
LT 50	8	+0.030 +0.015	7	0	58	0 0.300	4	
LT 60	12	+0.036	8	-0.036	67		6	
LT 80	16	+0.018	10		76]	8	0.8
LT 100	20	+0.043 +0.022	13	0 0.043	110	0 0.350	10	0.0

Rotary Ball Spline

With Geared Type Models LBG and LBGT



Fig.1 Structure of Rotary Ball Spline Model LBG

Point of Selection	⊠3-6
Point of Design	⊠3-117
Options	⊠3-120
Model No.	⊠3-122
Precautions on Use	⊠3-123
Accessories for Lubrication	⊠24-1
Mounting Procedure and Maintenance	₿3-30
Cross-sectional Characteristics of the Spline Shaft	A 3-17
Equivalent factor	▲3-27
Clearance in the Rotation Direction	⊠3-30
Accuracy Standards	⊠3-34
Maximum Manufacturing Length by Accuracy	⊠3-115

Structure and Features

With the Rotary Ball Spline, the spline shaft has three crests, and along both sides of each crest, two rows of balls (six rows in total) are arranged to hold the crest so that a reasonable preload is applied.

These models are unit types based on model LBR, but have gear teeth on the flange circumference and radial and thrust bearings on the spline nut, all compactly integrated.

The rows of balls are held in a special resin retainer so that they smoothly roll and circulate. With this design, balls will not fall even if the spline shaft is removed.

[No Angular Backlash]

The spline shaft has three crests positioned equidistantly at 120° and along both sides of each crest, two rows of balls (six rows in total) are arranged so as to hold the crest at a contact angle of 45° and provide a preload. As a result, backlash in the rotational direction is eliminated and the rigidity is increased.

[Compact Design]

The spline nut is compactly integrated with radial and thrust bearings, allowing compact design to be achieved.

[High Rigidity]

Since the contact angle is large and an appropriate preload is given, high rigidity against torque and moment is achieved.

Use of needle bearings in the support unit achieves a rigid nut support strong against a radial load.

[Optimal for Torque Transmission with Spline Nut Drive]

Since the support bearings allow a rigid nut support, these models are optimal for torque transmission with spline nut drive.

Types and Features

[Types of Spline Nuts]

Ball Spline with Gears Model LBG

These models are unit types based on model LBR, but have gear teeth on the flange circumference and radial and thrust bearings on the spline nut, all compactly integrated. It is optimal for a torque transmission mechanism with spline nut drive.



Ball Spline with Gears Model LBGT

These models are unit types based on model LBR, but have gear teeth on the flange circumference and radial and thrust bearings on the spline nut, all compactly integrated. It is optimal for a torque transmission mechanism with spline nut drive. With a thrust raceway

Specification Table⇒A3-98

[Types of Spline Shafts] For details,see **A3-55**.

▲3-94 17日比

Housing Inner-diameter Tolerance

Table1 shows housing inner-diameter tolerance for models LBG and LBGT.

Table1 Housing Inner-diameter Tolerance

Housing Inner-diameter	General conditions	H7
Tolerance	When clearance needs to be small	J6





		Spline nut dimensions												
Model No.	Model No. Spline nut outer diameter		L	ength	Oute	r diameter	\	Width						
	D	Tolerance	L	Tolerance	D4	Tolerance	Lı	Tolerance	н	E				
• LBG 20	30	0 -0.009	60		47	0 -0.011	20	0 0.16	12	24				
• LBG 25	40	0	70	0	60	0	23	0	14	28				
LBG 30	45	_0.011	80		65	-0.013	27	0.19	16	32				
• LBG 40	60	0	100		85		31		18	41				
• LBG 50	75	0.013	112		100	0 -0.015	32		20	46				
LBG 60	90	0	127	0	120		38	-0.25	22	52.5				
LBG 85	120	-0.015	155		150	0 -0.025	40		26	64.5				

Note) •: indicates model numbers for which felt seal types are available (see A3-120).

Model number coding

2 LBG50 UU CM +700L H K Т

Model No.

▲3-96 1元出版

Symbol for clearance in the rotational direction (*3)

(*2)

Number of spline nuts on one shaft (no symbol for one nut) (*1)

Contamination protection Overall spline shaft length (*5) accessory symbol

(in mm)

(*1) See A3-120. (*2) See A3-30. (*3) See A3-34. (*4) See A3-100. (*5) See A3-115.

Symbol for spline shaft (*4)



Accuracy symbol



I Init: mm

A3-97

Unit												
		Gear specifications* Basic torque rati			que rating	Basic loa	ad rating	Static permissible moment	Mass			
	Tip circle diameter D₅	Standard pitch diameter PCD	Module m	Number of teeth z	C⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _^ ** N-m	Spline nut unit kg	Spline shaft kg/m	
	56	52	2	26	90.2	213	9.4	20.1	103	0.61	1.8	
	70	65	2.5	26	176	381	14.9	28.7	171	1.4	2.7	
	75	70	2.5	28	312	657	22.5	41.4	295	2.1	3.8	
	96	90	3	30	696	1420	37.1	66.9	586	3	6.8	
	111	105	3	35	1290	2500	55.1	94.1	941	4.1	10.6	
	133	126	3.5	36	1870	3830	66.2	121	1300	6.3	15.6	
	168	160	4	40	4740	9550	119	213	3180	11.8	32	

Note) *The gear specifications in the table represent the dimensions with maximum module.

Special gear types such as helical gear and worm gear can also be manufactured at your request. **M_A indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above. For details on the maximum lengths of ball spline shafts by accuracy, please see **⊠3-115**.





		Spline nut dimensions												
Model No.		oline nut r diameter	Length		Oute	r diameter	,	Width	Thrust raceway width					
	D	Tolerance	L	Tolerance	D4	Tolerance	Lı	Tolerance	G	н	E			
LBGT 20	30	0 -0.009	60		47	0 0.011	20	0 -0.16	4	12	24			
LBGT 25	40	0	70	0	60	0	23	0	5	14	28			
LBGT 30	45		80		65	-0.013	27	_0.19	5	16	32			
LBGT 40	60	0	100		85		31		8	18	41			
LBGT 50	75	0.013	112		100	0 -0.015	32	0	10	20	46			
LBGT 60	90	0	127	0	120		38	-0.25	12	22	52.5			
LBGT 85	120	-0.015	155		150	0 -0.025	40	<u> </u>	16	26	64.5			

Note) ●: indicates model numbers for which felt seal types are available (see **△3-120**).

A 3-98



Download data by searching for the corresponding model number on the Technical Support site.



I Init: mm

17日代 ▲3-99

	One nin												
	Gear spec	ifications*		Basic tore	que rating	Basic loa	ad rating	Static permissible moment	Mass				
Tip circle diameter D₅	Standard pitch diameter PCD	Module m	Number of teeth z	C⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _A ** N-m	Spline nut unit kg	Spline shaft kg/m			
56	52	2	26	90.2	213	9.4	20.1	103	0.67	1.8			
70	65	2.5	26	176	381	14.9	28.7	171	1.5	2.7			
75	70	2.5	28	312	657	22.5	41.4	295	2.2	3.8			
96	90	3	30	696	1420	37.1	66.9	586	3.3	6.8			
111	105	3	35	1290	2500	55.1	94.1	941	4.8	10.6			
133	126	3.5	36	1870	3830	66.2	121	1300	7.2	15.6			
168	160	4	40	4740	9550	119	213	3180	13.4	32			

Note) *The gear specifications in the table represent the dimensions with maximum module. Special gear types such as helical gear and worm gear can also be manufactured at your request. **Ma indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above.

For details on the maximum lengths of ball spline shafts by accuracy, please see 3-115.

Spline Shaft

Spline shafts are divided in shape into precision solid spline shaft, special spline shaft and hollow spline shaft (type K), as described on **A3-55**.

Since production of a spline shaft with a specific shape is performed at your request, provide a drawing of the desired shaft shape when asking an estimate or placing an order.

[Sectional Shape of the Spline Shaft]

Table2 shows the sectional shape of a spline shaft. If the spline shaft ends need to be cylindrical, the minor diameter (ϕ d) value should not be exceeded if possible.



Nominal shaft diameter	20	25	30	40	50	60	85
Minor diameter ¢ d	15.3	19.5	22.5	31	39	46.5	67
Major diameter ϕD_0	19.7	24.5	29.6	39.8	49.5	60	84
Ball center-to- center diameter	20	25	30	40	50	60	85
Mass (kg/m)	1.8	2.7	3.8	6.8	10.6	15.6	32

Table2 Sectional Shape of the Spline Shaft

*The minor diameter ϕ d must be a value at which no groove is left after machining.

[Hole Shape of the Standard Hollow Type Spline Shaft]

Table3 shows the hole shape of the standard hollow type spline shaft (type K) for models LBG and LBGT.

Use this table when a requirement such as piping, wiring, air-vent or weight reduction needs to be met.



Table3 Sectional Shape of the Standard Hollow Type Spline Shaft											
Nominal shaft diameter	20	25	30	40	50	60	85				
Minor diameter ød	15.3	19.5	22.5	31	39	46.5	67				
Major diameter øD₀	19.7	24.5	29.6	39.8	49.5	60	84				
Ball center-to-center diameter ø dp	20	25	30	40	50	60	85				
Hole diameter ød4	6	8	12	18	24	30	45				
Mass (kg/m)	1.6	2.3	2.9	4.9	7	10	19.5				

*The minor diameter ϕ d must be a value at which no groove is left after machining.

A3-100

Unit: mm

Rotary Ball Spline

[Chamfering of the Spline Shaft Ends]

To facilitate the insertion of the spline shaft into a spline nut, the shaft ends are normally chamfered with the dimensions indicated below unless otherwise specified.

Chamfer A

If the spline shaft ends are stepped, tapped, or drilled as in Fig. 2, they are machined with the Chamfer A dimensions indicated in Table 4.



Ball Spline

I Init: mm

• Chamfer B

If either end of the spline shaft is not used, such as for cantilever support, it is machined with the Chamfer B dimensions indicated in Table 4.



Fig. 3 Chamfer B

Nominal shaft diameter	20	25	30	40	50	60	85				
Chamfer A	1	1.5	2.5	3	3.5	5	7				
Chamfer B	4.5	5.5	7	8.5	10	13	16				

Table 4 Chamfer Dimensions of Spline Shaft Ends

Note) Spline shafts with nominal diameters 6, 8, and 10 are chamfered to C0.5.

[Length of Imperfect Area of a Special Spline Shaft]

If the middle area or the end of a spline shaft is to be thicker than the minor diameter (ϕ d), an imperfect spline area is required to secure a recess for grinding. Table5 shows the relationship between the length of the incomplete section (S) and the flange diameter (ϕ df).

(This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.)



Table5 Length of Imperfect Spline Area: S

Unit: mm

Flange diameter ∉df Nominal shaft diameter	20	25	30	35	40	50	60	80	100	120	140
20	35	43	51	57	62	—	—	—	—	_	—
25	_	51	64	74	82	97	—	_	—	—	_
30	—	—	54	67	76	92	105	—	—	—	_
40	_	_	_	—	59	80	95	119	—	—	_
50	—	—	—	—	—	63	83	110	131	—	—
60	—	—	_	—	—	_	66	100	123	140	_
70	—	_	_	—	_	_	—	89	115	134	150
85	—	_	_	—	_	—	—	61	98	122	140

▲3-102 元光长



With Support Bearing Type Models LTR and LTR-A



Fig.1 Structure of Rotary Ball Spline Model LTR

Point of Selection	A3-6
Point of Design	⊠3-117
Options	⊠3-120
Model No.	⊠3-122
Precautions on Use	⊠3-123
Accessories for Lubrication	⊠24-1
Mounting Procedure and Maintenance	₿3-30
Cross-sectional Characteristics of the Spline Shaft	⊠3-17
Equivalent factor	A3-27
Clearance in the Rotation Direction	⊠3-30
Accuracy Standards	A3-34
Maximum Manufacturing Length by Accuracy	⊠3-115



Ball Spline

Structure and Features

With the Rotary Ball Spline model LTR, the spline shaft has three crests on the circumference, and along both sides of each crest, two rows of balls (six rows in total) are arranged to hold the crest so that a reasonable preload is applied.

Angular-contact ball raceways are machined on the outer surface of the spline nut to constitute support bearings, allowing the whole body to be compactly and lightly designed.

The rows of balls are held in a special resin retainer so that they smoothly roll and circulate. With this design, balls will not fall even if the spline shaft is removed.

In addition, a dedicated seal for preventing foreign material from entering the support bearings is available.

[No Angular Backlash]

Two rows of balls facing one another hold a crest, formed on the circumference of the spline nut, at a contact angle of 20° to provide a preload in an angular-contact structure. This eliminates an angular backlash in the rotational direction and increases the rigidity.

[Compact Design]

The spline nut is integrated with the support bearings, allowing highly accurate, compact design to be achieved.

[Easy Installation]

This ball spline can easily be installed by simply securing it to the housing using bolts.

[High Rigidity]

Since the contact angle is large and an appropriate preload is given, high rigidity against torque and moment is achieved.

The support bearing has a contact angle of 30° to secure high rigidity against a moment load, thus to achieve a rigid shaft support.

Model LTR-A, a compact type of LTR, has a contact angle of 45° .





Model LTR

Model LTR-A



Types and Features

[Types of Spline Nuts]

Ball Spline Model LTR

A compact unit type whose support bearings are directly integrated with the outer surface of the spline nut.



Ball Spline Model LTR-A

A compact type even smaller than LTR.

Specification Table⇒▲3-108



[Types of Spline Shafts] For details,see **A3-77**.

🗛 3-106 冗计比

Housing Inner-diameter Tolerance

For the housing inner-diameter tolerance for model LTR, class H7 is recommended.

[Important note concerning model LTR]



Note) Because of the divided outer ring, it is necessary to incorporate inner-diameter tolerance in the nut bracket (H7 is recommended) to prevent shifting of the outer ring on the side opposite the flange.







		Spline nut dimensions													
Model No.	-	Outer diameter		Flange diameter					Standard type	Туре К	Oil hole position				
	D	Toler- ance	L	D ₁	D₀ h7	D7	н	L ₁	Е	Е	F	E1	PCD	PCD1	S×ℓ
LTR8 A	32		25	44	24	16	3	10.5	6	8.5	4	3	38	19	M2.6×3
LTR10 A	36	-0.009 -0.025	33	48	28	21	3	10.5	9	11.5	4		42	23	M3×4
LTR16 A	48		50	64	36	31	6	21	10	10	10.5		56	30	M4×6
LTR20 A	56		63	72	43.5	35	6	21	12	12	10.5	—	64	36	M5×8
LTR25 A	66	-0.010 -0.029	71	86	52	42	7	25	13	13	12.5		75	44	M5×8
LTR32 A	78	L'	80	103	63	52	8	25	17	17	12.5	_	89	54	M6×10
LTR40 A	100	-0.012 -0.034	100	130	79.5	64	10	33	20	20	16.5	_	113	68	M6×10

Model number coding

A3-108

2 LTR32 K UU ZZ CL A +500L P K



(*2) See 🖾 3-120. (*3) See 🖾 3-120. (*4) See 🖾 3-30. (*5) See 🖾 3-34. (*6) See 🖾 3-112. (*7) See 🖾 3-115.

(*1) No Symbol: standard K: flange inversed


Rotary Ball Spline



Model LTR10A

	Spline shaft diameter		Basic toro	que rating	Basic load rating		Static permissible moment	Support bearing basic load rating		Mass	
d1	D₀ h7	Rows of balls	C⊤ N-m	С₀т N-m	C KN	C₀ KN	M _A ** N-m	C kN	C₀ kN	Spline Nut	Spline shaft kg/m
3.4	8	4	1.96	2.94	1.47	2.55	5.9	0.69	0.24	0.08	0.4
3.4	10	4	3.92	7.84	2.84	4.9	15.7	0.77	0.3	0.13	0.62
4.5	16	6	31.3	34.3	7.06	12.6	67.6	6.7	6.4	0.35	1.6
4.5	20	6	56.8	55.8	10.2	17.8	118	7.4	7.8	0.51	2.5
5.5	25	6	105	103	15.2	25.8	210	9.7	10.6	0.79	3.9
6.6	32	6	180	157	20.5	34	290	10.5	12.5	1.25	5.6
9	40	6	418	377	37.8	60.4	687	16.5	20.7	2.51	9.9

Note) **M_A indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure below.

For details on the maximum lengths of ball spline shafts by accuracy, please see 3-115.



Unit: mm

₩₩ ▲3-109

Model LTR



		Spline nut dimensions												
Model No.	Outer	⁻ diameter	Length	Flange diameter										
	D	Tolerance	L	D ₁	D₀ h7	н	Lı	E	PCD	PCD1	S×ℓ			
LTR 16	52		50	68	39.5	5	37	10	60	32	M5×8			
LTR 20	56	0	63	72	43.5	6	48	12	64	36	M5×8			
LTR 25	62	0.007	71	78	53	6	55	13	70	45	M6×8			
LTR 32	80		80	105	65.5	9	60	17	91	55	M6×10			
LTR 40	100	0	100	130	79.5	11	74	23	113	68	M6×10			
LTR 50	120	-0.008	125	156	99.5	12	97	25	136	85	M10×15			
LTR 60	134	0 -0.009	140	170	115	12	112	25	150	100	M10×15			

Model number coding

2 LTR50 K UU ZZ CM +1000L H K

 Model No. Flange orientation symbol(*1)
 Symbol for clearance in the rotational direction(*4)
 Accuracy symbol (*5)

 Number of spline nuts on one shaft (no symbol for one nut)
 Spline nut contamination protection accessory symbol(*2)
 Support bearings contamination protection accessory symbol(*3)
 Overall spline shaft length (*7) (in mm)

(*2) See 🖾 3-120. (*3) See 🖾 3-120. (*4) See 🖾 3-30. (*5) See 🖾 3-34. (*6) See 🖾 3-112. (*7) See 🖾 3-115.

(*1) No Symbol: standard K: flange inversed

▲3-110 万光火

Download data by searching for the corresponding model number on the Technical Support site.

Ball Spline



Unit: mm

			Spline shaft diameter		Basic toro	que rating	Basic load rating		Static permissible moment	Support bearing basic load rating		Mass	
	d₁	U ^{Note}	D₀ h7	Rows of balls	C⊤ N-m	С _{от} N-m	C kN	C₀ kN	M _A ** N-m	C kN	C₀ kN	Spline Nut kg	Spline shaft kg/m
	4.5	5	16	6	31.4	34.3	7.06	12.6	67.6	12.7	11.8	0.51	1.6
	4.5	7	20	6	56.9	55.9	10.2	17.8	118	16.3	15.5	0.7	2.5
	4.5	8	25	6	105	103	15.2	25.8	210	17.6	18	0.93	3.9
	6.6	10	32	6	180	157	20.5	34	290	20.1	24	1.8	5.6
	9	13	40	6	419	377	37.8	60.5	687	37.2	42.5	3.9	9.9
	11	13	50	6	842	769	60.9	94.5	1340	41.7	54.1	6.7	15.5
	11	13	60	6	1220	1040	73.5	111.7	1600	53.1	68.4	8.8	22.3

Note) **M_A indicates the permissible moment value in the axial direction when a single spline nut is used, as shown in the figure above. Dimension U represents the dimension from the head of the hexagonal-socket-head type bolt to the spline nut end. For details on the maximum lengths of ball spline shafts by accuracy, please see **⊠3-115**.

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Spline Shaft

Spline shafts are divided in shape into precision solid spline shaft, special spline shaft and hollow spline shaft (types K and N), as described on **Δ3-77**.

Since production of a spline shaft with a specific shape is performed at your request, provide a drawing of the desired shaft shape when asking an estimate or placing an order.

[Sectional Shape of the Spline Shaft]

Table1 shows the sectional shape of a spline shaft. If the spline shaft ends need to be cylindrical, the minor diameter (ϕ d) value should not be exceeded if possible.



Nominal shaft diameter	8	10	16	20	25	32	40	50	60
Minor diameter ød	7	8.5	14.5	18.5	23	30	37.5	46.5	56.5
Major diameter øD₀ h7	8	10	16	20	25	32	40	50	60
Ball center-to-center diameter	9.3	11.5	17.8	22.1	27.6	35.2	44.2	55.2	66.3
Mass (kg/m)	0.4	0.62	1.6	2.5	3.9	5.6	9.9	15.5	22.3

Table1 Sectional Shape of the Spline Shaft

Unit: mm

*The minor diameter ϕ d must be a value at which no groove is left after machining.

[Hole Shape of the Standard Hollow Type Spline Shaft]

Table2 shows the hole shape of the standard hollow type spline shaft (types K and N). Use this table when a requirement such as piping, wiring, air-vent or weight reduction needs to be met.



Type K (Thick) Type N (Thin)

Unit: mm

Nominal sl	haft diameter	8	10	16	20	25	32	40	50	60
Major diameter <i>ø</i> D₀ h7		8	10	16	20	25	32	40	50	60
Ball center-to-center diameter ødp		9.3	11.5	17.8	22.1	27.6	35.2	44.2	55.2	66.3
Turne K	Hole diameter ød₄	3	4	7	10	12	18	22	25	32
Туре К	Mass(kg/m)	0.35	0.52	1.3	1.8	3	4.3	6.9	11.6	16
Туре N –	Hole diameter ød₄	_	—	11	14	18	23	29	36	—
	Mass(kg/m)	—	_	0.8	1.3	1.9	3.1	4.7	7.4	—

Table2 Sectional Shape of the Standard Hollow Type Spline Shaft

Note) The standard hollow type Spline Shaft is divided into types K and N. Indicate "K" or "N" at the end of the model number to distinguish between them when placing an order.

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[Chamfering of the Spline Shaft Ends]

To facilitate the insertion of the spline shaft into a spline nut, the shaft ends are normally chamfered with the dimensions indicated below unless otherwise specified.

The ends are chamfered whether they are used, such as with stepped, tapped, or drilled ends, or not used, such as with cantilevered supports.



Table 3 Chamfer Dimensions of Model LTR-A and Model LTR Spline Shaft Ends

Nominal shaft diameter	8	10	16	20	25	32	40	50	60
Chamfer C	0.5	0.5	0.5	0.5	0.5	0.5	1.0	1.0	2.0

[Length of Incomplete Area of a Special Spline Shaft]

If the middle area or the end of a spline shaft is to be thicker than the minor diameter (ϕ d), an imperfect spline area is required to secure a recess for grinding. Table4 shows the relationship between the length of the incomplete section (S) and the flange diameter (ϕ df).

(This table does not apply to overall length of 1,500 mm or greater. Contact THK for details.)



Table4 Length of Incomplete Spline Area: S

Unit: mm

Flange diameter ødf Nominal shaft diameter	6	8	10	13	16	20	25	30	40	50	60	80	100	120	140	160
8	—	25	29	35	41	—	—	—	—	—	—	—	—	—	—	—
10	—	—	26	31	38	45	—	—	_	—	—	_	—	—	—	—
16	-	_	-	_	36	47	58	67	_	_	_	_	-	_	_	—
20	—	—	—	—	—	37	50	60	76	—	—	—	—	—	—	—
25	—	—	—	—	—	—	38	51	72	88	—	—	—	—	—	—
32	—	—	_	—	—	—	—	—	40	75	88	109	—	—	—	—
40	-	_	_	_	_	_	_	_	42	63	81	107	-	_	_	—
50	—	—	—	—	—	—	—	_	—	45	65	96	118	—	—	—
60	—	—	—	—	—	_	_	—	_	—	50	87	114	134	—	—



Ball Spline

Unit: mm

Permissible Rotational Speed for Rotary Ball Splines

For model LTR rotary ball splines, the speed is restricted by whichever is lower of the support bearing permissible rotational speed and the critical speed of the spline. When using the product, do not exceed the permissible rotational speed.

	Table5 Model LTR permissible rotational speed Unit:mir										
	Permissible Rotational Speed										
Model No.	Ball spline	Support	bearing								
	Calculated using shaft length	Grease Lubrication	Oil Lubrication								
LTR16		4000	5400								
LTR20		3600	4900								
LTR25		3200	4300								
LTR32	see 🖾 3-16 .	2400	3300								
LTR40		2000	2700								
LTR50		1600	2200								
LTR60		1400	2000								

Table6 Model LTR-A permissible rotational speed

Unit:min⁻¹

	Permissible Ro	otational Speed				
Model No.	Ball spline	Support bearing				
	Calculated using shaft length	Grease Lubrication	Oil Lubrication			
LTR8A		6900	9300			
LTR10A		5900	7900			
LTR16A		4000	5400			
LTR20A	see 🖾 3-16 .	3600	4900			
LTR25A		3200	4300			
LTR32A		2400	3300			
LTR40A		2000	2700			

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Dimensional Drawing, Dimensional Table

Maximum Manufacturing Length by Accuracy

Maximum Manufacturing Length by Accuracy

Table1, Table2, Table3 and Table4 show the maximum manufacturing lengths of ball spline shafts by accuracy.

Nominal shaft diameter		Accuracy	
Nominal Shall diameter	Normal grade (No symbol)	High accuracy grade (H)	Precision Grade (P)
25	2000	1500	1000
30	2000	1600	1250
40	2000	2000	1500
50	3000	2000	1500
60	4000	2000	2000
70	4000	2000	2000
80	4000	2000	2000
100	4000	3000	3000

Table1 Maximum Manufacturing Length of Models SLS, SLS-L and SLF

Table2 Maximum Manufacturing Length of Models LBS, LBST, LBF, LBR, LBH, LBG and LBGT by Accuracy Unit: mm

Nominal shaft diameter		Accuracy	
Nominal shall diameter	Normal grade (No symbol)	High accuracy grade (H)	Precision Grade (P)
6	200	150	100
8	600	200	150
10	600	400	300
15	1800	600	600
20	1800	700	700
25	3000	1400	1400
30	3000	1400	1400
40	3000	1400	1400
50	3000	1400	1400
60	3800	2500	2000
70	3800	2500	2000
85	3800	3000	3000
100	4000	3000	3000
120	3000	3000	3000
150	3000	3000	3000

Ball Spline

Unit: mm

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Table3 Models LT-X and LF-X Maximum Manufacturing Length by Precision	
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Nominal shaft diameter	Accuracy				
Nominal shart diameter	Normal grade (No symbol)	High accuracy grade (H)	Precision Grade (P)		
4	200	200	200		
5	250	200	200		
6	315	250	200		
8	500	400	315		
10	1000	630	500		
13	1000	800	630		
16	2000	1000	1000		
20	2000	1500	1000		
25	3000	1500	1000		
30	3000	1600	1250		

Table4 Maximum Manufacturing Length of Models LT, LF, LTR and LTR-A by Accuracy

Unit: mm

Unit: mm

Nominal shaft diameter	Accuracy				
Nominal Shan diameter	Normal grade (No symbol)	High accuracy grade (H)	Precision Grade (P)		
4	600	200	200		
5	600	315	200		
6	600	400	315		
8	1000	500	400		
10	1000	630	500		
13	1000	800	630		
16	2000	1000	1000		
20	2000	1500	1000		
25	3000	1500	1000		
30	3000	1600	1250		
40	3000	2000	1520		
50	3000	2000	1500		
60	4000	2000	2000		
80	4000	2000	2000		
100	4000	3000	3000		

The length in the table represents the overall shaft length.
 With standard hollow shaft type (K), the values in the table apply.
 With standard hollow shaft type (N), the available maximum length for both the normal grade and the high accuracy grade is up to the length defined for the precision grade in the table.

Ball Spline

Checking List for Spline Shaft End Shape

If desiring a ball spline type with its end specially machined, check the following items when placing an order.

The diagram below shows a basic configuration of the Ball Spline.



[Check Items]

- a. Type of the spline nut to be fit
- b. Number of spline nuts
- c. Clearance in the rotation direction
- d. Accuracy
- e. With/without a seal (for a single seal, check its orientation)
- f. Overall length (including all dimensions? Total value correct?)
- g. Effective spline length
- h. Hardened area (mark the location with symbol \precsim and indicate the purpose of hardening)
- i. Orientation of the flange (for flanged type)

- j. Spline shaft end shape (thicker than the minimum spline diameter?) (black, mill scale)
- Positional relationship between the spline nut and the spline shaft end shape (keyway of the spline nut, flange mounting hole)
- I. Indication of chamfering for each part
- m.Shape of chamfer on the spline shaft end (see $\blacksquare 3-70)$
- n. Intended purpose of the though hole in the spline shaft if any
- o. o'. Snap ring groove
- p. Maximum length
- q. Precedented or not



Housing Inner-diameter Tolerance

When fitting the spline nut to the housing, tight fitting is normally recommended. If the accuracy of the Ball Spline does not need to be very high, clearance fitting is also acceptable.

Table1 Housing Inner-diameter Tolerance			
Housing Inner-diam-	General conditions	H7	
	When clearance needs to be small	J6	

Note) For the housing inner-diameter tolerance of Rotary Ball Spline model LTR, H7 is recommended.

Positions of the Spline-nut Keyway and Mounting Holes

The keyways formed on the outer surface of straight nuts for Ball Spline models are positioned where balls under a load are placed as shown in Fig.1.

The flange-mounting holes of the flange types are positioned as shown in Fig.2.

When placing an order, indicate their positions in relation to the keyway or the like to be formed on the spline shaft.







Model LT8X or smaller Model LT10X or greater



Model SLS

Model LBS

Model LT-X



Model LT13 or smaller

Model LT16 or greater

Model LT

Fig.1 Positions of Keyways

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Point of Design

Positions of the Spline-nut Keyway and Mounting Holes



Model LF4X

Models LF5X to 8X

Model LF10X or greater

Model LF-X

Fig.2 Positions of Flange Mounting Holes



Options

Lubrication

To prevent foreign material from entering the spline nut and the lubricant from leaking, special synthetic resin seals with high wear resistance are available for the Ball Spline.

Spline nuts with seals (seal for both ends type UU, and seal for one end) contain high-quality lithium-soap group grease No. 2. However, if using them at high speed or with a long stroke, replenish grease of the same type through the greasing hole on the spline nut after running in.

Afterward, replenish grease of the same type as necessary according to the service conditions.

The greasing interval differs depending on the conditions. Normally, replenish the lubricant (or replace the product) roughly every 100 km of travel distance (six months to one year) as a rule of thumb.

For a Ball Spline model type without a seal, apply grease to the interior of the spline nut or to the raceways of the spline shaft.

Material and Surface Treatment

Depending on the service environment, the Ball Spline requires anticorrosive treatment or a different material. For details of anticorrosive treatment and material change, contact THK.

Contamination Protection

Entrance of dust or other foreign material into the spline nut will cause abnormal wear or shorten the service life. Therefore, it is necessary to prevent detrimental foreign material from entering the Ball Spline. When entrance of dust or other foreign material is a possibility, it is important to select effective seals and/or dust-control device that meets the environment conditions.

For the Ball Spline, a special synthetic rubber seal that is highly resistant to wear is available as a contamination protection accessory. If desiring a higher contamination protection effect, a felt seal is also available for some types. For details about the felt seal, contact THK.

In addition, THK produces round bellows. Contact us for details.

Symbol	Contamination protection accessory
No Symbol	Without seal
UU	Rubber seal attached on both ends of spline nut
U	Rubber seal attached on either end of spline nut
DD	Felt seal attached on both ends of spline nut
D	Felt seal attached on either end of spline nut
ZZ	Rubber seal attached on both ends of support bearings
Z	Rubber seal attached on either end of support bearings

Table1 Dust prevention accessory symbol

Specifications of the Bellows

Bellows are available as a contamination protection accessory. Use this specification sheet.



Specifications of the Bellows

Supported Ball Screw models:

Dimension	ns of the	Bellows					
Stroke:() mm	MAX:() r	nm	MIN:() mm	
Permissible o	uter diamete	er:(øOD) D	esired	inner diam	eter:(ø ID)
How It Is	Used						
Installation di	rection:(hori	izontal, vertio	cal, slant)		Speed:() mm/sec. ı	nin.
Motion:(recip	rocation, vib	ration)					
Condition	5						
Resistance to	oil and wate	er: (necessar	y, unnec	essary) Oil r	name ()
Chemical resi	stance: Nam	e () × ()%
Location: (ind	loor, outdoo	r)					
Remarks:							
Number of	f Units To	Be Manu	ufactur	ed:			

Model Number Coding

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Model number configurations differ depending on the model features. Refer to the corresponding sample model number configuration.

[Ball Spline]

• Models SLS, SLS-L, SLF, LBS, LBST, LBF, LBR, LBH, LT, LF, LT-X and LF-X



Precautions on Use

[Handling]

- (1) Please use at least two people to move any product weighing 20 kg or more, or use a dolly or another conveyance. Doing so may cause injury or damage.
- (2) Do not disassemble the parts. This will result in loss of functionality.
- (3) Tilting a spline nut or spline shaft may cause them to fall by their own weight.
- (4) Take care not to drop or strike the Ball Spline. Doing so may cause injury or damage. Giving an impact to it could also cause damage to its function even if the product looks intact.
- (5) When assembling, do not remove the spline nut from the spline shaft.
- (6) When handling the product, wear protective gloves, safety shoes, etc., as necessary to ensure safety.

[Precautions on Use]

- Prevent foreign material, such as cutting chips or coolant, from entering the product. Failure to do so may cause damage.
- (2) If the product is used in an environment where cutting chips, coolant, corrosive solvents, water, etc., may enter the product, use bellows, covers, etc., to prevent them from entering the product.
- (3) Do not use the product at temperature of 80°C or higher. Except for the heat-resistant models, exposure to higher temperatures may cause the resin/rubber parts to deform/be damaged.
- (4) If foreign material such as cutting chips adheres to the product, replenish the lubricant after cleaning the product.
- (5) Micro-strokes tend to obstruct oil film to form on the raceway in contact with the rolling element, and may lead to fretting corrosion. Take consideration using grease offering excellent fretting prevention. It is also recommended that a stroke movement corresponding to the length of the spline nut be made on a regular basis to make sure oil film is formed between the raceway and rolling element.
- (6) Do not use undue force when fitting parts (pin, key, etc.) to the product. This may generate permanent deformation on the raceway, leading to loss of functionality.
- (7) Skewing or misalignment of the spline shaft support and spline nut can shorten service life substantially. Inspect the components carefully and make sure they are mounted correctly.
- (8) The spline nut must contain all its internal rolling elements (balls) when mounted on the spline shaft. Using a spline nut with any balls removed may result in premature damage.
- (9) Please contact THK if any balls fall out of the spline nut; do not use the spline nut if any balls are missing.
- (10) To mount the spline nut on the spline shaft, first locate the alignment indicators on both components, then insert the shaft through the opening in the spline nut, without forcing it, and adjust the position until the indicators are aligned. Forcing the shaft could cause balls to fall out. When mounting a spline nut equipped with a seal or preload, first lubricate the outer surface of the spline shaft.
- (11) Manipulate the spline nut gently, using a jig, when inserting it into the housing, taking care not to strike the side plate, end cap, or seal.
- (12) If an attached component is insufficiently rigid or mounted incorrectly, the bearing load will be concentrated at one location and performance will decline significantly. Make sure the housing and base are sufficiently rigid, the anchoring bolts are strong enough, and the component is mounted correctly.
- (13) If desiring to have a flanged-type Ball Spline additionally machined, such as having a dowel pin hole, contact THK.

Ball Spline



[Lubrication]

- (1) Thoroughly wipe off anti-rust oil and feed lubricant before using the product.
- (2) Do not combine different lubricants. Mixing lubricants can cause adverse interaction between disparate additives or other ingredients.
- (3) If the product will be exposed to constant vibration or high or low temperatures, or used in a clean room, vacuum, or other special environment, apply a lubricant suitable for both the specifications and the environment.
- (4) To lubricate a product that has no grease nipple or oil hole, apply lubricant directly to the raceway surface and execute a few preliminary strokes to ensure that the interior is fully lubricated.
- (5) Bear in mind that the Ball Spline's slide resistance is affected by changes in the consistency of the lubricant, which varies according to the temperature.
- (6) The Ball Spline may encounter increased slide resistance following lubrication, due to the lubricant's agitation resistance. Make sure to put the unit through some preliminary motions to ensure that it is fully lubricated before starting up the machine.
- (7) Excess lubricant may spatter immediately after lubrication. If necessary, wipe off any spattered grease.
- (8) Because lubricant performance declines over time, lubrication must be monitored regularly and fresh lubricant applied when needed, depending on how frequently the machine is operated.
- (9) The appropriate lubrication schedule will depend on usage conditions and the surrounding environment. In general, the unit should be lubricated after every 100 kilometers of operation (every 3 to 6 months). The actual lubrication schedule and amount of lubricant used should be determined by the condition of the machinery.
- (10) With oil lubrication, the lubricant may not always be thoroughly disseminated inside the Ball Spline, depending on its mounting position. If the preferred lubrication method is oil lubrication, please consult THK in advance.

[Storage]

When storing the Ball Spline, enclose it in a package designated by THK and store it in a room in a horizontal orientation while avoiding high temperature, low temperature and high humidity. After the product has been in storage for an extended period of time, lubricant inside may have deteriorated, so add new lubricant before use.

[Disposal]

Dispose of the product properly as industrial waste.

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Ball Spline 示张 General Catalog



Ball Spline THK General Catalog

B Support Book

Features and Types	. B 3-4
Features of the Ball Spline	
Structure and Features	
Classification of Ball Splines	
Point of Selection	. <mark>B</mark> 3-8
Flowchart for Selecting a Ball Spline	. <mark>B</mark> 3-8
Steps for Selecting a Ball Spline	. 🖪 3-8
Selecting a Type	. 🖪 3-10
 Studying the Spline Shaft Strength 	. 🖪 3-14
Predicting the Service Life	. 🖪 3-19
 Example of Calculating the Service Life 	. 🖪 3-24
Mounting Procedure and Maintenance.	
Assembling the Ball Spline	
Mounting the Spline	. 83-30
Installing the Spline Nut	
Installation of the Spline Shaft	
Lubrication	. 🖪 3-33
Options	
Material and Surface Treatment	
Contamination Protection	. 🖪 3-34
Model No.	
Model Number Coding	B 3-35
Descentions on Use	D 0.00
Precautions on Use	. 🖬 3-36

Product Descriptions (Separate)

Classification of Ball Splines	A 3-4
Point of Selection	A 3-6
Flowchart for Selecting a Ball Spline	A3-6
Steps for Selecting a Ball Spline	A 3-6
Selecting a Type	A 3-8
Studying the Spline Shaft Strength	A 3-12
Predicting the Service Life	A3-21
Selecting a Preload	A 3-30
Clearance in the Rotation Direction	A 3-30
Preload and Rigidity	A 3-30
Conditions and Guidelines for Selecting of a Preload	A3-31
Determining the Accuracy	A 3-34
Accuracy Grades	A3-34
Accuracy Standards	

High Torque Caged Ball Spline

Nodels SLS, SLS-L and SLF	. A3-36
Structure and Features	. A 3-37
Types and Features	. A3-40
Housing Inner-diameter Tolerance	. A3-41

Dimensional Drawing, Dimensional Table

Model SLS	A 3-42
Model SLF	A3-44
Spline Shaft	A3-46
Accessories	A 3-48

High Torque Type Ball Spline Models LBS LBST LBE LBR and LBH M3-50

	J -JU
Structure and Features	A3-51
Applications	A3-52
Types and Features	A3-53
Housing Inner-diameter Tolerance	A3-55

Dimensional Drawing, Dimensional Table

A3-56
A 3-60
A 3-62
A 3-64
A 3-66
A 3-68
A 3-69
A 3-72

Medium Torque Type Ball Spline

	14
Structure and Features	75
Types and Features	76

N 2 7/

٠	Housing	Inner-diameter	Tolerance	A 3-77
---	---------	----------------	-----------	--------

Dimensional Drawing, Dimensional Table

Model LT	A 3-78
Model LF	A3-80
Model LT-X	A 3-82
Model LF-X	A 3-84
Model LT with Recommended Shaft End Shape	A3-86
Spline Shaft	A 3-87
Accessories	A 3-91

Rotary Ball Spline

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Vith	Geared	Туре	Models	LBG	and	LBGT	A 3-92
• St	tructure	and F	eatures				A3-93

- Types and Features A3-94

Dimensional Drawing, Dimensional Table

Model LBG	A 3-96
Model LBGT	A3-98
Spline Shaft	A3-100

Rotary Ball Spline

With Support Bearing Type Models LTR and LTR-A. A3-104

Dimensional Drawing, Dimensional Table

Model LTR-A Compact Type	A3-108
Model LTR	A3-110
Spline Shaft	A3-112
Permissible Rotational Speed for Rotary Ball Splines	A3-114
Maximum Manufacturing Length by Accuracy	A3-115

Point of Design 3-117
Checking List for Spline Shaft End Shape M3-117
Housing Inner-diameter Tolerance 3-118
Positions of the Spline-nut Keyway and Mounting Holes 3-118

Options	. A3-120
Lubrication	. A3-120
Material and Surface Treatment	. A3-120
Contamination Protection	. A3-120
Specifications of the Bellows	. A3-121
Model No.	A3-122

- Precautions on Use A3-123





Ball Spline

Features of the Ball Spline



Fig.1 Structure of Ball Spline Models LBS and LT

Structure and Features

The Ball Spline is an innovative linear motion system in which balls accommodated in the spline nut transmit torque while linearly moving on precision-ground raceways on the spline shaft.

Unlike the conventional structure, a single spline nut can provide a preload with THK's Ball Spline. As a result, the Ball Spline demonstrates high performance in environments subject to vibrations and impact loads, locations where a high level of positioning accuracy is required or areas where high-speed kinetic performance is required.

In addition, even when used as an alternative to a linear bushing, the Ball Spline achieves a rated load more than 10 times greater than the linear bushing with the same shaft diameter, allowing it to compactly be designed and used in locations where an overhung load or a moment load is applied. Thus, the Ball Spline provides a high degree of safety factor and long service life.



Features and Types Features of the Ball Spline

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Square Nut Type Model LBH

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Features and Types

Classification of Ball Splines



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Flowchart for Selecting a Ball Spline

Steps for Selecting a Ball Spline

The following is a flowchart to reference when selecting a Ball Spline.



Point of Selection

Flowchart for Selecting a Ball Spline



Selecting a Type

There are three types of the Ball Spline: high torque type, medium torque type and rotary type. You can choose a type according to the intended use. In addition, wide arrays of spline nut shapes are available for each type, enabling the user to choose a desired shape according to the mounting or service requirements.

	Classification	Туре	Shape	Shaft diameter
High torque Caged Ball type		Type SLS Type SLS-L		Nominal shaft diameter 25 to 100mm
		Type SLF		Nominal shaft diameter 25 to 100mm
		Type LBS Type LBST		Nominal shaft diameter 6 to 150mm
High torque type		Type LBF		Nominal shaft diameter 15 to 100mm
High tor		Type LBR		Nominal shaft diameter 15 to 100mm
		Type LBH		Nominal shaft diameter 15 to 50mm

*For the specification table for each model, please see "A Product Descriptions."



Specification Table*	Structure and features	Major application
⊠3-42	 Redesigning the shape of the conventional high torque type spline shaft to be more circular significantly improves its torsion and flexural rigidity. Models SLS/SLF adopt the caged-ball technology to enable the circulating motion of evenly spaced balls to be maintained and high-speed response to be achieved, the cycle time of the machine can be improved. Models SLS/SLF adopt the caged-ball technology, they eliminate collision and mutual technology. 	
⊠3-44	 tual friction between balls, and realize low noise, pleasant running sound and low particle generation. Models SLS/SLF adopt the caged-ball technology to substantially increase the grease retention, thus achieving long-term maintenance-free operation. Models SLS/SLF adopt the caged-ball technology and a new circulation method, thus achieving stable and smooth motion with small rolling fluctuation. 	 Column and arm of industrial robot Automatic loader Transfer machine Automatic conveyance system Tire molding machine Spindle of spot-welding machine Guide shaft of high-speed automatic coating machine
⊠3-56	 The spline shaft has three crests equidis- tantly formed at angles of 120°. On both 	 machine Riveting machine Wire winder Work head of electric discharge machine Spindle drive shaft of grinding machine Speed gears Precision indexing machine
⊠3-62	sides of each crest, two rows (six rows in total) of balls are arranged to hold the crest from both sides. The angular-contact design of the ball contact areas allows an appropri- ate preload to be evenly applied. Since the balls circulate inside the spline nut, the outer dimensions of the spline nut	
⊠3-64	 are compactly designed. Even under a large preload, smooth straight motion is achieved. Since the contact angle is large (45°) and the displacement is minimal, high rigidity is achieved. 	
⊠3-66	 No angular backlash occurs. Capable of transmitting a large torque. 	

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	Classification	Туре	Shape	Shaft diameter
		Type LT		Nominal shaft diameter 4 to 100mm
Medium torque type		Type LF		Nominal shaft diameter 6 to 50mm
Medium to		Type LT-X		Nominal shaft diameter 4 to 30mm
		Type LF-X		Nominal shaft diameter 4 to 30mm
Rotary type	Rotation	Type LBG Type LBGT		Nominal shaft diameter 20 to 85mm
	Rotation	Type LTR-A Type LTR		Nominal shaft diameter 8 to 60mm

*For the specifi cation table for each model, please see "A Product Descriptions."

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Selecting a Type

Specification Table*	Ne* Structure and features Major application	
⊠3-78	 The spline shaft has two to three crests. On both sides of each crest, two rows (four to six rows in total) of balls are arranged to hold the crest from both sides. This design allows 	
⊠3-80	 an appropriate preload to be evenly applied. The contact angle of 20° and an appropriate preload level eliminate angular backlash, providing high-torque moment rigidity. 	 Die-set shaft and similar applications requiring straight motion under a heavy load Loading system and similar applications requiring Book-binding ma-
⊠3-82	• The length and external diameter of the LT-X ball spline's outer cylinder are the same as those of an LM-series linear bushing, meaning the nut can be replaced with a linear bushing.	rotation to a given angle at a fixed position • Automatic gas-welding machine spindle and simi- lar applications requiring a whirl-stop on one shaft
⊠3-84	• The length and external diameter of the LF-X ball spline's nut are the same as those of the Model LMF linear bushing, meaning the nut can be replaced with a linear bushing.	
⊠3-96	 A unit type that has the same contact structure as model LBS. The flange circumference on the spline nut is machined to have gear teeth, and radial and thrust needle bearings are compactly combined on the circumference of the spline nut. 	Speed gears for high torque transmission
⊠3-108	 A lightweight and compact type based on model LT, but has a spline nut circumference machined to have angular-contact type ball raceways to accommodate support bearings. 	Z axis of scalar robotWire winder



Studying the Spline Shaft Strength

The spline shaft of the Ball Spline is a compound shaft capable of receiving a radial load and torque. When the load and torgue are large, the spline shaft strength must be taken into account.

[Spline Shaft Receiving a Bending Load]

When a bending load is applied to the spline shaft of a Ball Spline, obtain the spline shaft diameter using the equation (1) below.

- Μ : Maximum bending moment acting on the spline shaft (N-mm)
- : Permissible bending stress of the σ spline shaft (98N/mm²)
- 7 : Modulus section factor of the spline shaft (mm³) (see Table3 on ▲3-17, Table4 on ▲3-18, Table5 on ▲3-19 and Table6 on ▲3-20)

[Reference] Section Modulus (Solid Circle)

Z =	$\frac{\pi \cdot d^3}{32}$	
Ζ	: Section Modulus	(mm ³)
d	: Shaft outer diameter	(mm)

[Spline Shaft Receiving a Torsion Load]

When a torsion load is applied on the spline shaft of a Ball Spline, obtain the spline shaft diameter using the equation (2) below.

$$\mathbf{T} = \boldsymbol{\tau}_{a} \cdot \mathbf{Z}_{P} \quad \text{and} \quad \mathbf{Z}_{P} = \frac{\mathbf{T}}{\boldsymbol{\tau}_{a}} \quad \dots \dots \dots (2)$$

т : Maximum torsion moment (N-mm)

- : Permissible torsion stress of the τa spline shaft
- (49N/mm²)
- Z₀ : Polar modulus of section of the spline nut (mm^3) (see Table3 on A3-17, Table4 on A3-18, Table5 on A3-19 and Table6 on A3-20)

[Reference] Section Modulus (Solid Circle)

$$Z_{\rm P} = \frac{\pi \cdot d^3}{16}$$

- ZΡ : Section modulus (mm³)
- d : Shaft outer diameter (mm)







Point of Selection

Studying the Spline Shaft Strength

[When the Spline Shaft Simultaneously Receives a Bending Load and a Torsion Load]

When the spline shaft of a Ball Spline receives a bending load and a torsion load simultaneously, calculate two separate spline shaft diameters: one for the equivalent bending moment (M_e) and the other for the equivalent torsion moment (T_e). Then, use the greater value as the spline shaft diameter.

Equivalent bending moment

$$\mathbf{M}_{\circ} = \frac{\mathbf{M} + \sqrt{\mathbf{M}^2 + \mathbf{T}^2}}{2} = \frac{\mathbf{M}}{2} \left\{ \mathbf{1} + \sqrt{\mathbf{1} + \left(\frac{\mathbf{T}}{\mathbf{M}}\right)^2} \right\} \dots (3)$$

Equivalent torsion moment

 $T_e = \tau_a \cdot Z_p$

[Rigidity of the Spline Shaft]

The rigidity of the spline shaft is expressed as a torsion angle per meter of shaft length. Its value should be limited within $1^{\circ}/4$.

$$\theta = 57.3 \times \frac{\mathbf{T} \cdot \mathbf{L}}{\mathbf{G} \cdot \mathbf{I}_{P}} \quad \dots \dots \dots (5)$$

Rigidity of the shaft = $\frac{\text{Torsion angle}}{\text{Unit length}} = \frac{\theta \cdot \ell}{L} < \frac{1^{\circ}}{4}$

- θ : Torsion angle (°)
- L : Spline shaft length (mm)
- G : Transverse elastic modulus

- ℓ : Unit length (1000mm)
- I_{P} : Polar moment of inertia (mm⁴)

(see Table3 on A3-17, Table4 on A3-18, Table5 on A3-19 and Table6 on A3-20)



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[Deflection and Deflection Angle of the Spline Shaft]

The deflection and deflection angle of the Ball Spline shaft need to be calculated using equations that meet the relevant conditions. Table1 and Table2 represent these conditions and the corresponding equations.

Table3 on $\square 3-17$, Table4 on $\square 3-18$, Table5 on $\square 3-19$ and Table6 on $\square 3-20$ show the section modulus of the spline shaft (Z) and the second moment of area (I). Using the Z and I values from the tables, the strength and displacement (deflection) of a typical ball spline within each model type can be obtained.

Support method	Condition	Deflection equation	Deflection angle equation
Both ends free		$\delta_{\max} = \frac{P\ell^3}{48EI}$	$i_1 = 0$ $i_2 = \frac{P\ell^2}{16EI}$
Both ends fas- tened		$\delta_{\max} = \frac{P\ell^3}{192EI}$	$i_1 = 0$ $i_2 = 0$
Both ends free	Uniform load p <i>i</i> 2 <i>l</i>	$\delta_{\max} = \frac{5p\ell^4}{384EI}$	$i_2 = \frac{p\ell^3}{24EI}$
Both ends fas- tened	Uniform load p	$\delta_{\max} = \frac{p\ell^4}{384EI}$	<i>i</i> ₂ = 0

Ball Spline

Point of Selection

Studying the Spline Shaft Strength



Table2 Deflection and Deflection Angle Equations

 δ_{max} : Maximum deflection (mm)

- Mo: Moment (N-mm)
- ℓ: Span (mm)
- I: Geometrical moment of inertia (mm⁴)
- *i*₁: Deflection angle at loading point

- *i*₂: Deflection angle at supporting point
- P: Concentrated load (N)
- p: Uniform load (N/mm)
- E: Modulus of longitudinal elasticity 2.06×10^{5} (N/mm²)



[Dangerous Speed of the Spline Shaft]

When a Ball Spline shaft is used to transmit power while rotating, as the rotational speed of the shaft increases, the rotation cycle nears the natural frequency of the spline shaft. It may cause resonance and eventually result in inability to move. Therefore, the maximum rotational speed of the shaft must be limited to below the dangerous speed that does not cause resonance.

The dangerous speed of the spline shaft is obtained using the equation (6).

(0.8 is multiplied as a safety factor)

If the shaft's rotation cycle exceeds or nears the resonance point during operation, it is necessary to reconsider the spline shaft diameter.

• Dangerous Speed

$$\mathbf{N}_{\rm e} = \frac{60\lambda^2}{2\pi \cdot \ell_{\rm b}^2} \cdot \sqrt{\frac{\mathbf{E} \times \mathbf{10}^3 \cdot \mathbf{I}}{\gamma \cdot \mathbf{A}}} \times \mathbf{0.8} \quad \cdots (6)$$

 $\ell_{\text{b}} \hspace{0.5cm} : \text{Distance between two mounting surfaces} \hspace{0.5cm} (\text{mm})$

E : Young's modulus (2.06×10⁵ N/mm²) I : Minimum geometrical moment of

inertia of the shaft (mm⁴)

$$I = \frac{\pi}{64} d^4$$
 d:Minor diameter (mm)

(see Table10, Table11, Table12 and Table13 on 3-24)

 γ : Density (specific gravity) (7.85×10⁻⁶kg/mm³)

$$A = \frac{\pi}{4} d^2 \qquad d: Minor diameter \qquad (mm)$$

(see Table10, Table11, Table12 and Table13 on 3-24)

- A : Spline shaft cross-sectional area (mm²)
- λ : Factor according to the mounting method
 - (1) Fixed free $\lambda = 1.875$
 - (2) Supported supported λ =3.142
 - (3) Fixed supported $\lambda = 3.927$
 - (4) Fixed fixed λ =4.73











Fixed - supported



Fixed - fixed



Predicting the Service Life

Predicting the Service Life

[Static Safety Factor]

To calculate a load applied to the ball spline, you must first know the average load used to calculate the service life and the maximum load used to calculate the static safety factor.

In particular, if the system starts and stops frequently, or if impact loads are applied, a large moment load or torque caused by overhung loads may be applied to the system. When selecting a model number, make sure that the desired model is capable of handling the required maximum load (whether stationary or in motion). The reference values for the static safety factor are shown in the table below.

e	f _T · f _c · C _o
IS -	Pmax

s	: Static	safety	factor
---	----------	--------	--------

$C_{\scriptscriptstyle 0}$: Basic static load rating*	(N)
Pma	: Maximum applied load	(N

- f⊤ : Temperature factor (see Fig.1 on A3-23)
- fc : Contact factor (see Table8 on A3-23)

*The basic static load rating is a static load of a defined direction and size where the sum of the permanent deformation of the ball and that of the rolling groove at the contact area under maximum stress is 0.0001 times the ball diameter.

Table3 Reference Values of Static Safety Factor (fs)

Machine using the Ball Spline	Load conditions	Minimum reference values
	Without vibration or impact	3.0 to 6.0
General industrial machinery	Without vibration or impact	4.0 to 7.0
	With vibration or impact under combined loads	5.0 to 8.0

*The reference values for the static safety factor may vary depending on the load conditions as well as the environment. lubrication status, precision of the mounted surface, and/or rigidity.

[Nominal Life]

The service life of a Ball Spline varies from unit to unit even if they are manufactured through the same process and used in the same operating conditions. Therefore, the nominal life defined below is normally used as a guidepost for obtaining the service life of a Ball Spline.

Nominal life is the total travel distance that 90% of a group of identical ball splines independently operating under the same conditions can achieve without showing flaking (scale-like pieces on a metal surface).



[Calculating the Nominal Life]

The nominal life of a Ball Spline varies with types of loads applied during operation: torque load, radial load and moment load. The corresponding nominal life values are obtained using the equations (7) to (10) below. (The basic load ratings in these loading directions are indicated in the specification table for the corresponding model number.)

When a Torque Load is Applied

$$\mathbf{L} = \left(\frac{\mathbf{f}_{\tau} \cdot \mathbf{f}_{c}}{\mathbf{f}_{w}} \cdot \frac{\mathbf{C}_{\tau}}{\mathbf{T}_{c}}\right)^{3} \times 50 \quad \dots \dots (7)$$

When a Radial Load is Applied

$$\mathbf{L} = \left(\frac{\mathbf{f}_{\tau} \cdot \mathbf{f}_{c}}{\mathbf{f}_{w}} \cdot \frac{\mathbf{C}}{\mathbf{P}_{c}}\right)^{3} \times 50 \quad \dots \dots (8)$$

L	: Nominal life	(km)
C_{T}	: Basic dynamic torque rating	(Ň-m)
T_{C}	: Calculated torque applied	(N-m)
С	: Basic dynamic load rating	(N)
P_{C}	: Calculated radial load	(N)
f⊤	: Temperature factor	
	(see Fig.1	on B3-21)
\mathbf{f}_{c}	: Contact factor	
	(see Table4	on B3-21)

fw : Load factor (see Table5 on **B3-21**)



When a Torque Load and a Radial Load are Simultaneously Applied

When a torque load and a radial load are simultaneously applied, calculate the nominal life by obtaining the equivalent radial load using the equation (9) below.

$\mathbf{P}_{\mathbf{E}} = \mathbf{P}_{\mathbf{c}} + \frac{4 \cdot \mathbf{T}_{\mathbf{c}}}{i \cdot \mathbf{d} \mathbf{p}} \cdot$	×10 ³ cosα(9)		
P _E : Equivalent ra	adial load	(N)	
$\cos \alpha$: Contact angle <i>i</i> =Number of rows of balls under a load			
/ Type LBS α =45°	<i>i</i> =2 (LBS10 or smaller)	Type SLS α =40°	<i>i</i> =3 \
	<i>i</i> =3 (LBS15 or greater)		
Type LTα=70°	<i>i</i> =2 (LT13 or smaller)	Type LT-X α =65°	<i>i</i> =2
\	<i>i</i> =3 (LT16 or greater)		/

- dp : Ball center-to-center diameter (mm) (see Table10, Table11, Table12 and Table13 on 3-24)
- When a Moment Load is Applied to a Single Nut or Two Nuts in Close Contact with Each Other

Obtain the equivalent radial load using the equation (10) below.

$\mathbf{P}_{u} = \mathbf{K} \cdot \mathbf{M}$(10)

- P. : Equivalent radial load (N) (with a moment applied)
- Κ : Equivalent Factors (see Table14 on **3-27**, Table15 on **3-28**, Table16 and Table17 on **3-29**)
- : Applied moment (N-mm) Μ

However, M should be within the range of the static permissible moment.

When a Moment Load and a Radial Load are Simultaneously Applied

Calculated the nominal life from the sum of the radial load and the equivalent radial load.

Calculating the Service Life Time

When the nominal life (L) has been obtained in the equation above, if the stroke length and the number of reciprocations per minute are constant, the service life time is obtained using the equation (11) below.

-)
- : Service life time l h (h) (m)
- ls : Stroke length
- : Number of reciprocations per minute n₁

(min⁻¹)


Ball Spline

Point of Selection

Predicting the Service Life

■f_T: Temperature Factor

If the temperature of the environment surrounding the operating Ball Spline exceeds 100°C, take into account the adverse effect of the high temperature and multiply the basic load ratings by the temperature factor indicated in Fig.1.

In addition, the Ball Spline must be of a high temperature type.

Note) If the environment temperature exceeds 80°C, hightem-perature types of seal and retainer are required. ContactTHK for details.

1.0 0.9 0.8 0.7 0.6 0.6 0.5 100 150 CO Raceway temperature (°C) Fig.1 Temperature Factor (fr)

■fc: Contact Factor

When multiple spline nuts are used in close contact with each other, their linear motion is affected by moments and mounting accuracy, making it difficult to achieve uniform load distribution. In such applications, multiply the basic load rating (C) and (C₀) by the corresponding contact factor in Table4. Note) if uneven load distribution is expected in a large ma-

chine, take into account the respective contact factor indicated in Table4.

■f_w: Load Factor

In general, reciprocating machines tend to involve vibrations or impact during operation. It is extremely difficult to accurately determine vibrations generated during high-speed operation and impact during frequent start and stop. When loads applied on a Ball Spline cannot be measured, or when speed and impact have a significant influence, divide the basic load rating (C or C_0), by the corresponding load factor in the table of empirically obtained data on Table5.

Table4 Contact Factor (f_c)

Number of spline nuts in close contact with each other	Contact factor f.
2	0.81
3	0.72
4	0.66
5	0.61
Normal use	1

Table5 Load Factor (f_w)

Vibrations/ impact	Speed (V)	f _w
Faint	Very low V≦0.25m/s	1 to 1.2
Weak	Slow 0.25 <v≦1m s<="" td=""><td>1.2 to 1.5</td></v≦1m>	1.2 to 1.5
Medium	Medium 1 <v≦2m s<="" td=""><td>1.5 to 2</td></v≦2m>	1.5 to 2
Strong	High V>2m/s	2 to 3.5

[Calculating the Average Load]

When the load applied on the spline shaft fluctuates according to varying conditions, such as an industrial robot arm traveling forward while holding a workpiece and traveling backward with empty weight, and a machine tool handling various workpieces, this varying load condition must be taken into account in service life calculation.

The average load (P_m) is a constant load under which the service life of an operating Ball Spline with its spline nut receiving a fluctuation load in varying conditions is equivalent to the service life under this varying load condition.

The following is the basic equation.

$$\mathbf{P}_{m} = \sqrt[3]{\frac{1}{L} \cdot \sum_{n=1}^{n} (\mathbf{P}_{n}^{3} \cdot \mathbf{L}_{n})}$$

Fm Average Loau (N	Pm	: Average Load	(N)
--------------------	----	----------------	-----

- P_n : Varying load (N)
- L : Total travel distance (mm)
- L_n : Distance traveled under P_n (mm)

• When the Load Fluctuates Stepwise

$$\mathbf{P}_{m} = \sqrt[3]{\frac{1}{L} (\mathbf{P}_{1}^{3} \cdot \mathbf{L}_{1} + \mathbf{P}_{2}^{3} \cdot \mathbf{L}_{2} \cdots + \mathbf{P}_{n}^{3} \cdot \mathbf{L}_{n})} \cdots \cdots \cdots \cdots \cdots (12)$$

- P_m : Average Load (N)
- P_n : Varying load (N)
- L : Total travel distance (m)
- L_n : Distance traveled under load P_n (m)



Point of Selection

Predicting the Service Life





P_{min}	: Minimum load	(N)
P_{max}	: Maximum load	(N)







(a) $P_m \doteq 0.65P_{max} \dots (14)$



(b) P_m **≑0.75P**_{max} ······(15)





[Equivalent Factor]

Table14 on **A3-27**, Table15 on **A3-28**, Table16 and Table17 on **A3-29** show equivalent radial load factors calculated under a moment load.

Example of Calculating the Service Life

• Example of Calculation - 1

An industrial robot arm (horizontal) [Conditions] Mass applied to the arm end m=50kg Stroke ℓ_s =200mm Spline nut mounting span (estimate) L₁=150mm

Arm length at maximum stroke L_{max} =400mm L_2 =325mm L_3 =50mm



Shaft Strength Calculation

Calculate the bending moment (M) and the torsion moment (T) applied on the shaft.

M=m×9.8×L_{max} =196000N-mm

 $T=m \times 9.8 \times L_3 = 24500$ N-mm

Since the bending and torsion moments are applied simultaneously, obtain the corresponding bending moment (M_e) and torsion moment (T_e), and then determine the shaft diameter based on the greater value. From equations (3) and (4) on **3.15**,

$$M_{e} = \frac{M + \sqrt{M^{2} + T^{2}}}{2} \approx 196762.7N \cdot mm$$
$$T_{e} = \sqrt{M^{2} + T^{2}} \approx 197525.3N \cdot mm$$

 $M_{\text{e}} < T_{\text{e}}$

 $:: T_e = \tau_a \times Z_P$ Hence,

$$Z_{\rm P} = \frac{T_{\rm e}}{\tau_{\rm a}} \doteq 4031 {\rm mm}^3$$

Thus, judging from Table4 on **\square3-18**, the nominal shaft diameter that meets Z_p is at least 40 mm.

₿3-24 冗光比

Point of Selection

Example of Calculating the Service Life

Average Load Pm

Obtain an applied load value when the arm is extended to the maximum length (P_{max}), and another when the arm is contracted (P_{min}). Based on the values obtained, calculate the average load on the spline shaft nut.

$$P_{1max} = \frac{m \times 9.8 (L_1 + L_2)}{L_1} \doteq 1551.7N$$

$$P_{2max} = \frac{m \times 9.8 \times L_2}{L_1} \doteq 1061.7N$$

When the arm is contracted

1

$$P_{1\min} = \frac{m \times 9.8 \times [(L_2 - \ell_s) + L_1]}{L_1} \approx 898.3N$$

$$P_{2\min} = \frac{m \times 9.8 \times (L_2 - \ell_s)}{L_1} \approx 408.3N$$

As this load is monotonically varying as shown in the Fig.3 on **3-23**, calculate the average load using the equation (13) on **3-23**.

The average load (P_{1m}) on spline nut 1

$$\mathsf{P}_{1\mathsf{m}} \doteq \frac{1}{3}(\mathsf{P}_{1\mathsf{min}} + 2\mathsf{P}_{1\mathsf{max}}) = 1333.9\mathsf{N}$$

The average load (P2m) on spline nut 2

$$\mathsf{P}_{2\mathsf{m}} \doteq \frac{1}{3}(\mathsf{P}_{2\mathsf{min}} + 2\mathsf{P}_{2\mathsf{max}}) = 843.9\mathsf{N}$$

Obtain the torque applied on one spline nut.

$$T = \frac{m \times 9.8 \times L_3}{2} = 12250 \text{N} \cdot \text{mm}$$

Since the radial load and the torque are simultaneously applied, calculate the equivalent radial load using equation (9) on **E3-20**.

$$P_{1E} = P_{1m} + \frac{4 \times T}{3 \times dp \times \cos\alpha} = 1911.4N$$
$$P_{2E} = P_{2m} + \frac{4 \times T}{3 \times dp \times \cos\alpha} = 1421.4N$$

■Nominal Life L_n

Based on the nominal life equation (8) on **3-19**, each nominal life is obtained as follows.

Nominal life of the spline nut $L_1 = \left(\frac{f_T \times f_C}{f_W} \times \frac{C}{P_{1E}}\right)^3 \times 50 = 68867.4 \text{km}$

Nominal life of the spline nut L₂ =
$$\left(\frac{f_T \times f_c}{f_W} \times \frac{C}{P_{2E}}\right)^3 \times 50 = 167463.2 \text{km}$$



- f_T: Temperature factor = 1 (from Fig.1 on **■3-21**)
- f_c: Contact factor = 1 (from Table4 on **B3-21**)
- fw: Load factor = 1.5 (from Table5 on **B3-21**)
- C: Basic dynamic load rating = 31.9 kN (model LBS40)

Given the nominal life obtained for each spline nut above, the nominal life of the Ball Spline unit is equal to that of spline nut 1, which is 68867.4km.

• Example of Calculation - 2

[Conditions] Thrust position: F_s Stroke velocity: $V_{max} = 0.25m/sec$ Acceleration: $a=0.36m/sec^2$ (from the respective velocity diagram) Stroke: S=700mm Housing mass: $m_i=30kg$ Arm mass : $m_2=20kg$ Head mass: $m_3=15kg$ Work mass: $m_4=12kg$

Distance from the thrust position to each mass $l_1=200$ mm $l_2=500$ mm $l_3=1276$ mm

- Cycle (1 cycle: 30 sec)
- 1. Descent (3.5sec) 2.Dwell (1sec): with a work
- 3. Ascend (3.5sec) 4.Dwell (7sec)
- 5. Descent (3.5sec) 6.Dwell (1sec): without a work
- 7. Ascend (3.5sec) 8.Dwell (7sec)





B3-26



(The Ball Spline type is LBF in this example.)

Fig.6

Point of Selection

Example of Calculating the Service Life

Shaft Strength Calculation

Calculate the shaft strength while assuming the shaft diameter to be 60 mm. (with double spline nut in contact with each other)

■Calculating the Moment (M_n) Applying on the Spline Nut during Acceleration, Uniform Motion and Deceleration with Different Masses (m_n)

Applied moment during deceleration: M1

$$\mathbf{M}_{1} = \mathbf{m}_{n} \times \mathbf{9.8} \left(\mathbf{1} \pm \frac{\mathbf{a}}{\mathbf{g}} \right) \times \boldsymbol{\ell}_{n} \quad \cdots \cdots \cdots (\mathbf{a})$$

Applied moment during uniform motion: M2

 $M_2 = m_n \times 9.8 \times \ell_n$

·····(b)

Applied moment during deceleration: M₃

m_n: Mass

(kg)

- a : Acceleration (m/sec²)
- g : Gravitational acceleration (m/sec2)

 $\ell_{\rm n}$: Offset from each loading point to the trust center (mm) Assume:

$$\mathsf{A} = \left(1 + \frac{\mathsf{a}}{\mathsf{g}}\right), \quad \mathsf{B} = \left(1 - \frac{\mathsf{a}}{\mathsf{g}}\right)$$

- During descent From equation (c), during acceleration $M_{m1} = m_1 \times 9.8 \times B \times \ell_1 + m_2 \times 9.8 \times B \times (\ell_1 + \ell_2) + m_3 \times 9.8 \times B \times (\ell_1 + \ell_3)$ = 398105.01N - mmFrom equation (b), during uniform motion $M_{m2} = m_1 \times 9.8 \times \ell_1 + m_2 \times 9.8 \times (\ell_1 + \ell_2) + m_3 \times 9.8 \times (\ell_1 + \ell_3)$ = 412972N - mmFrom equation (a), during deceleration $M_{m3} = m_1 \times 9.8 \times A \times \ell_1 + m_2 \times 9.8 \times A \times (\ell_1 + \ell_2) + m_3 \times 9.8 \times A \times (\ell_1 + \ell_3)$ = 427838.99N - mm
- During ascent

From equation (a), during acceleration

 $\begin{array}{ll} M_{m1}' &= m_1 \times 9.8 \times A \times \ell_1 + m_2 \times 9.8 \times A \times (\ell_1 + \ell_2) + m_3 \times 9.8 \times A \times (\ell_1 + \ell_3) \\ &= 427838.99N \text{-mm} \\ \\ \text{From equation (b), during uniform motion} \end{array}$

 $M_{m2}' = m_1 \times 9.8 \times \ell_1 + m_2 \times 9.8 \times (\ell_1 + \ell_2) + m_3 \times (\ell_1 + \ell_3)$ =412972N-mm



```
From equation (c), during deceleration
   M_{m3}' =m_1 \times 9.8 \times B \times \ell_1 + m_2 \times 9.8 \times B \times (\ell_1 + \ell_2) + m_3 \times 9.8 \times B \times (\ell_1 + \ell_3)
           =398105.01N-mm
• During descent (with a work loaded)
   From equation (c), during acceleration
   M_{m1}" = M_{m1} + m_4 \times 9.8 \times B \times (\ell_1 + \ell_3)
           =565433.83N-mm
   From equation (b), during uniform motion
   M_{m2}" = M_{m2}+ m_4 \times 9.8 \times (\ell_1 + \ell_3)
           =586549.6N-mm
   From equation (a), during deceleration
   M_{m3}" = M_{m3}+m_4 \times 9.8 \times A \times (\ell_1 + \ell_3)
           =607665.37N-mm
• During ascent (with a work loaded)
   From equation (a), during acceleration
   M_{m1}" = M_{m1}'+ m_4 \times 9.8 \times A \times (\ell_1 + \ell_3)
           =607665.37N-mm
   From equation (b), during uniform motion
   M_{m2} = M_{m2} + m_4 \times 9.8 \times (\ell_1 + \ell_3)
           =586549.6N-mm
   From equation (c), during deceleration
   M_{m3}''' = M_{m3}'+m_4 \times 9.8 \times B \times (\ell_1 + \ell_3)
           =565433.83N-mm
M_1 = M_{m_1} = M_{m_3} = 398105.01 \text{N-mm}
   M<sub>2</sub>=M<sub>m2</sub>=M<sub>m2</sub>'=412972N-mm
   M<sub>3</sub>=M<sub>m3</sub>=M<sub>m1</sub>'=427838.99N-mm
   M<sub>1</sub>'=M<sub>m1</sub>''=M<sub>m3</sub>'''=565433.83N-mm
   M<sub>2</sub>'=M<sub>m2</sub>''=M<sub>m2</sub>'''=586549.6N-mm
   M<sub>3</sub>'=M<sub>m3</sub>"=M<sub>m1</sub>"=607665.37N-mm
```

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■3-28 17出版

Ball Spline

Point of Selection

Example of Calculating the Service Life

■Calculating the Equivalent Radial Load Considered to be Applied to the Spline Nut with Different Moments Relational expression between moment Mn and Pn

 $\mathbf{P}_{n} = \mathbf{M}_{n} \times \mathbf{K} \qquad \cdots \cdots \cdots (d)$

P_n : Equivalent radial load (N)

M_n : Applied moment (N-mm)

K : Equivalent factor

(from Table15 to **\blacksquare3-28**) (If two spline nuts of LBF60 contact with each other, K = 0.013) Calculate the equivalent radial load with different applied moments using equation (d).

 $\begin{array}{l} P_{m1} = P_{m3}' = M_1 \times 0.013 \rightleftharpoons 5175.4N \\ P_{m2} = P_{m2}' = M_2 \times 0.013 \rightleftharpoons 5368.6N \\ P_{m3} = P_{m1}' = M_3 \times 0.013 \rightleftharpoons 5561.9N \\ P_{m1}'' = P_{m3}''' = M_1' \times 0.013 \rightleftharpoons 7350.7N \\ P_{m2}'' = P_{m2}''' = M_2' \times 0.013 \rightleftharpoons 7625.2N \\ P_{m3}'' = P_{m1}''' = M_3' \times 0.013 \rightleftharpoons 7899.7N \end{array}$



 $P_1 = P_{m1} = P_{m3}' \rightleftharpoons 5175.4N$ $P_2 = P_{m2} = P_{m2}' \doteqdot 5368.6N$ $P_3 = P_{m3} = P_{m1}' \doteqdot 5561.9N$ $\begin{cases} P_4 = P_{m1}" = P_{m3}"" \rightleftharpoons 7350.7N \\ P_5 = P_{m2}" = P_{m2}"" \doteqdot 7625.2N \\ P_6 = P_{m3}" = P_{m1}"" \doteqdot 7899.7N \end{cases}$

$$\begin{split} S &= S_a = S_b = S_c = S_d = 700 mm \\ S_1 &= S_1 = S_1' = S_1'' = S_1''' = 87.5 mm \\ S_2 &= S_2 = S_2' = S_2'' = S_2''' = 525 mm \\ S_3 &= S_3 = S_3' = S_3'' = S_3''' = 87.5 mm \end{split}$$

■Calculating the Average Load P_m

Using equation (12) on **B3-22**,

$$P_{m} = \sqrt[3]{\frac{1}{4 \times S}} \left[2 \left\{ (P_{1}^{3} \times S_{1}) + (P_{2}^{3} \times S_{2}) + (P_{3}^{3} \times S_{3}) \right\} + 2 \left\{ (P_{4}^{3} \times S_{3}) + (P_{5}^{3} \times S_{2}) + (P_{6}^{3} \times S_{1}) \right\} \right]$$

$$= 6689.5N$$

■Calculating the Rated Life L from the Average Load Using equation (8) on ■3-19.

$$L = \left(\frac{f_{T} \cdot f_{c}}{f_{W}} \cdot \frac{C}{P_{m}}\right)^{3} \times 50$$
$$= 7630 \text{ km}$$

f_⊤ : Temperature factor = 1 (from Fig.1 on **□3-21**) f_c : Contact factor=0.81 (from Table4 on **□3-21**) f_w : Load factor=1.5 (from Table5 on **□3-21**) C : Basic dynamic load rating = 66.2 kN (model LBF60)

Given the result above, the nominal life of model LBF60 with double spline nuts used in close contact with each other is 7,630 km.



Ball Spline

Assembling the Ball Spline

Mounting the Spline

Fig.1 and Fig.2 shows examples of mounting the spline nut. Although the Ball Spline does not require a large strength for securing it in the spline shaft direction, do not support the spline only with driving fitting.

Note) On both ends of the spline nut of Caged Ball Ball Spline model SLS, resin end caps are installed. Hitting them or pressing hard may cause damage. You must take care not to apply an excessive load.

Straight nut type







Fig.1 Examples of Fitting the Spline Nut



Mounting Procedure and Maintenance

Assembling the Ball Spline

Model LBH



Flanged type





Model LTR





Model LBG



Fig.2 Examples of Fitting the Spline Nut



Installing the Spline Nut

When installing the spline nut into the housing, do not hit the side plate or the seal, but gently insert it using a jig (Fig.3).



	Table1 Dimensions of the Jig for Model LBS										Unit: mm	
Nominal shaft diameter	15	20	25	30	40	50	60	70	85	100	120	150
di	12.5	16.1	20.3	24.4	32.4	40.1	47.8	55.9	69.3	83.8	103.8	131.8

Table2 Dimensions of the Jig for Model LT

Unit: mm

									Onit. mini				
Nominal shaft diameter	6	8	10	13	16	20	25	30	40	50	60	80	100
di	5.0	7	8.5	11.5	14.5	18.5	23	28	37.5	46.5	56	75.5	94.5

Installation of the Spline Shaft

When installing the spline shaft into the spline nut, identify the matching marks (Fig.4) on the spline shaft and the spline nut, and then insert the shaft straightforward while checking their relative positions.

Note that forcibly inserting the shaft may cause balls to fall off.

If the spline nut is attached with a seal or given a preload, apply a lubricant to the outer surface of the spline shaft.

JUHK

B3-32



Mounting Procedure and Maintenance

Lubrication

To prevent foreign material from entering the spline nut and the lubricant from leaking, special synthetic resin seals with high wear resistance are available for the Ball Spline.

Spline nuts with seals (seal for both ends type UU, and seal for one end) contain high-quality lithium-soap group grease No. 2. However, if using them at high speed or with a long stroke, replenish grease of the same type through the greasing hole on the spline nut after running in.

Afterward, replenish grease of the same type as necessary according to the service conditions.

The greasing interval differs depending on the conditions. Normally, replenish the lubricant (or replace the product) roughly every 100 km of travel distance (six months to one year) as a rule of thumb.

For a Ball Spline model type without a seal, apply grease to the interior of the spline nut or to the raceways of the spline shaft.



Material and Surface Treatment

Depending on the service environment, the Ball Spline requires anticorrosive treatment or a different material. For details of anticorrosive treatment and material change, contact THK.

Contamination Protection

Entrance of dust or other foreign material into the spline nut will cause abnormal wear or shorten the service life. Therefore, it is necessary to prevent detrimental foreign material from entering the Ball Spline. When entrance of dust or other foreign material is a possibility, it is important to select effective seals and/or dust-control device that meets the environment conditions.

For the Ball Spline, a special synthetic rubber seal that is highly resistant to wear is available as a contamination protection accessory. If desiring a higher contamination protection effect, a felt seal is also available for some types. For details about the felt seal, contact THK.

In addition, THK produces round bellows. Contact us for details.

Symbol	Contamination protection accessory
No Symbol	Without seal
UU	Rubber seal attached on both ends of spline nut
U	Rubber seal attached on either end of spline nut
DD	Felt seal attached on both ends of spline nut
D	Felt seal attached on either end of spline nut
ZZ	Rubber seal attached on both ends of support bearings
Z	Rubber seal attached on either end of support bearings

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Table1 Dust prevention accessory symbol

Model Number Coding

Model number configurations differ depending on the model features. Refer to the corresponding sample model number configuration.

[Ball Spline]

Models SLS, SLS-L, SLF, LBS, LBST, LBF, LBR, LBH, LT, LF, LT-X and LF-X



(*1) No Symbol: standard K: flange inversed



Precautions on Use

Ball Spline

[Handling]

- (1) Please use at least two people to move any product weighing 20 kg or more, or use a dolly or another conveyance. Doing so may cause injury or damage.
- (2) Do not disassemble the parts. This will result in loss of functionality.
- (3) Tilting a spline nut or spline shaft may cause them to fall by their own weight.
- (4) Take care not to drop or strike the Ball Spline. Doing so may cause injury or damage. Giving an impact to it could also cause damage to its function even if the product looks intact.
- (5) When assembling, do not remove the spline nut from the spline shaft.
- (6) When handling the product, wear protective gloves, safety shoes, etc., as necessary to ensure safety.

[Precautions on Use]

- (1) Prevent foreign material, such as cutting chips or coolant, from entering the product. Failure to do so may cause damage.
- (2) If the product is used in an environment where cutting chips, coolant, corrosive solvents, water, etc., may enter the product, use bellows, covers, etc., to prevent them from entering the product.
- (3) Do not use the product at temperature of 80°C or higher. Except for the heat-resistant models, exposure to higher temperatures may cause the resin/rubber parts to deform/be damaged.
- (4) If foreign material such as cutting chips adheres to the product, replenish the lubricant after cleaning the product.
- (5) Micro-strokes tend to obstruct oil film to form on the raceway in contact with the rolling element, and may lead to fretting corrosion. Take consideration using grease offering excellent fretting prevention. It is also recommended that a stroke movement corresponding to the length of the spline nut be made on a regular basis to make sure oil film is formed between the raceway and rolling element.
- (6) Do not use undue force when fitting parts (pin, key, etc.) to the product. This may generate permanent deformation on the raceway, leading to loss of functionality.
- (7) Skewing or misalignment of the spline shaft support and spline nut can shorten service life substantially. Inspect the components carefully and make sure they are mounted correctly.
- (8) The spline nut must contain all its internal rolling elements (balls) when mounted on the spline shaft. Using a spline nut with any balls removed may result in premature damage.
- (9) Please contact THK if any balls fall out of the spline nut; do not use the spline nut if any balls are missing.
- (10) To mount the spline nut on the spline shaft, first locate the alignment indicators on both components, then insert the shaft through the opening in the spline nut, without forcing it, and adjust the position until the indicators are aligned. Forcing the shaft could cause balls to fall out. When mounting a spline nut equipped with a seal or preload, first lubricate the outer surface of the spline shaft.
- (11) Manipulate the spline nut gently, using a jig, when inserting it into the housing, taking care not to strike the side plate, end cap, or seal.
- (12) If an attached component is insufficiently rigid or mounted incorrectly, the bearing load will be concentrated at one location and performance will decline significantly. Make sure the housing and base are sufficiently rigid, the anchoring bolts are strong enough, and the component is mounted correctly.
- (13) If desiring to have a flanged-type Ball Spline additionally machined, such as having a dowel pin hole, contact THK.



[Lubrication]

- (1) Thoroughly wipe off anti-rust oil and feed lubricant before using the product.
- (2) Do not combine different lubricants. Mixing lubricants can cause adverse interaction between disparate additives or other ingredients.
- (3) If the product will be exposed to constant vibration or high or low temperatures, or used in a clean room, vacuum, or other special environment, apply a lubricant suitable for both the specifications and the environment.
- (4) To lubricate a product that has no grease nipple or oil hole, apply lubricant directly to the raceway surface and execute a few preliminary strokes to ensure that the interior is fully lubricated.
- (5) Bear in mind that the Ball Spline's slide resistance is affected by changes in the consistency of the lubricant, which varies according to the temperature.
- (6) The Ball Spline may encounter increased slide resistance following lubrication, due to the lubricant's agitation resistance. Make sure to put the unit through some preliminary motions to ensure that it is fully lubricated before starting up the machine.
- (7) Excess lubricant may spatter immediately after lubrication. If necessary, wipe off any spattered grease.
- (8) Because lubricant performance declines over time, lubrication must be monitored regularly and fresh lubricant applied when needed, depending on how frequently the machine is operated.
- (9) The appropriate lubrication schedule will depend on usage conditions and the surrounding environment. In general, the unit should be lubricated after every 100 kilometers of operation (every 3 to 6 months). The actual lubrication schedule and amount of lubricant used should be determined by the condition of the machinery.
- (10) With oil lubrication, the lubricant may not always be thoroughly disseminated inside the Ball Spline, depending on its mounting position. If the preferred lubrication method is oil lubrication, please consult THK in advance.

[Storage]

When storing the Ball Spline, enclose it in a package designated by THK and store it in a room in a horizontal orientation while avoiding high temperature, low temperature and high humidity. After the product has been in storage for an extended period of time, lubricant inside may have deteriorated, so add new lubricant before use.

[Disposal]

Dispose of the product properly as industrial waste.

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